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Takechi et al.

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(54) **AIR CONDITIONING APPARATUS FOR VEHICLE**

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Dec. 24, 1996 (JP) 8-343410
Apr. 9, 1997 (JP) 9-091003

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(52) **U.S. Cl.** **165/42; 165/43; 165/203; 165/204; 237/12.3 A; 237/12.3 B**

(58) **Field of Search** **165/42, 43, 203, 165/204; 237/12.3 A, 12.3 B**

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(57) **ABSTRACT**

According to the present invention, in an air conditioning apparatus setting a double laminar mode in which inside air and outside air are blown while being partitioned, when the maximum heating state (including the high heating capacity close to the maximum heating operation) is set, a warm air bypass door for introducing warm air directly into a foot opening portion, without being mixed into cool air, functions as a movable partition member for partitioning an air passage at a downstream side of a heating heat exchanger, into an inside air passage and an outside air passage. In this way, it is possible to downsize the air conditioning unit.

18 Claims, 28 Drawing Sheets

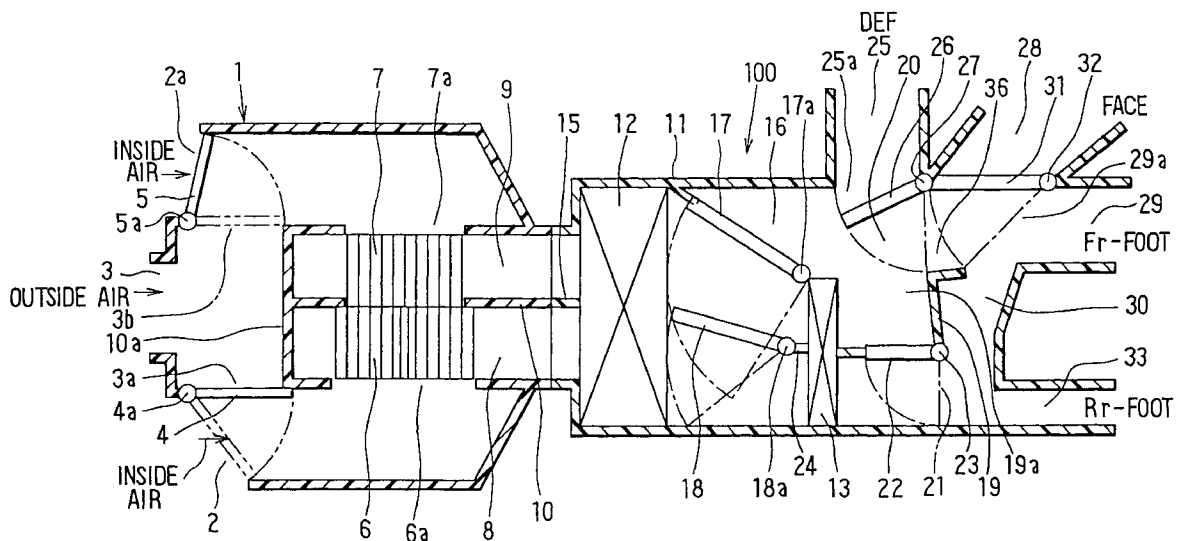


FIG. 1

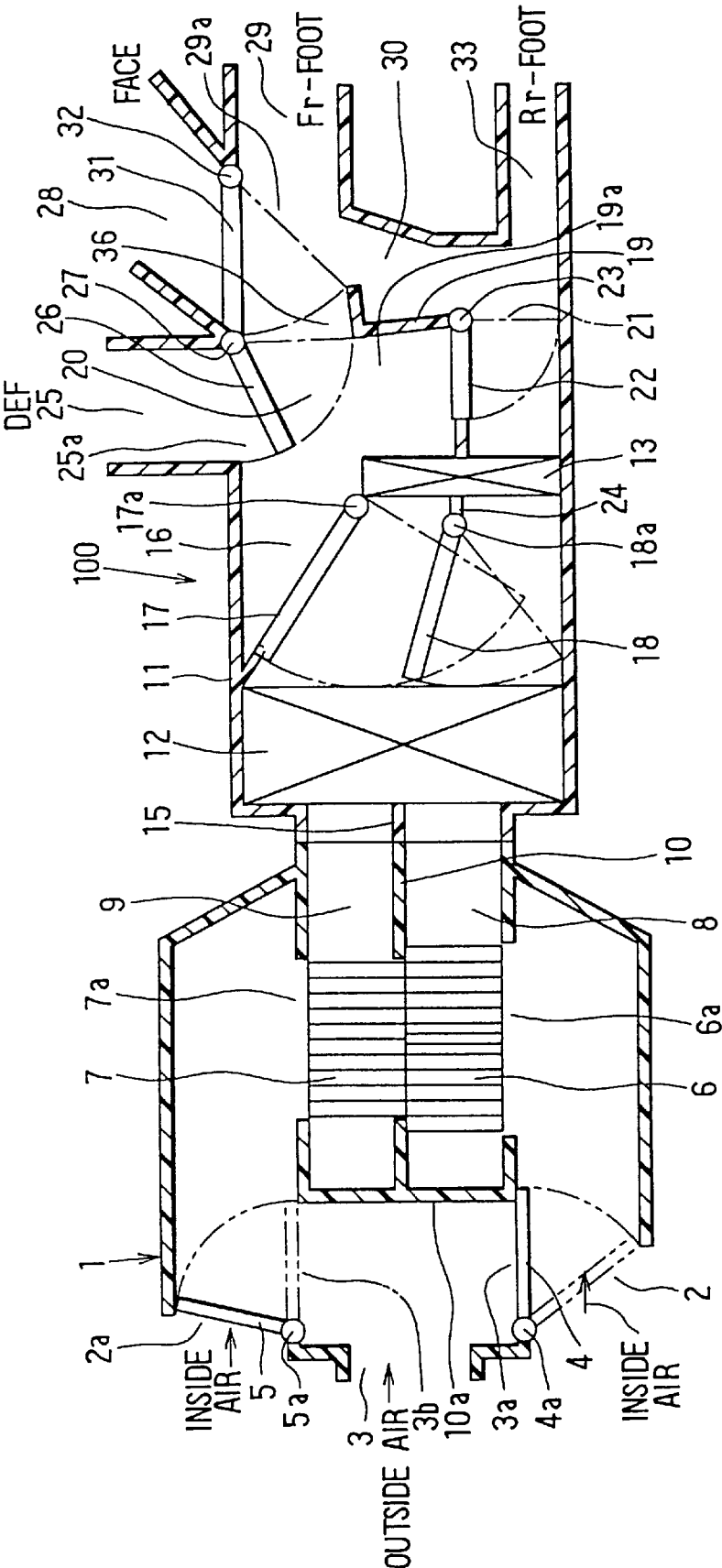


FIG. 2

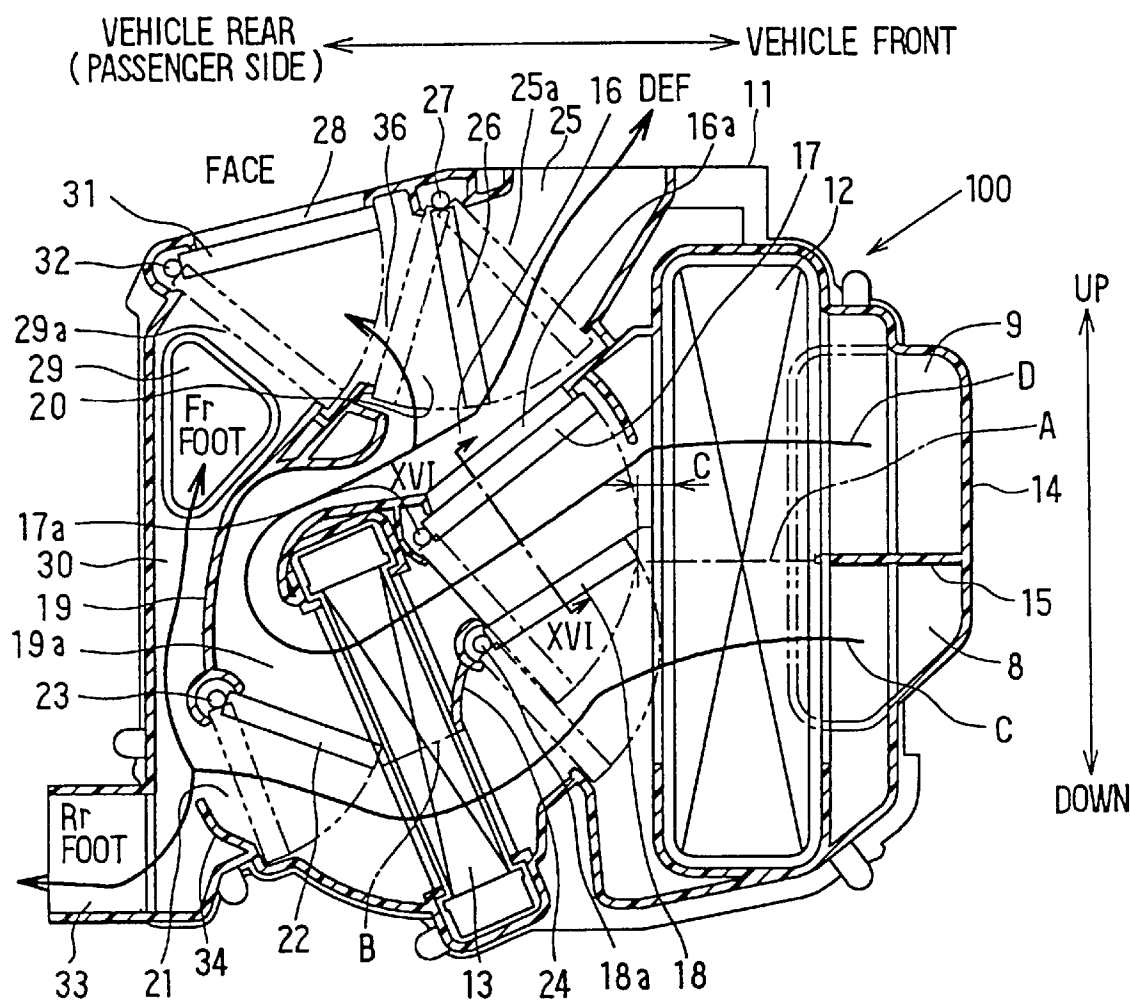


FIG. 3

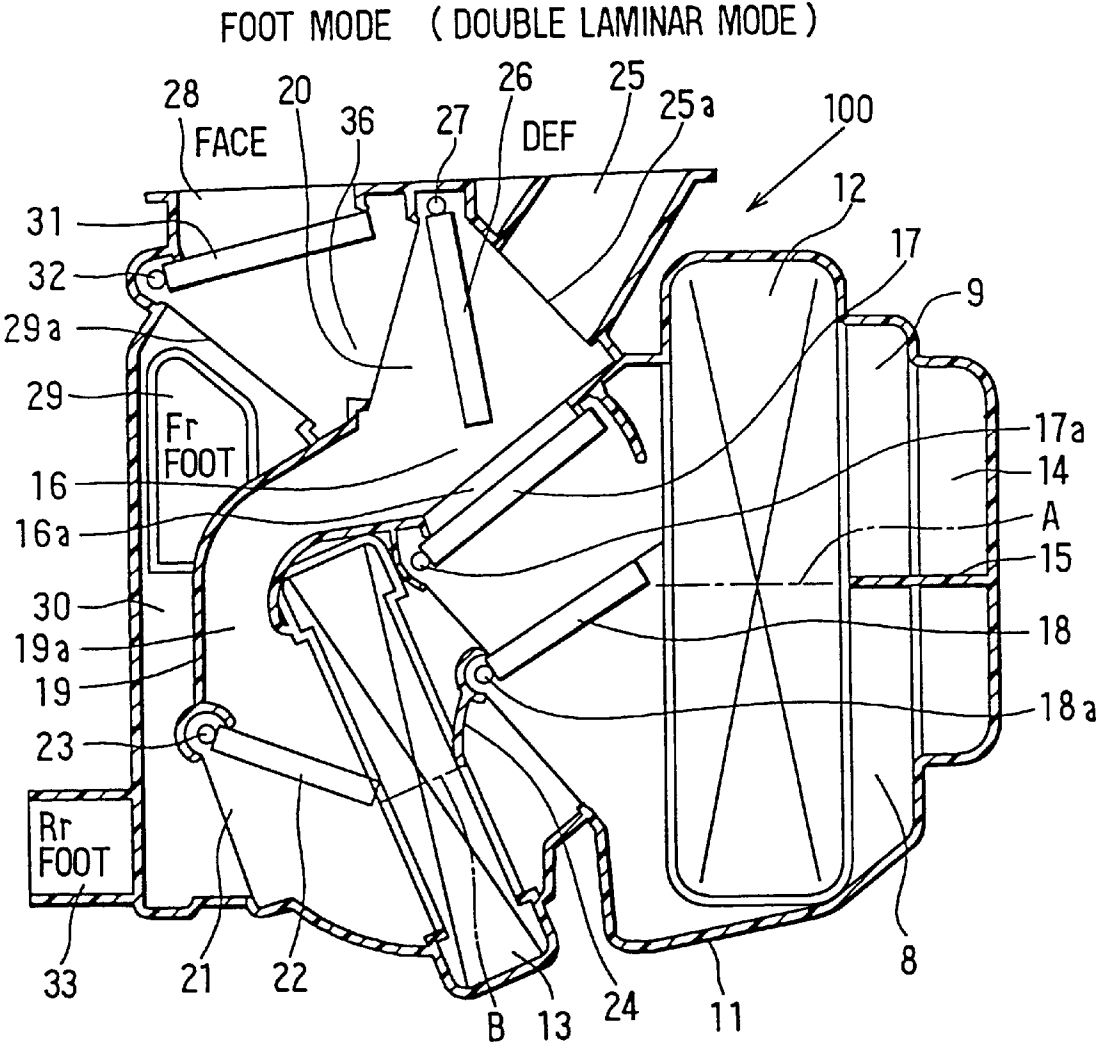


FIG. 4

