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(54) **Method for controlling variable capacity compressor of air conditioner**

Verfahren zur Steuerung eines Verdichters mit veränderlicher Fördermenge einer Klimaanlage

Procédé pour le contrôle de la capacité d'un compresseur à capacité variable d'un appareil de climatisation

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EP-A- 0 551 008 EP-A- 1 110 774
WO-A-2005/021300

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Description**BACKGROUND OF THE INVENTION****Field of the Invention**

[0001] The present invention relates to a method for controlling a variable capacity compressor of an air conditioner, and more particularly, to a method for controlling a variable capacity compressor of an air conditioner, which can improve convergence and response properties of temperature by changing a control duty change rate, namely, the maximum value in a slew rate, according to a size of the control duty or a change in an air conditioning environment.

Background Art

[0002] In general, a variable capacity compressor for an air conditioner uses a pressure control valve to adjust a refrigerant discharge volume, but may adopt an ECV (Electronic Control Valve) using electric power instead of the pressure control valve of a mechanical structure. A variable capacity compressor adopting the ECV determines the refrigerant discharge volume of the compressor according to an inclination of a swash plate since the inclination of the swash plate is changed by the duty of the ECV. So, a refrigerant volume supplied to an evaporator is changed according to the duty of the ECV, and it means that the duty of the ECV is the main factor to determine evaporator temperature. The duty of the ECV is a value to indicate a time period, while the ECV is turned on, into percentage. Therefore, the refrigerant discharge volume of the compressor by time is increased when the duty is high, but is decreased when the duty is low.

[0003] EP 1 110 774 A2 discloses that power consumption reduction and a cooling in a vehicle air-conditioning system have a compressor selectively driven by a vehicle engine or a motor while the compressor is driven by the motor. At least one of a capacity of a blower and a capacity of a compressor arranged in a refrigeration cycle R while the compressor being driven by a motor is reduced in comparison to the capacity of the blower and the capacity of the compressor while the compressor being driven by a vehicle engine. Furthermore, while a cooling heat load of an evaporator is greater than a predetermined value, the capacity of the compressor is first reduced prior to reducing the capacity of the blower.

[0004] EP 0 551 008 A1 is directed to a control apparatus for use in an automotive air conditioning system which includes a variable capacity type refrigerant compressor. The automotive air conditioning system comprises a refrigerant circuit including a refrigerant compressor with an eternally controlled variable capacity control mechanism and an evaporator connected to a suction chamber of the compressor, and a control apparatus which controls an operation thereof. The control apparatus includes an adjusting device for adjusting a control point of the compressor suction chamber pressure. In a primary initial stage of the operation of the automotive air conditioning system, the control point of the compressor suction chamber pressure is adjusted to be maintained at one constant value which is determined by the temperature of air immediately downstream the evaporator at a time an operation of the refrigerant circuit is initiated. In a secondary initial stage of the operation of the automotive air conditioning system, the control point of the compressor suction chamber pressure is adjusted to be varied by a result of operation of a proportional control action when the temperature of air immediately downstream the evaporator during the operation of the refrigerant circuit becomes equal to or lower than one certain value. Accordingly, a passenger compartment of the automobile can be more adequately air conditioned in the whole of the initial stage of the operation of the automotive air conditioning system.

[0005] As shown in FIG. 1, the air conditioner adopting the variable capacity compressor generally includes: an air conditioning case 210; an blower 220 mounted on an inlet side of the air conditioning case 210; an evaporator 200 mounted inside the air conditioning case 210 for cooling air using refrigerant discharged from a compressor 100; a heater core 230 mounted inside the air conditioning case 210 for receiving cooling water heated by an engine (E); a temperature control door 240 for controlling an opening and closing level of a cold air passageway and a hot air passageway of air passing through the evaporator 200; the compressor 100 for compressing and discharging refrigerant returned from the evaporator 200; a condenser 170 for condensing and discharging refrigerant supplied from the compressor 100; a receiver dryer 180 for gas-liquid separating refrigerant supplied from the condenser 170; and an expansion valve 190 for throttling refrigerant supplied from the receiver dryer 180 and sending it to the evaporator 200.

[0006] In the drawing, the unexplained reference numerals 212, 214 and 216 designate vents, and 212d, 214d and 216d designate doors for controlling the opening and closing amount of each of the vents 212, 214 and 216.

[0007] Meanwhile, a control unit 300 controls driving output of devices such as an electromagnetic clutch 146 for intermittently transmitting driving power of the engine (E) to the compressor 100, an ECV 160 for controlling a discharge volume of the compressor 100 by adjusting an inclination angle of the swash plate 144, and an actuator 310 for controlling the opening and closing level of the temperature control door 240. That is, the control unit 300 applies or interrupts electricity to or from the electromagnetic clutch 146, controls output voltage to the actuator 310 in such a way that the temperature control door 240 turns round toward a flow channel directing to the heater core 230 or a flow channel going

round the heater core 230, and controls the duty of the ECV 160, namely, a time period while the ECV 160 is turned on, to increase and decrease the discharge volume of the compressor 100 by changing the inclination angle of the swash plate 144 against a driving shaft.

[0008] The unexplained reference numeral 320 designates an evaporator temperature sensor, 330 designates an outdoor temperature sensor, 340 designates an indoor temperature sensor, 350 designates a solar radiation sensor, 360 designates a cooling water temperature sensor. Sensing signals sensed by the above sensors are inputted to the control unit 300, and temperature set by a user, blower voltage, and so on are inputted to the control unit 300.

[0009] A target evaporator temperature is calculated by the set temperature of a car inputted by the user, the indoor temperature, the outdoor temperature and a solar radiation sensed and inputted from the sensors 330, 340 and 350 mounted at predetermined positions of the car. When the target evaporator temperature is calculated, a target duty of the ECV is calculated by evaporator temperature measured by the evaporator temperature sensor 320 and the target evaporator temperature.

[0010] When the target duty is calculated, the duty is changed from the current duty to the target duty by a predetermined slew rate, and in this instance, the slew rate is set not to exceed a basic slew rate (S0). Here, the slew rate means a change rate of the duty.

[0011] The basic slew rate (S0) is determined as a value to prevent hunting of the compressor, namely, a bump generated due to a sudden change of an introduced refrigerant volume of the compressor, and to minimize a pulsation of a refrigerant flow even though the introduced refrigerant volume of the compressor is changed.

[0012] According to a conventional compressor controlling method, the target duty is calculated by using the real evaporator temperature measured by the evaporator temperature sensor, the duty controlling the ECV, and the target evaporator temperature as a variable, and the calculated duty controls the ECV of the compressor. After the control of the ECV, the evaporator temperature is measured again, and the above process for calculating the target duty is repeated.

[0013] Even though the user suddenly manipulates a blower switch or changes the set temperature, the conventional compressor controlling method is deteriorated in convergence and response properties of the evaporator temperature since the duty change rate does not exceed the basic slew rate.

[0014] In other words, the conventional compressor controlling method controls in such a way that the duty change rate of the ECV 160, namely, the slew rate, cannot exceed the basic slew rate (S0) without regard to changes of the outside conditions. So, since the ECV duty is changed slowly even though the environment is suddenly changed, there occurs excessive overshoot or undershoot in evaporator temperature till the duty reaches the target duty, and thereby, it takes much time that the evaporator temperature reaches a stabilized state. Therefore, the conventional compressor controlling method has a problem in that convergence of the evaporator temperature and response properties to the user's manipulation of the air conditioner are deteriorated.

SUMMARY OF THE INVENTION

[0015] Accordingly, to solve the above mentioned problems occurring in the prior art, and it is an object of the present invention to provide a method for controlling a variable capacity compressor of an air conditioner, which prevents excessive overshoot or undershoot of evaporator temperature by changing the maximum value of a slew rate to rapidly remove disturbance when set temperature inputted by a user or blower voltage is suddenly changed, thereby improving convergence and response properties of evaporator temperature.

[0016] To accomplish the above objects, according to an embodiment of the present invention, there is provided a method for controlling a variable capacity compressor of an air conditioner comprising the steps of: setting a target indoor temperature of a car by a user; sensing and inputting car indoor temperature, car outdoor temperature and solar radiation using sensors mounted at predetermined positions of the car; calculating a target discharge temperature of a vent using data of the target indoor temperature, the car indoor temperature, the car outdoor temperature and the solar radiation; calculating a target evaporator temperature and blower voltage according to the target discharge temperature; calculating a control duty according to the target evaporator temperature; calculating a target evaporator temperature change valve and a blower voltage change valve; determining whether or not it comes under a sudden variable condition through the control duty, the target evaporator temperature change valve and the blower voltage change valve; and setting the maximum value of the control duty change rate to an accelerative slew rate (Sc) greater than a basic slew rate (S0) when the sudden variable condition is determined, but setting the maximum value of the control duty change rate to the basic slew rate (S0) when it does not come under the sudden variable condition, wherein

[0017] the sudden variable condition satisfies one of conditions that the control duty reaches the maximum value or the minimum value, that the target evaporator temperature change valve is more than a predetermined value (a) and that the blower voltage change valve is more than a predetermined value (b).

[0018] Moreover, the method for controlling the variable capacity compressor further includes the step of returning to the step of calculating the control duty after setting blower voltage when the user changes the stage of a blower.

[0019] In addition, the accelerative slew rate (Sc) satisfies the following formula, $40\%/min. \leq Sc \leq 60\%/min.$

BRIEF DESCRIPTION OF THE DRAWINGS

[0020] The above and other objects, features and advantages of the present invention will be apparent from the following detailed description of the preferred embodiments of the invention in conjunction with the accompanying drawings, in which:

[0021] FIG. 1 is a structural view of an air conditioner to which a variable capacity compressor is applied;

[0022] FIG. 2 is a graph showing a change in evaporator temperature when the maximum value of a slew rate is set to a basic slew rate (S0) to change a duty in case that blower voltage is suddenly changed from 4V to 12V and from 12V to 4V, in a conventional method for controlling a variable capacity compressor of an air conditioner;

[0023] FIG. 3 is a graph showing a refrigerant flow rate according to a duty change of the variable capacity compressor;

[0024] FIG. 4 is a flow chart showing a method for controlling a variable capacity compressor of an air conditioner according to the present invention;

[0025] FIG. 5 is a flow chart showing a method for controlling a variable capacity compressor of an air conditioner according to another embodiment of the present invention;

[0026] FIG. 6 is a graph for comparing changes in evaporator temperature generated when the maximum value of a control duty change rate is controlled by the basic slew rate (S0) and when it is controlled by an accelerative slew rate (Sc) in case that blower voltage is suddenly changed from 12V to 7V; and

[0027] FIG. 7 is a graph showing a change in evaporator temperature when the maximum value of the control duty change rate is controlled by the accelerative slew rate (Sc) in case that blower voltage is suddenly changed from 4V to 12V and from 12V to 4V.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

[0028] Reference will be now made in detail to the preferred embodiment of the present invention with reference to the attached drawings.