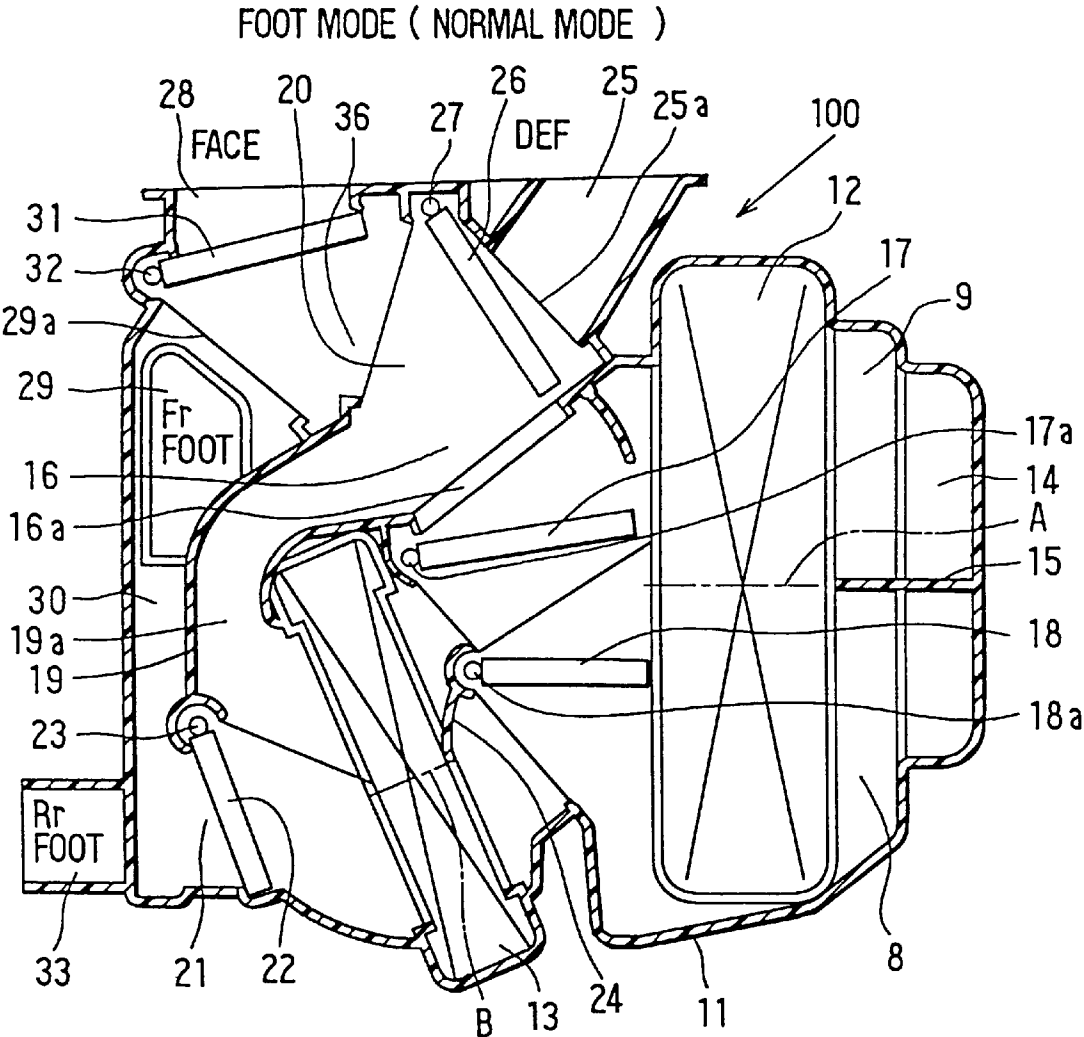


FIG. 6

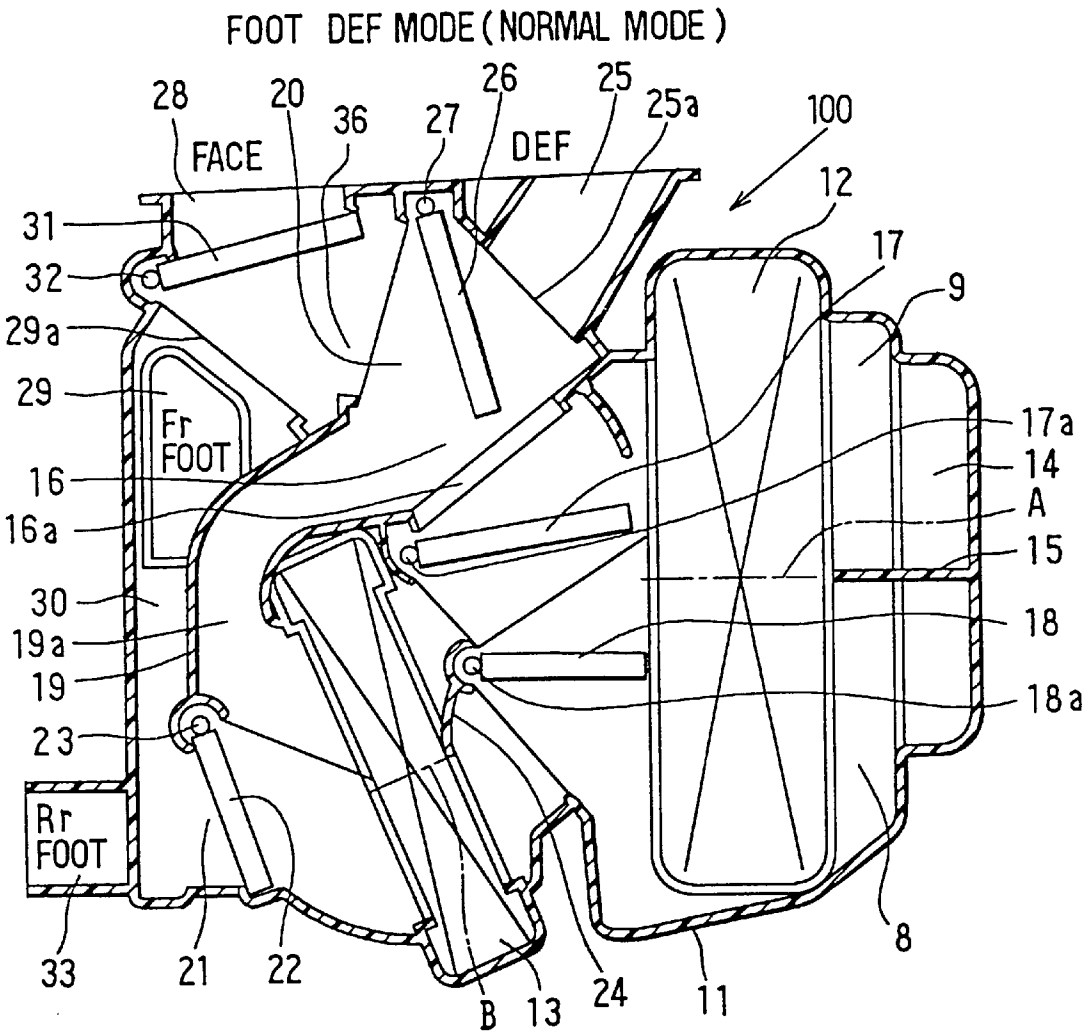


FIG. 7

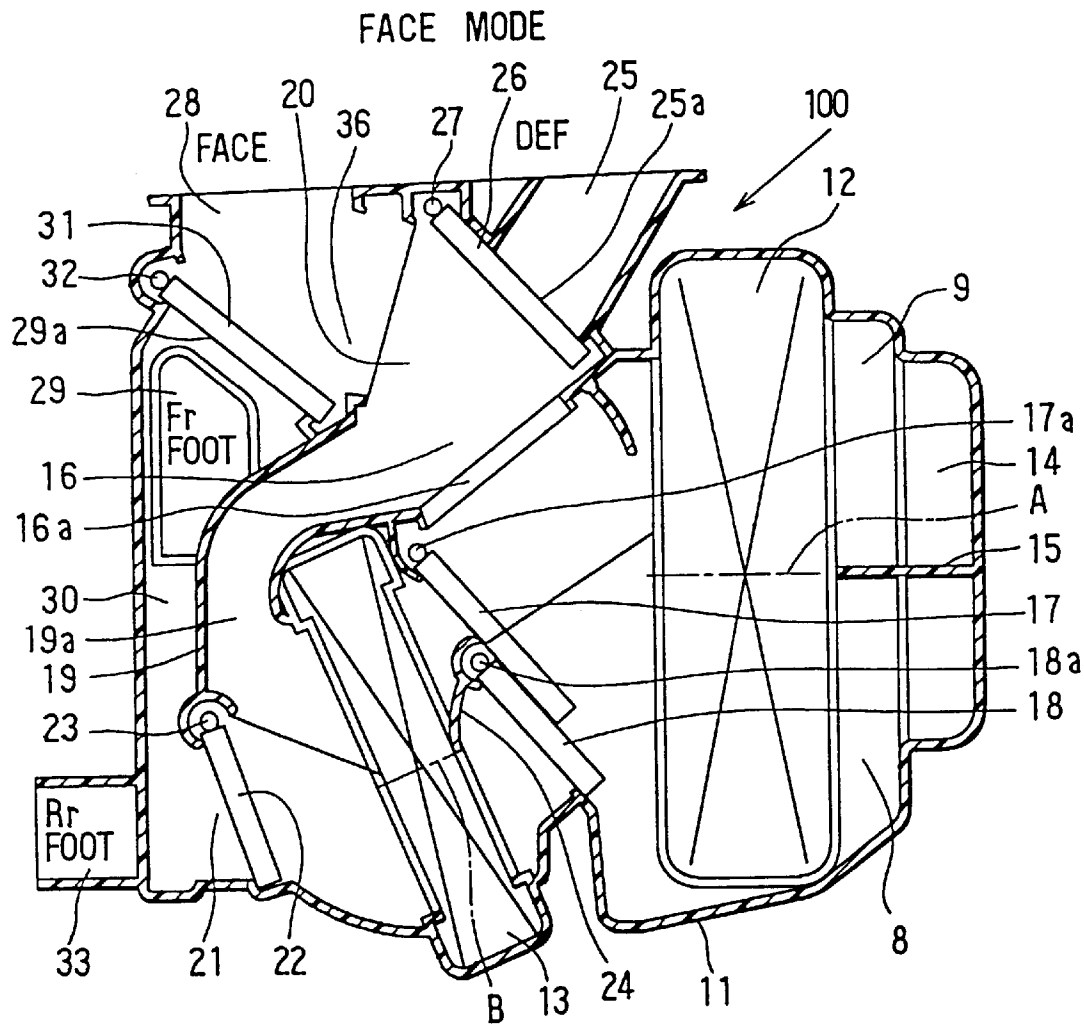


FIG. 8

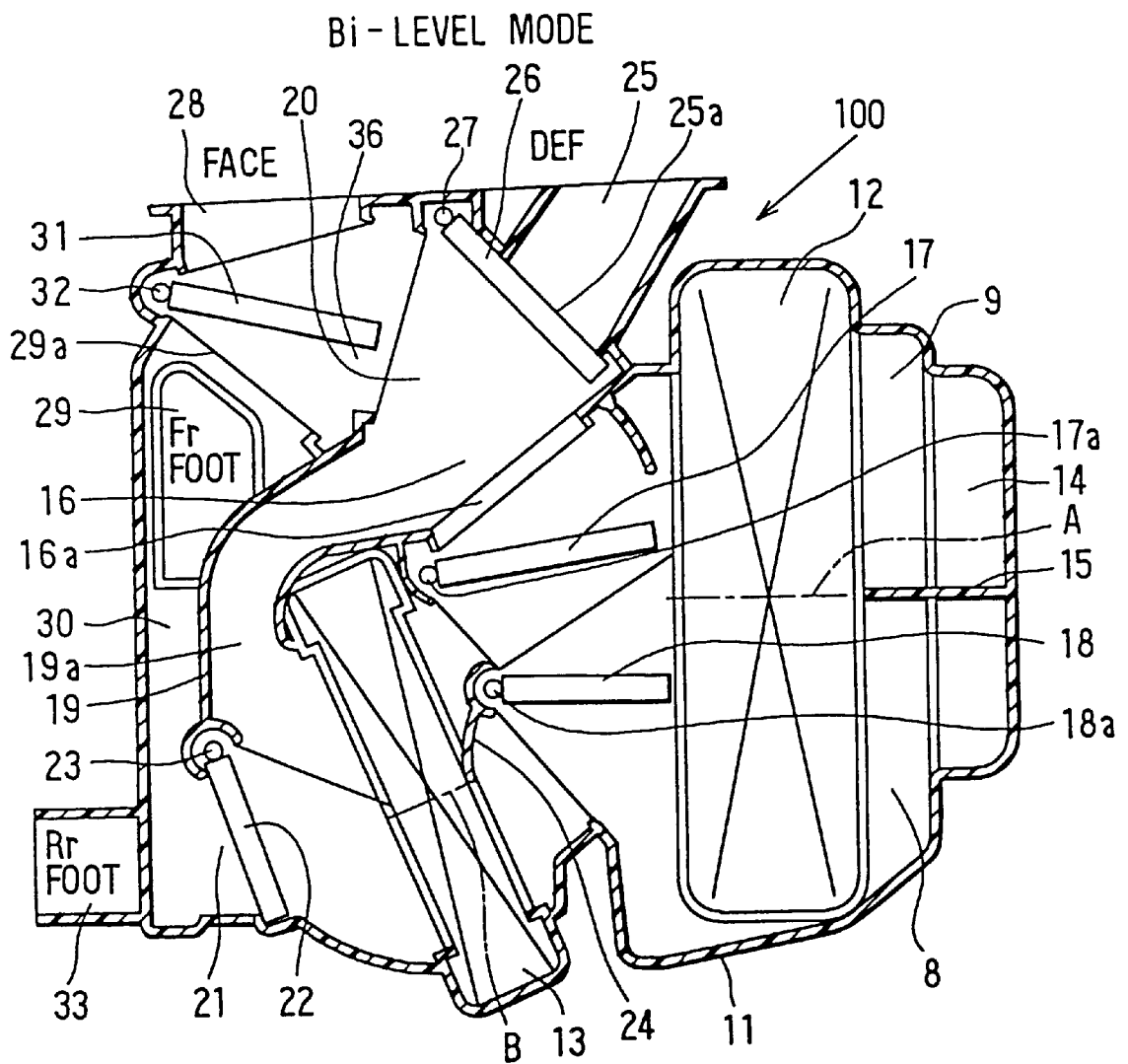


FIG. 9

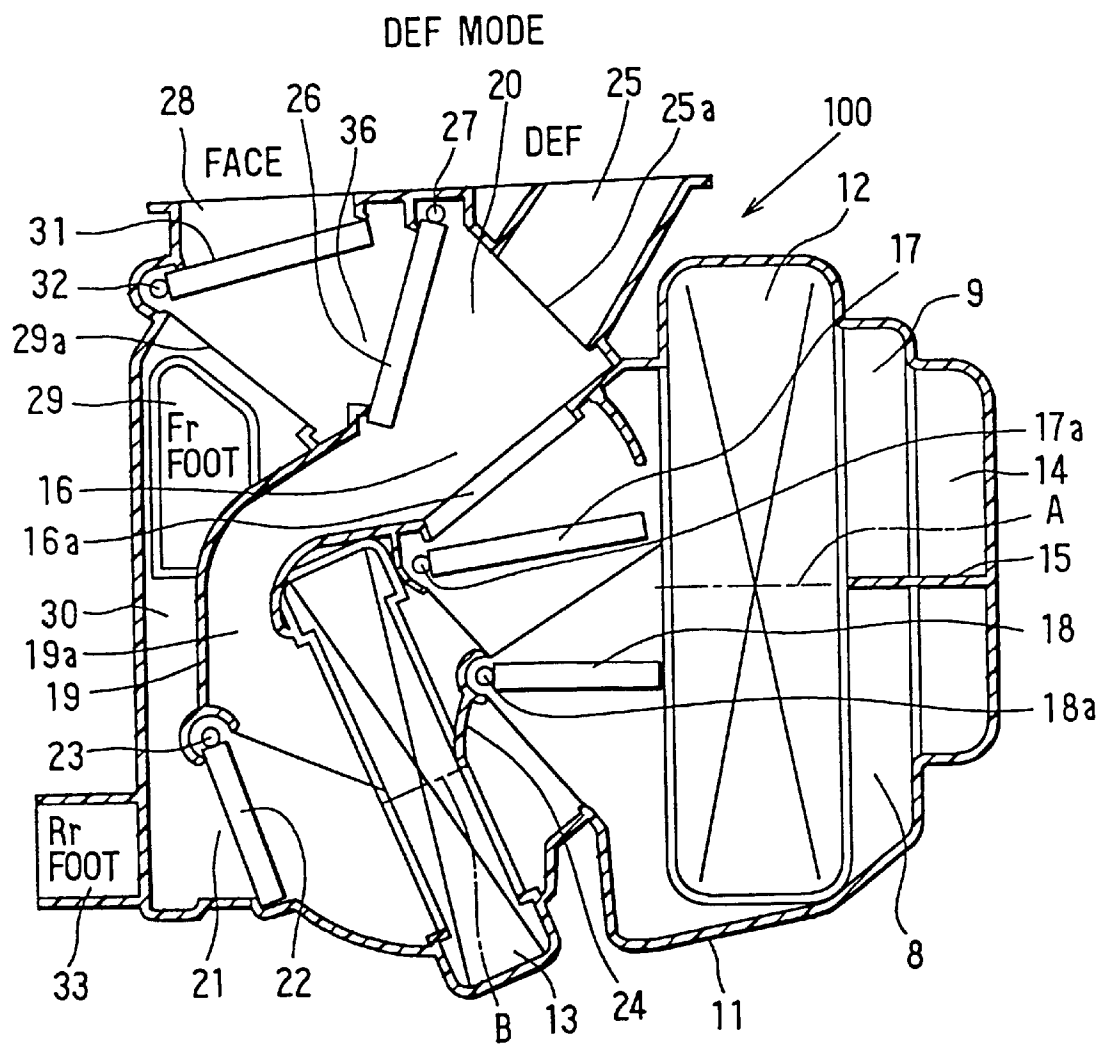


FIG.10

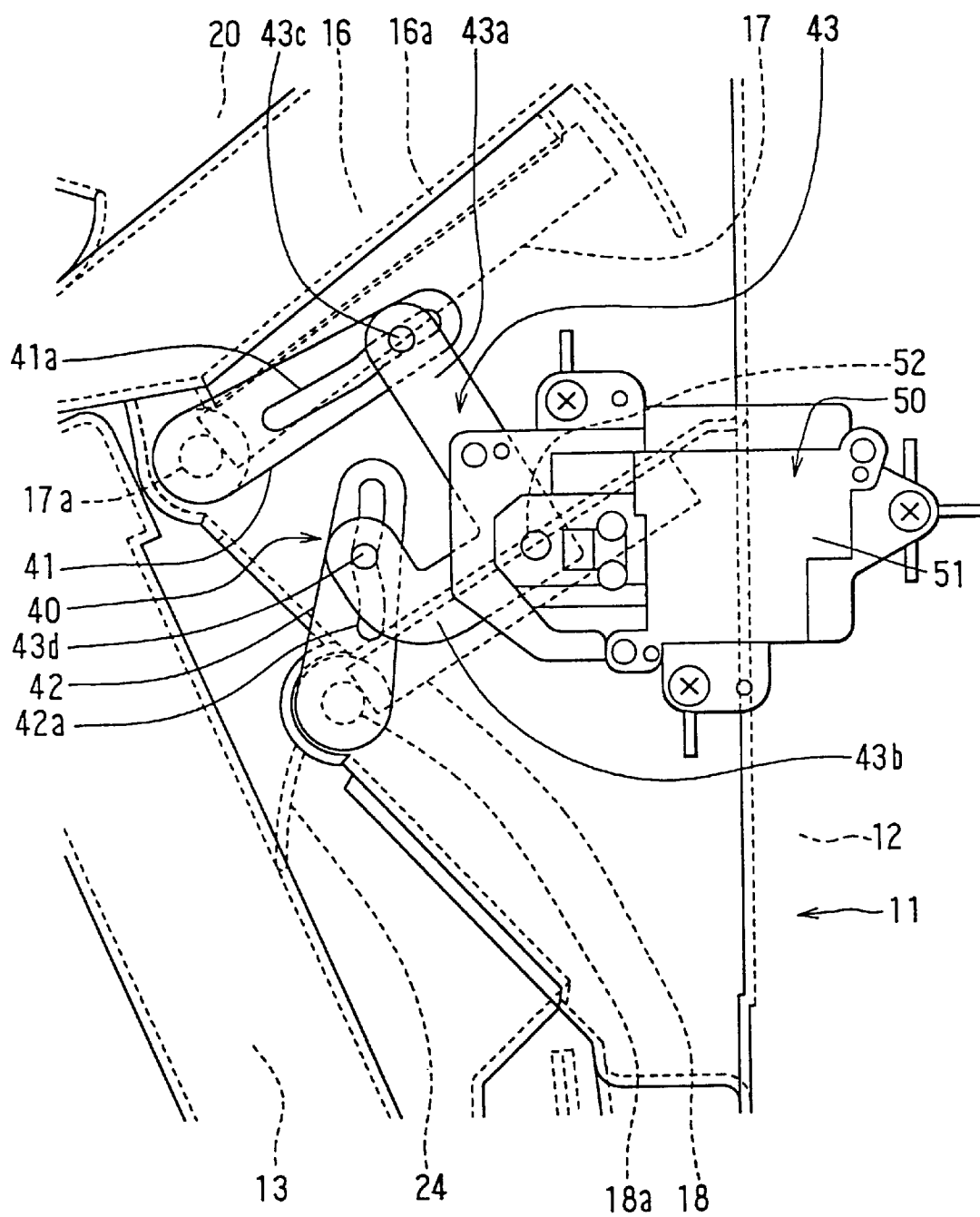