[0029] FIGS. 4 and 5 are flow charts showing a method for controlling a variable capacity compressor of an air conditioner according to the present invention, FIG. 6 is a graph for comparing changes in evaporator temperature generated when the maximum value of a control duty change rate is controlled by the basic slew rate (S0) and when it is controlled by an accelerative slew rate (Sc) in case that blower voltage is suddenly changed from 12V to 7V, and FIG. 7 is a graph showing a change in evaporator temperature when the maximum value of the control duty change rate is controlled by the accelerative slew rate (Sc) in case that blower voltage is suddenly changed from 4V to 12V and from 12V to 4V.

[0030] As shown in FIG. 4, the method for controlling the variable capacity compressor of the air conditioner according to the present invention includes the steps of: setting a target indoor temperature of a car by a user (S10); sensing and inputting car indoor temperature, car outdoor temperature and solar radiation using sensors mounted at predetermined positions of the car (S20); calculating a target discharge temperature of a vent using data of the target indoor temperature, the car indoor temperature, the car outdoor temperature and the solar radiation (S30); calculating a target evaporator temperature and blower voltage according to the target discharge temperature (S40); calculating a control duty according to the target evaporator temperature; calculating a target evaporator temperature change valve and a blower voltage change valve (S60); determining whether or not it comes under a sudden variable condition through the control duty, the target evaporator temperature change valve and the blower voltage change valve (S70); and setting the maximum value of the control duty change rate to an accelerative slew rate (Sc) greater than a basic slew rate (S0) when the sudden variable condition is determined, but setting the maximum value of the control duty change rate to the basic slew rate (S0) when it does not come under the sudden variable condition.

[0031] As shown in FIG. 5, if the user changes the stage of a blower, the controlling method may further include the step of returning to the step of calculating the control duty after setting blower voltage (S90).

[0032] Here, the sudden variable condition means to satisfy one of conditions that the control duty reaches the maximum value or the minimum value, that the target evaporator temperature change valve is more than a predetermined value (a) and that the blower voltage change valve is more than a predetermined value (b).

[0033] In this instance, the predetermined value (a) in the target evaporator temperature change valve satisfies the following formula, $3^{\circ}\text{C} \leq |\Delta| \leq 7^{\circ}\text{C}$, and the predetermined value (b) in the blower voltage change valve satisfies the following formula, $3\text{V} \leq |\Delta| \leq 7\text{V}$. In addition, the accelerative slew rate (Sc) satisfies the following formula, $40\%/min. \leq Sc \leq 60\%/min.$

[0034] Meanwhile, FIG. 6 shows changes in blower voltage, control duty, and evaporator temperature according to the maximum value of the control duty change rate in a state where the target duty is the greatest when the blower voltage is high or the outdoor temperature is very high. In FIG. 6, a solid line shows a case that the maximum value of the control duty change rate is controlled by the basic slew rate (S0), and a dotted line shows a case that the maximum value of the control duty change rate is controlled by the accelerative slew rate (Sc).

[0035] In general, when the blower voltage is high or the outdoor temperature is very high, the evaporator does not perform a sufficient heat exchange. Therefore, even though the air conditioner is calculated, it is difficult to reach the target evaporator temperature, and the target duty is set to the maximum value since the evaporator temperature measured by the evaporator temperature sensor is high.

[0036] In the above state, when the blower voltage is reduced from 12V to 7V, the stage of the blower drops so as to perform the sufficient heat exchange. So, the evaporator temperature measured by the evaporator temperature sensor drops, and the target duty is calculated with a low value.

[0037] Here, when the maximum value of the duty change rate is controlled by the basic slew rate (S0), undershoot is generated. The basic slew rate (S0) is a duty change rate set to minimize a pulsation of a refrigerant flow according to a change of refrigerant introduced into the compressor and to minimize hunting of the compressor, and is set by 20%/min.

[0038] On the other hand, if the maximum value of the duty change rate is controlled by the accelerative slew rate (Sc), undershoot is not generated and a stable evaporator temperature can be obtained. Here, the accelerative slew rate (Sc) is set by 50%/min.

[0039] The method for controlling the variable for the air conditioner according to the present invention provides a good response property according to the control of the air conditioner by changing the duty change rate.

[0040] First, sensor values of an evaporator temperature sensor 320, an outdoor temperature sensor 330, an indoor temperature sensor 340, a solar radiation sensor 350, a cooling water temperature sensor 360, and so on are inputted to a control unit 300, and then, set temperature and blower voltage set by the user are inputted to the control unit 300. According to cars, there is a difference in blower voltage, but most of the cars provide the lowest stage of about 3V and the highest stage of about 12V.

[0041] The target evaporator temperature is calculated by the set temperature of the car inputted by the user, the car indoor temperature, the car outdoor temperature and the solar radiation sensed and inputted by the sensors 330, 340 and 350. When the target evaporator temperature is calculated, the target duty of the ECV is calculated by the current evaporator temperature and the target evaporator temperature.

[0042] When the target duty is calculated, the control duty change rate is set after determining whether or not it comes under the sudden variable condition. That is, if the target evaporator temperature change value is more than the predetermined value (a), if the blower voltage change value is more than the predetermined value (b), or if the duty is the maximum value or the minimum value, the control duty change rate is set to the accelerative slew rate (Sc) greater than the basic slew rate (S0) set to prevent the pulsation of the refrigerant flow. Of course, if it does not come under the sudden variable condition, the maximum value of the control duty change rate is set to the basic slew rate (S0). When the control duty change rate is determined, the duty is changed from the current duty toward the target duty.

[0043] Even though the blower voltage is changed greatly, the target evaporator temperature is change greatly, or the duty reaches the maximum value or the minimum value, when the maximum value of the control duty change rate is set to the basic slew rate (S0), excessive overshoot or undershoot in the evaporator temperature may be generated while the duty is changed to the target duty. On the other hand, when the maximum value of the control duty change rate is controlled by the accelerative slew rate (Sc), a stable target evaporator temperature can be achieved.

[0044] That is, as shown in FIG. 2, if the blower voltage is suddenly changed from 4V to 12V or from 12V to 4V, when the maximum value of the control duty change rate is controlled by the basic slew rate (Sc) to change the duty, excessive overshoot or undershoot in evaporator temperature may be generated. On the other hand, as shown in FIG. 7, if the maximum value of the control duty change rate is controlled by the accelerative slew rate (Sc), excessive overshoot or undershoot in evaporator temperature is not generated while the duty is changed, and so, more stable evaporator temperature can be obtained.

[0045] Meanwhile, when the user changes the stage of the blower, the control duty is calculated again after blower voltage is set. Therefore, air conditioning can be controlled according to the user's manipulation.

[0046] Here, the maximum value of the control duty means a value that changes little the flow rate of refrigerant even though the duty is increased, and the minimum value of the control duty means a value, which changes little the flow rate of refrigerant even though the duty is decreased. As described above, when the slew rate is determined, the duty is changed in such a way that the control duty change rate does not exceed the maximum value of the set slew rate so as to control to the target duty.

[0047] As described above, when the user suddenly changes blower voltage, when the air conditioning environment, for example, the set temperature, is suddenly changed, or when the duty of the ECV reaches the maximum value or the minimum value, the method for controlling the variable capacity compressor for the air conditioner according to the present invention controls the maximum value of the duty change rate of the ECV with the accelerative slew rate greater than the basic slew rate to prevent pulsation of the refrigerant flow, hunting of the compressor, and the excessive overshoot or undershoot, thereby improving convergence and response properties of temperature.

[0048] While the present invention has been described with reference to the particular illustrative embodiments, it is not to be restricted by the embodiments but only by the appended claims. It is to be appreciated that those skilled in the

art can change or modify the embodiments without departing from the scope of the present invention.

Claims

1. A method for controlling a variable capacity compressor of an air conditioner comprising the steps of:

setting a target indoor temperature of a car by a user;
sensing and inputting car indoor temperature, car outdoor temperature and solar radiation using sensors mounted at predetermined positions of the car;
calculating a target discharge temperature of a vent using data of the target indoor temperature, the car indoor temperature, the car outdoor temperature and the solar radiation;
calculating a target evaporator temperature and blower voltage according to the target discharge temperature;
calculating a control duty according to the target evaporator temperature;
calculating a target evaporator temperature change value and a blower voltage change value;
determining whether or not it comes under a sudden variable condition through the control duty, the target evaporator temperature change fate value and the blower voltage change value; and
setting the maximum value of the control duty change rate to an accelerative slew rate (S_c) greater than a basic slew rate (S_0) when the sudden variable condition is determined, but setting the maximum value of the control duty change rate to the basic slew rate (S_0) when it does not come under the sudden variable condition, wherein the sudden variable condition satisfies one of conditions that the control duty reaches the maximum value or the minimum value, that the target evaporator temperature change fate value is more than a predetermined value (a) and that the blower voltage change value is more than a predetermined value (b).

2. The method for controlling a variable capacity compressor according to claim 1, wherein the predetermined value (a) in the target evaporator temperature change value satisfies the following formula,

$$3^{\circ}\text{C} \leq |a| \leq 7^{\circ}\text{C}.$$

3. The method for controlling a variable capacity compressor according to claim 1, wherein the predetermined value (b) in the blower voltage change fate value satisfies the following formula,

$$3\text{V} \leq |b| \leq 7\text{V}.$$

4. The method for controlling a variable capacity compressor according to claim 1, further comprising the step of returning to the step of calculating the control duty after setting blower voltage when the user changes the stage of a blower.

5. The method for controlling a variable capacity compressor according to claim 1, wherein the accelerative slew rate (S_c) satisfies the following formula,

$$40\%/min. \leq S_c \leq 60\%/min.$$

Patentansprüche

1. Verfahren zur Steuerung eines Verdichters mit veränderlicher Fördermenge einer Klimaanlage, mit den folgenden Schritten:

- Einstellen einer Zielinnentemperatur eines Fahrzeugs durch einen Benutzer;
- Erfassen und Eingeben der Fahrzeuginnentemperatur, der Fahrzeugaußentemperatur und der Sonneneinstrahlung unter Verwendung von Sensoren, die an vorbestimmten Positionen des Fahrzeugs angebracht sind;
- Berechnen einer Zielauslasstemperatur einer Lüftung unter Verwendung von Daten der Zielinnentemperatur, der Fahrzeuginnentemperatur, der Fahrzeugaußentemperatur und der Sonneneinstrahlung;
- Berechnen einer Zielverdampfertemperatur und der Gebläsespannung gemäß der Zielauslasstemperatur;
- Berechnen eines Steuerungsaufwands gemäß der Zielverdampfertemperatur;
- Berechnen eines Änderungswerts der Zielverdampfertemperatur und eines Änderungswerts der Gebläsespannung;
- Ermitteln, ob es durch den Steuerungsaufwand, den Änderungswert der Zielverdampfertemperatur und den Änderungswert der Gebläsespannung zu einem plötzlich veränderlichen Zustand kommt oder nicht; und
- Einstellen des Maximalwerts der Änderungsrate des Steuerungsaufwands auf eine Beschleunigungsanstiegsgeschwindigkeit (S_c) größer als eine Grundanstiegsgeschwindigkeit (S_0), wenn der plötzlich veränderliche Zustand festgestellt wird, aber Einstellen des Maximalwerts der Änderungsrate des Steuerungsaufwands auf die Grundanstiegsgeschwindigkeit (S_0), wenn es nicht zu dem plötzlich veränderlichen Zustand kommt;
- wobei der plötzlich veränderliche Zustand eine der Bedingungen erfüllt, dass der Steuerungsaufwand den Maximalwert oder den Minimalwert erreicht, dass der Änderungswert der Zielverdampfertemperatur größer ist als ein vorbestimmter Wert (a) und dass der Änderungswert der Gebläsespannung größer ist als ein vorbestimmter Wert (b).

2. Verfahren zum Steuern eines Verdichters mit veränderlicher Fördermenge nach Anspruch 1, wobei der vorbestimmte Wert (a) bei dem Änderungswert der Zielverdampfertemperatur der folgenden Formel genügt:

$$3^{\circ}\text{C} \leq |a| \leq 7^{\circ}\text{C}.$$

3. Verfahren zum Steuern eines Verdichters mit veränderlicher Fördermenge nach Anspruch 1, wobei der vorbestimmte Wert (b) bei dem Änderungswert der Gebläsespannung der folgenden Formel genügt:

$$3V \leq |b| \leq 7V.$$

4. Verfahren zum Steuern eines Verdichters mit veränderlicher Fördermenge nach Anspruch 1, das ferner den Schritt der Rückkehr zu dem Schritt des Berechnens des Steuerungsaufwands nach dem Einstellen der Gebläsespannung umfasst, wenn der Benutzer die Stufe eines Gebläses ändert.

5. Verfahren zum Steuern eines Verdichters mit veränderlicher Fördermenge nach Anspruch 1, wobei die Beschleunigungsanstiegsgeschwindigkeit (S_c) der folgenden Formel genügt:

$$40\% / \text{min.} \leq S_c \leq 60\% / \text{min.}$$

Revendications

1. Procédé pour la commande d'un compresseur à capacité variable d'un appareil de climatisation comprenant les étapes de:

régler une température intérieure cible d'une voiture par un utilisateur;
détecter et entrer la température intérieure de la voiture, la température extérieure de la voiture et le rayonnement solaire en utilisant des capteurs installés à des positions prédéterminées de la voiture;

calculer une température d'évacuation cible d'un événement en utilisant les données de la température intérieure cible, de la température intérieure de la voiture, de la température extérieure de la voiture et du rayonnement solaire;

calculer un service de commande selon la température d'évaporateur cible;

calculer une valeur de changement de température d'évaporateur cible et une valeur de changement de tension de soufflante;

déterminer si oui ou non elle vient sous une condition variable brusque par le service de commande, la valeur de changement de température d'évaporateur cible et la valeur de changement de tension de soufflante; et

régler la valeur maximale du taux de changement de commande de service à un taux de changement d'accélération (S_c) plus grand qu'un taux de changement de base (S_0) lorsqu'une condition variable brusque est déterminée, mais régler la valeur maximale du taux de changement de commande de service au taux de changement de base (S_0) lorsqu'elle ne vient pas sous la condition variable brusque,

où la condition variable brusque satisfait à une des conditions, que la commande de service atteint la valeur maximale ou la valeur minimale, que la valeur de changement de température d'évaporateur cible est supérieure à une valeur prédéterminée (a) et que la valeur de changement de tension de soufflante est supérieure à une valeur prédéterminée (b).

2. Procédé de commande d'un compresseur à capacité variable selon la revendication 1, dans lequel la valeur prédéterminée (a) dans la valeur de changement de température d'évaporateur cible satisfait à la formule suivante,

$$3^{\circ}\text{C} \leq |a| \leq 7^{\circ}\text{C}.$$

3. Procédé de commande d'un compresseur à capacité variable selon la revendication 1, dans lequel la valeur prédéterminée (b) dans la valeur de changement de tension de soufflante satisfait à la formule suivante,