FIG. 11

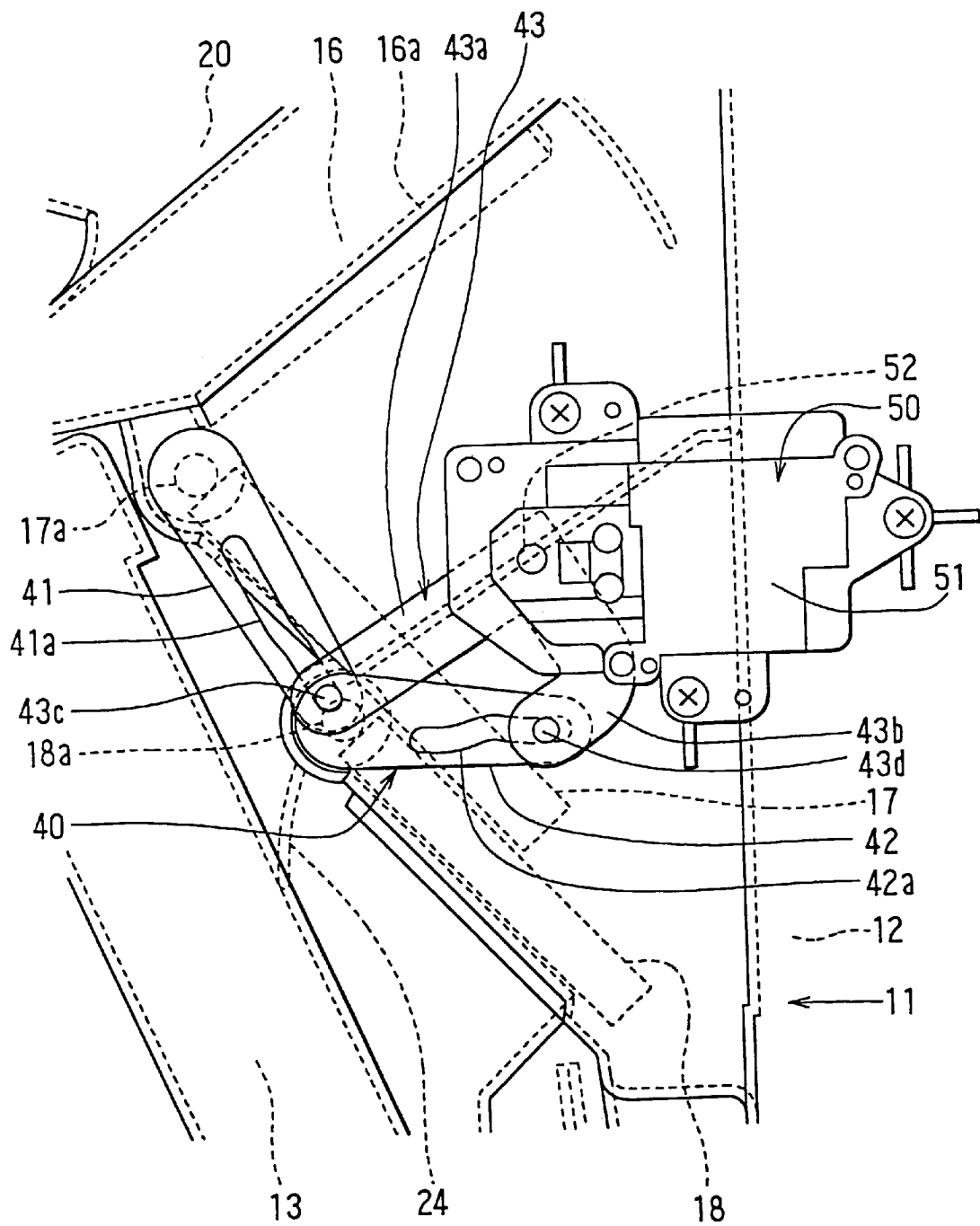


FIG. 12

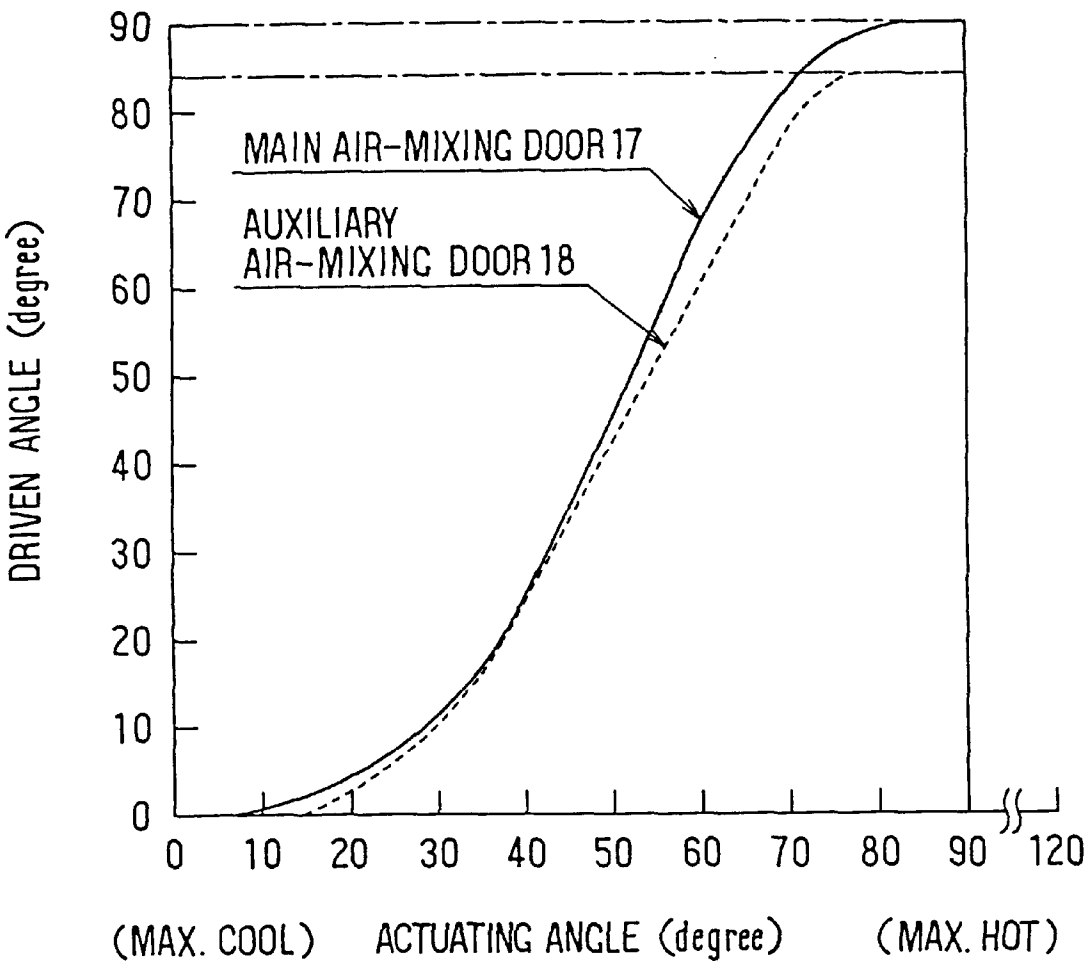


FIG. 13

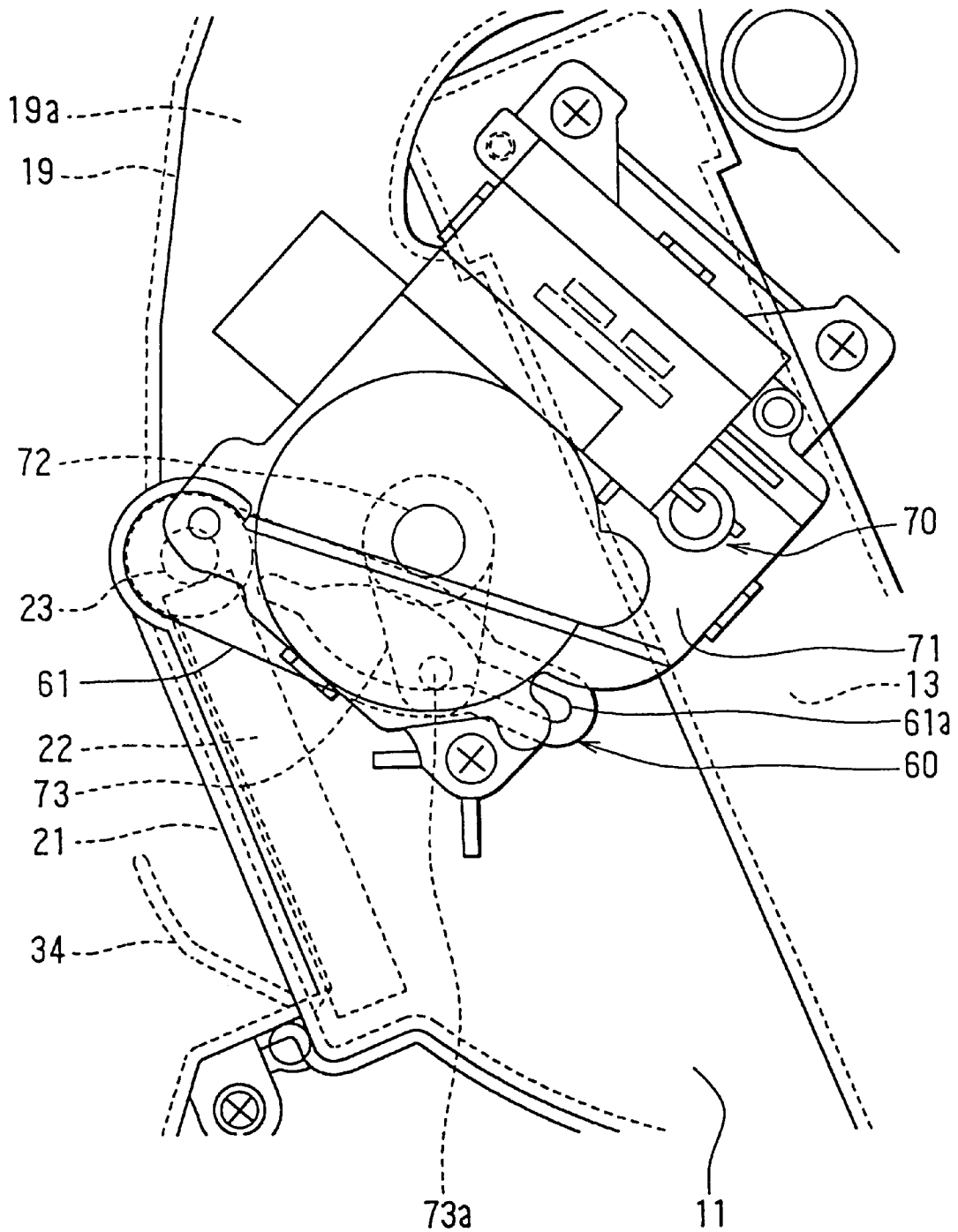


FIG. 14

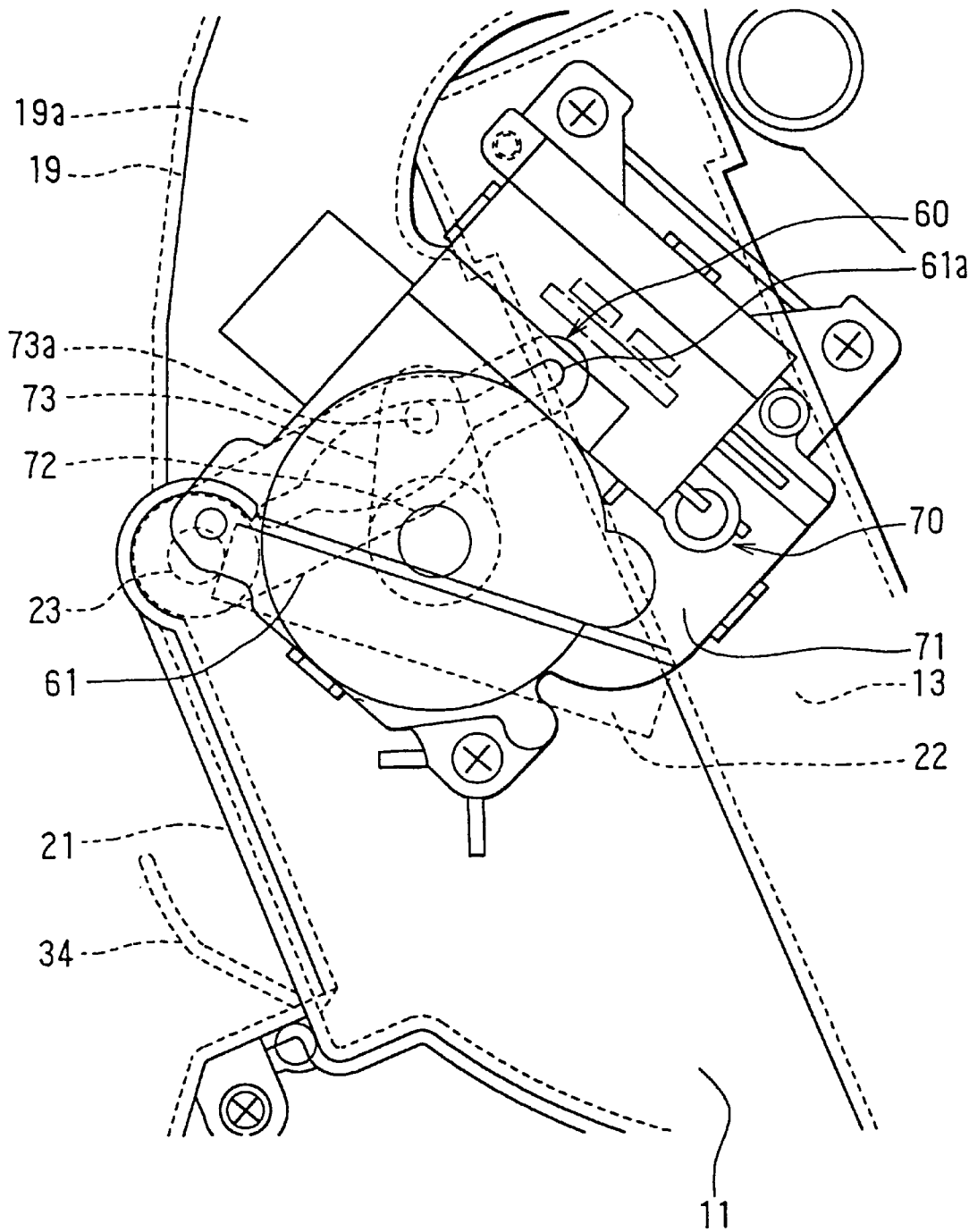


FIG. 15A

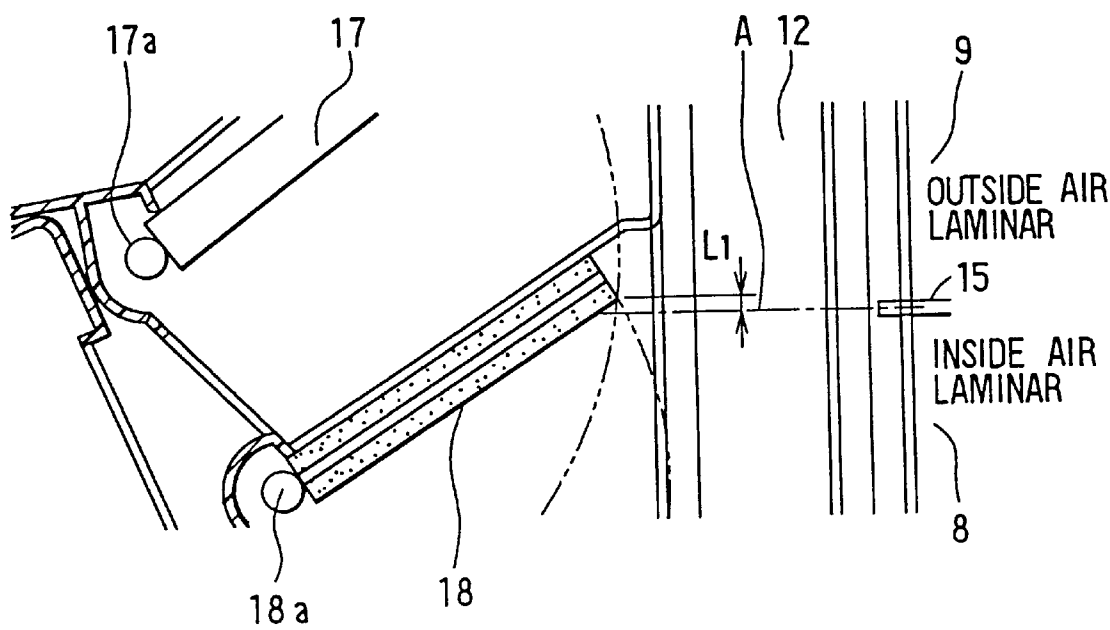


FIG. 15B