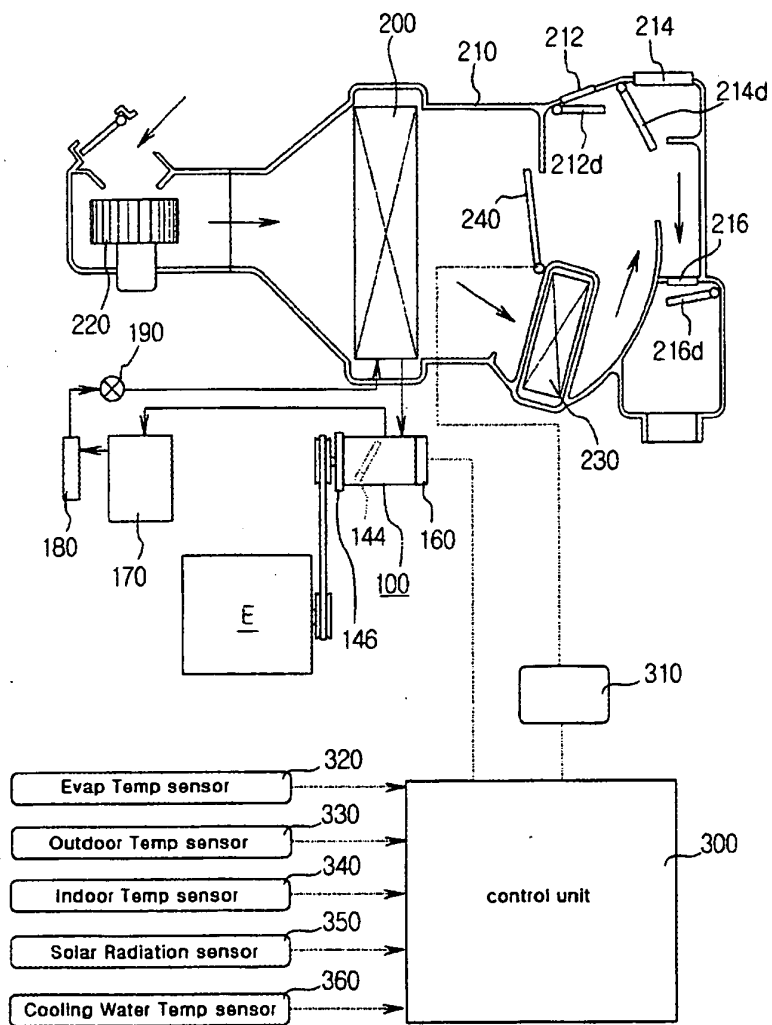
$$3\text{V} \leq |b| \leq 7\text{V}.$$

4. Procédé de commande d'un compresseur à capacité variable selon la revendication 1, comprenant en outre l'étape consistant à retourner à l'étape du calcul de la commande de service après le réglage de la tension de la soufflante lorsqu'un utilisateur change l'étape d'une soufflante.

5. Procédé de commande d'un compresseur à capacité variable selon la revendication 1, dans lequel le taux de changement d'accélération (S_c) satisfait à la formule suivante,

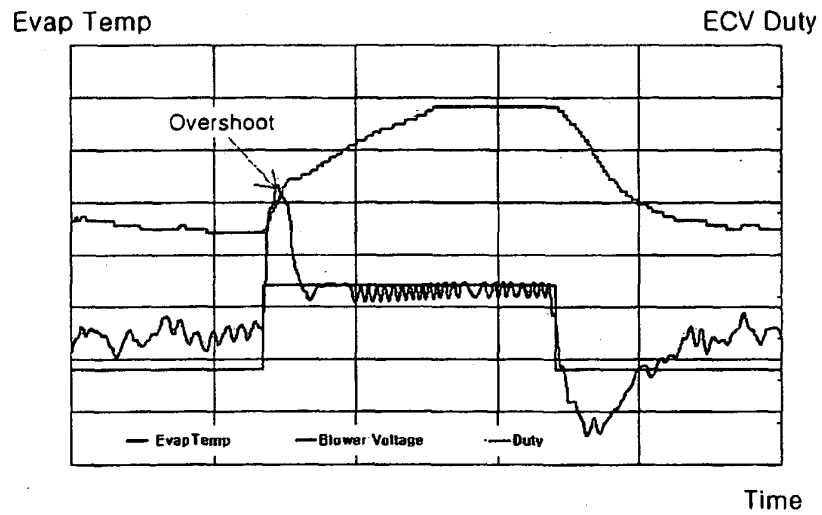
$$40\%/min. \leq S_c \leq 60\%/min.$$

FIG. 1



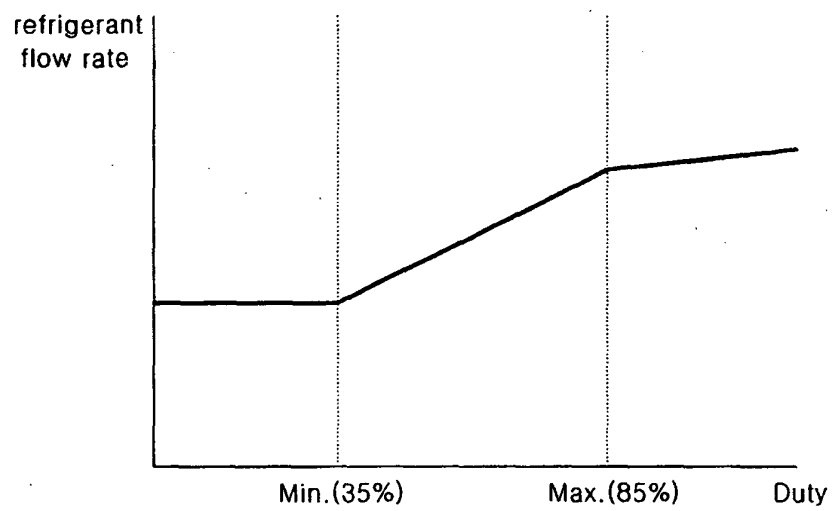
PRIOR ART

FIG. 2



PRIOR ART

FIG. 3



PRIOR ART

FIG. 4

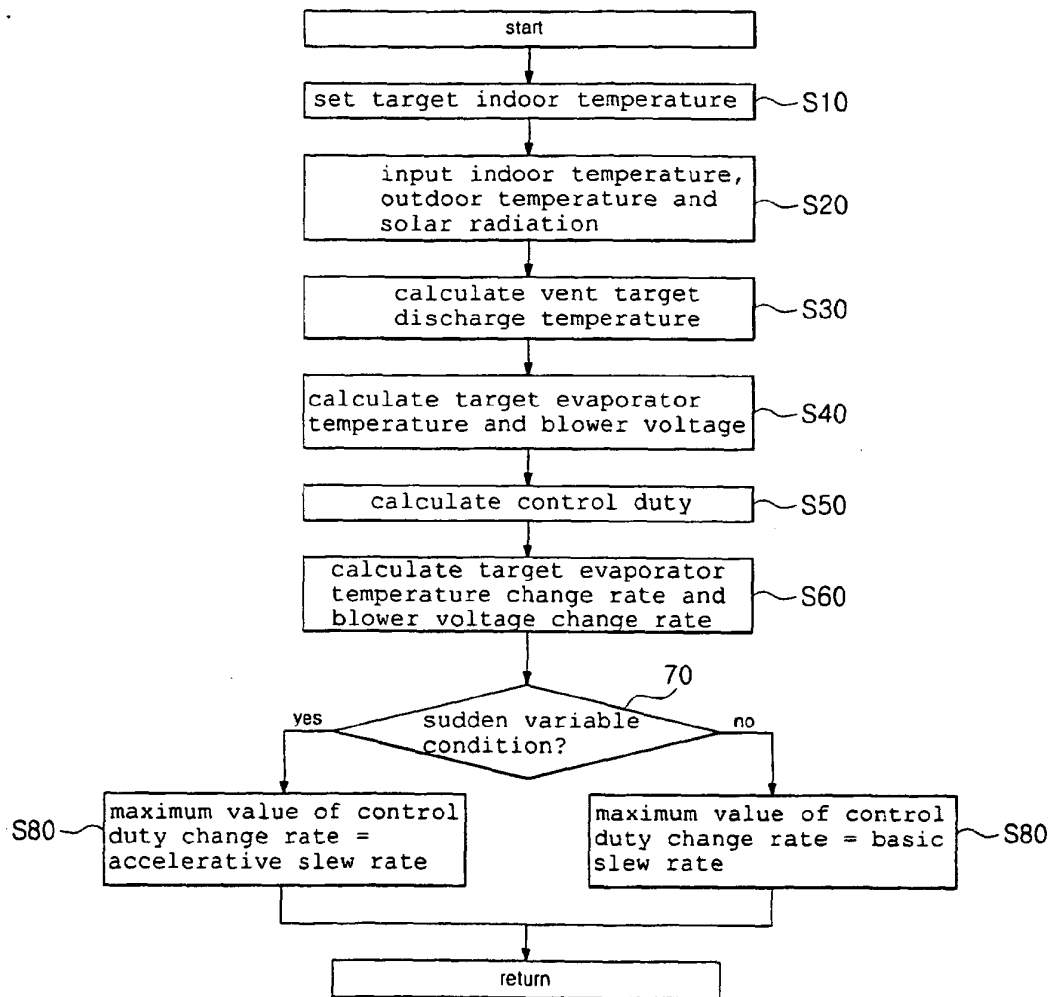


FIG. 5

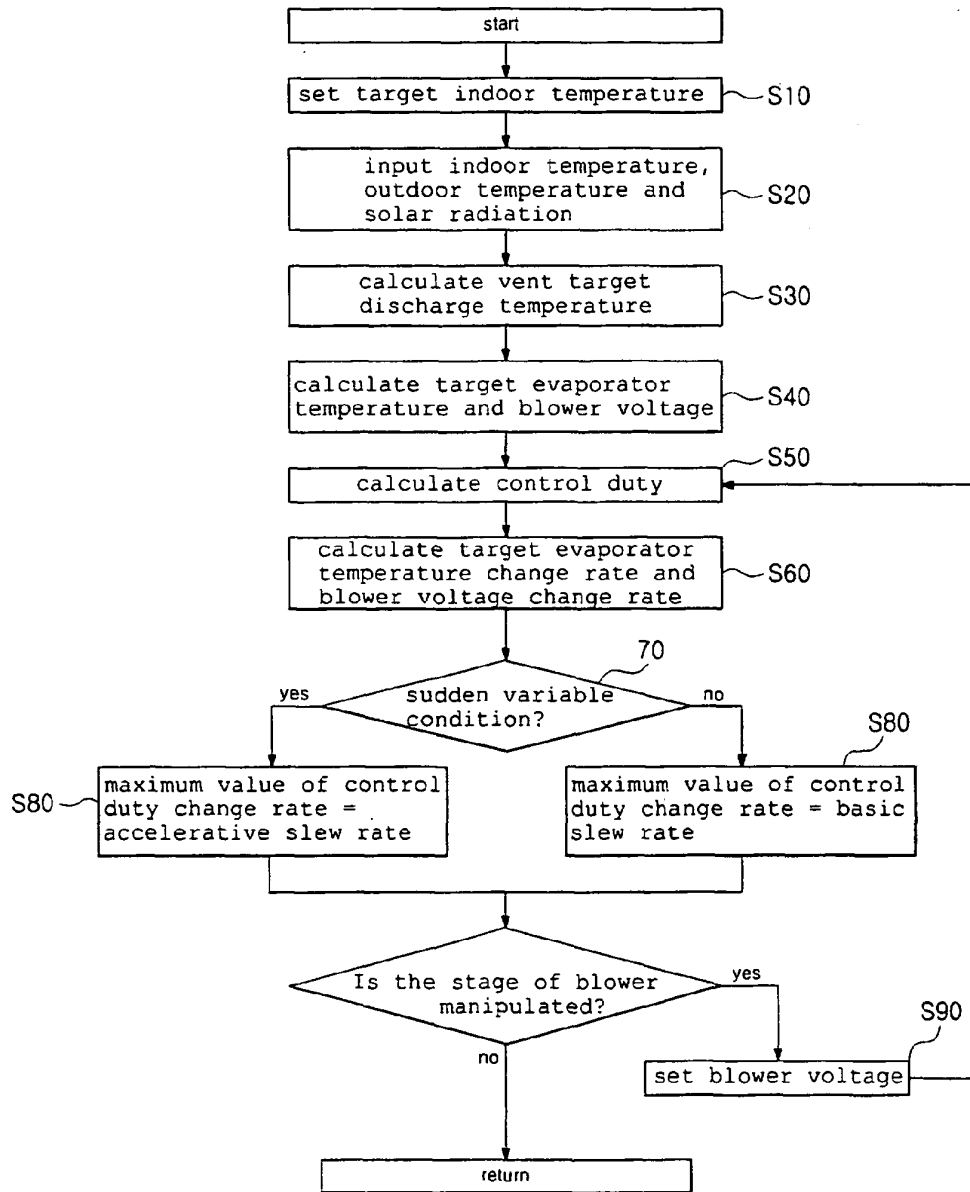


FIG. 6

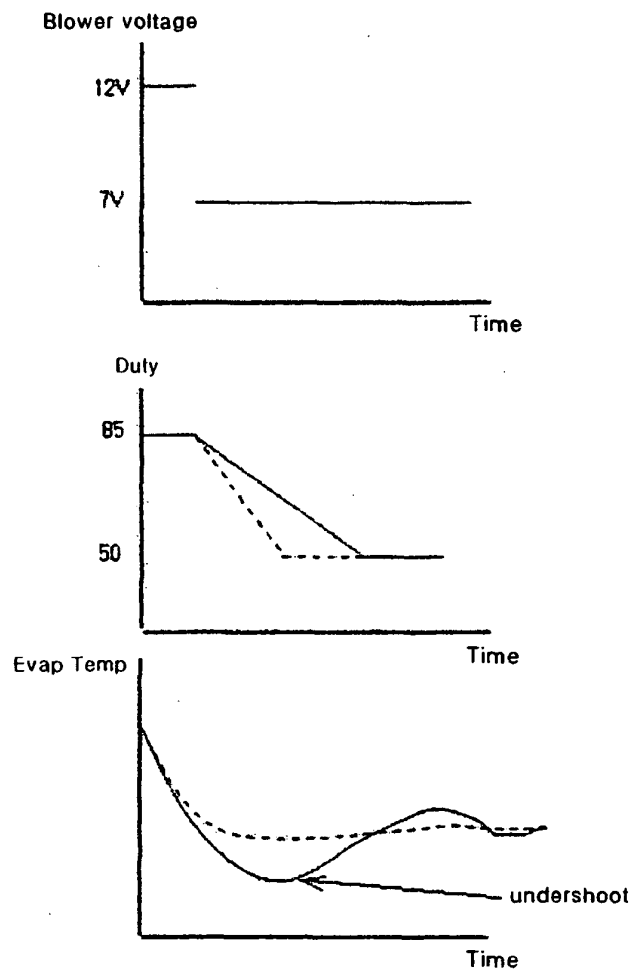
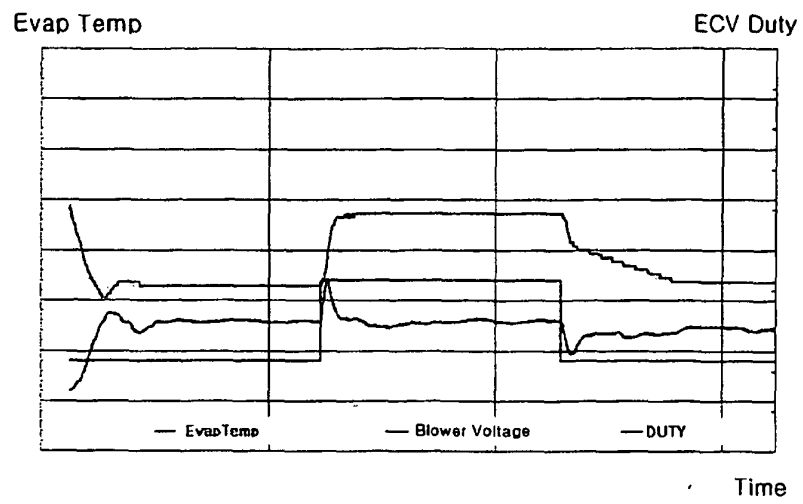


FIG. 7



REFERENCES CITED IN THE DESCRIPTION

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