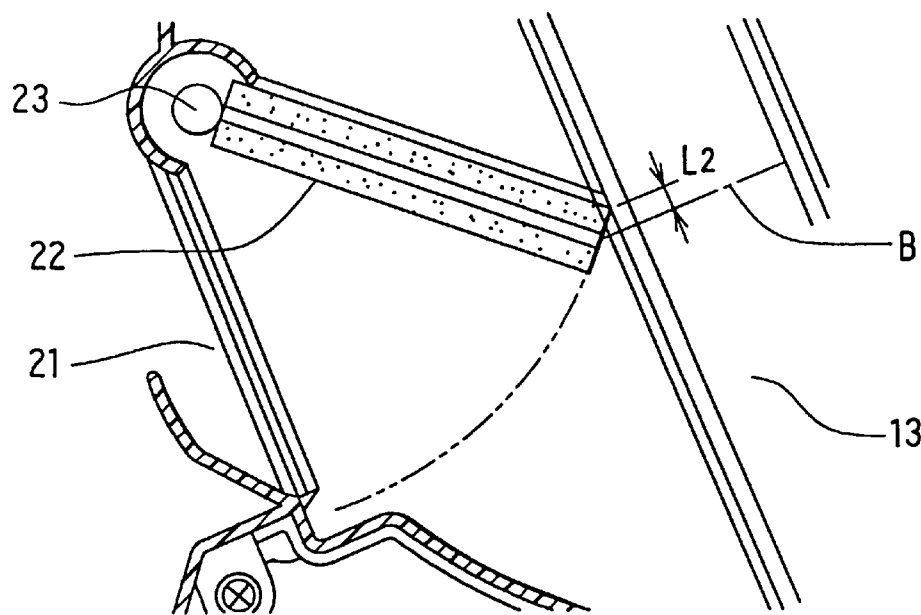


FIG. 16

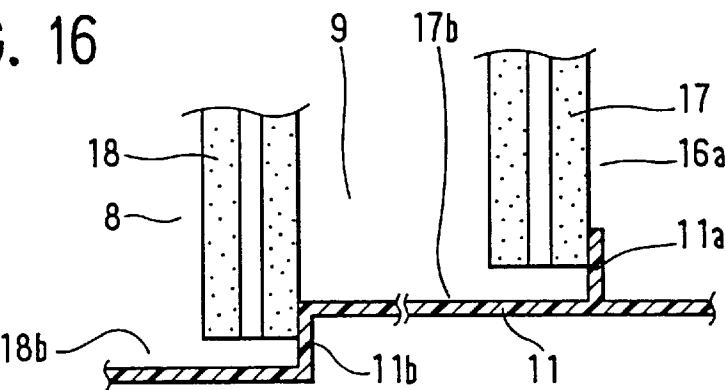


FIG. 17

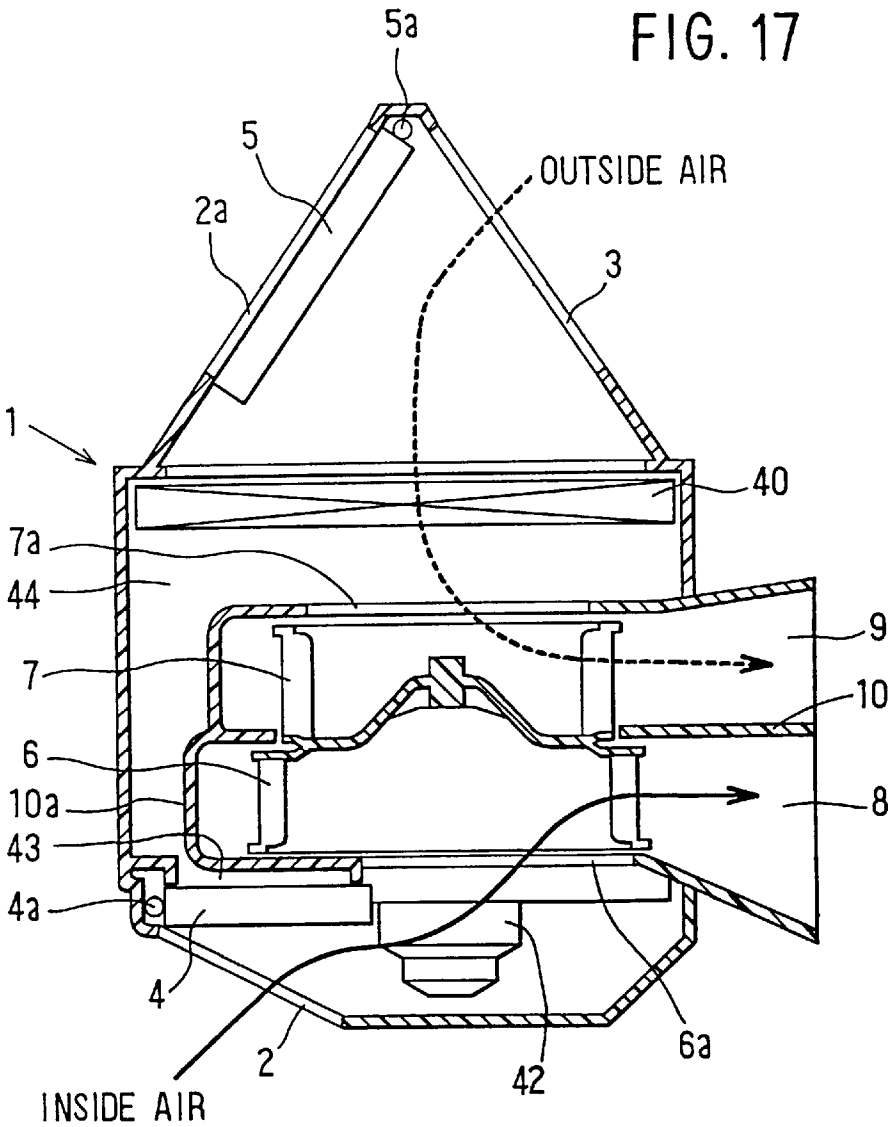


FIG. 18

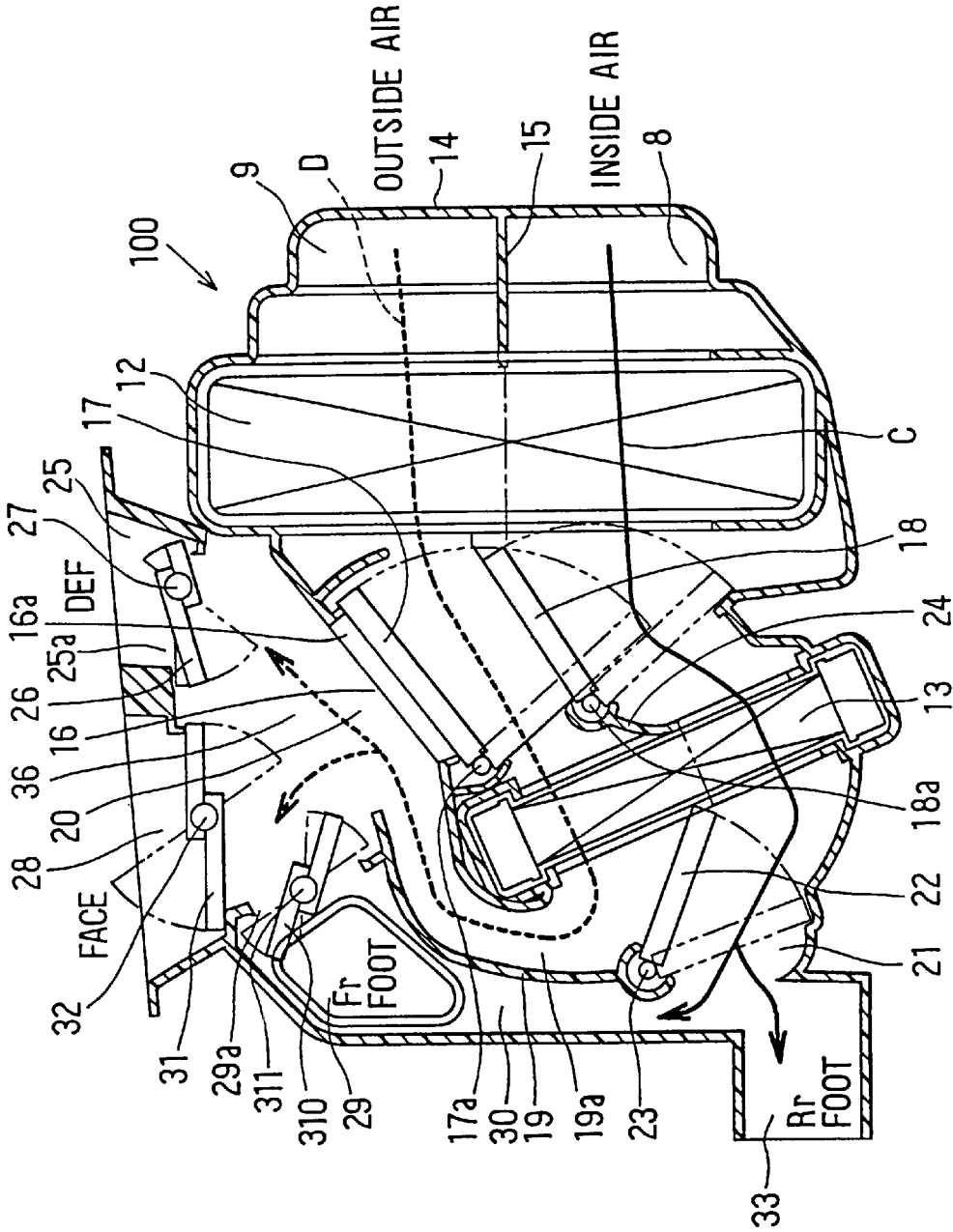


FIG. 19A

WHOLE OUTSIDE
AIR MODE

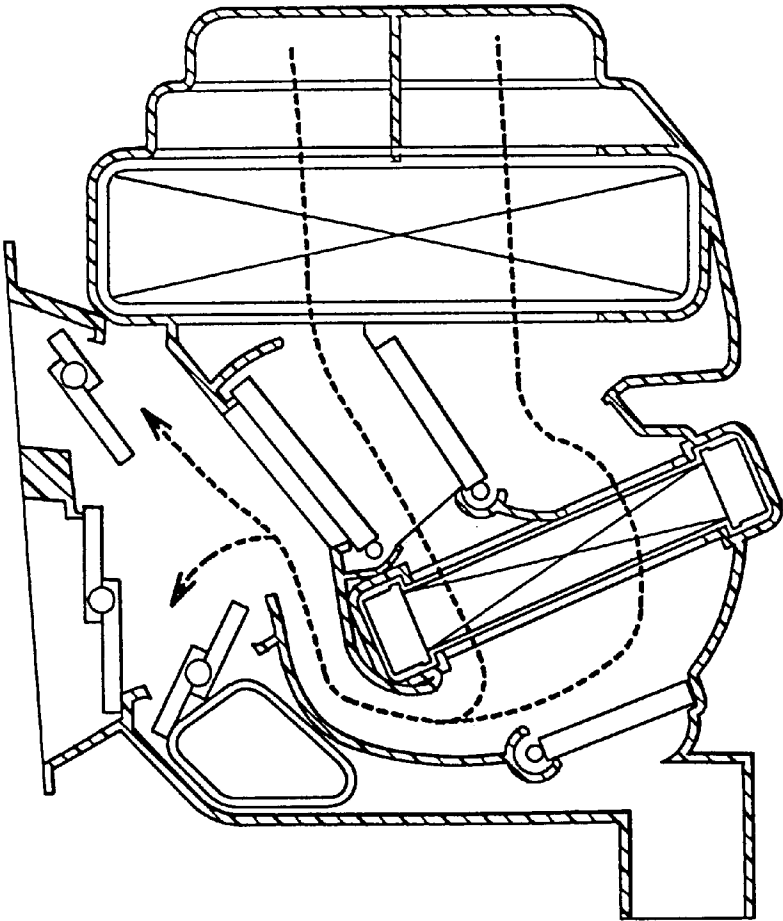


FIG. 19B

OUTSIDE AIR

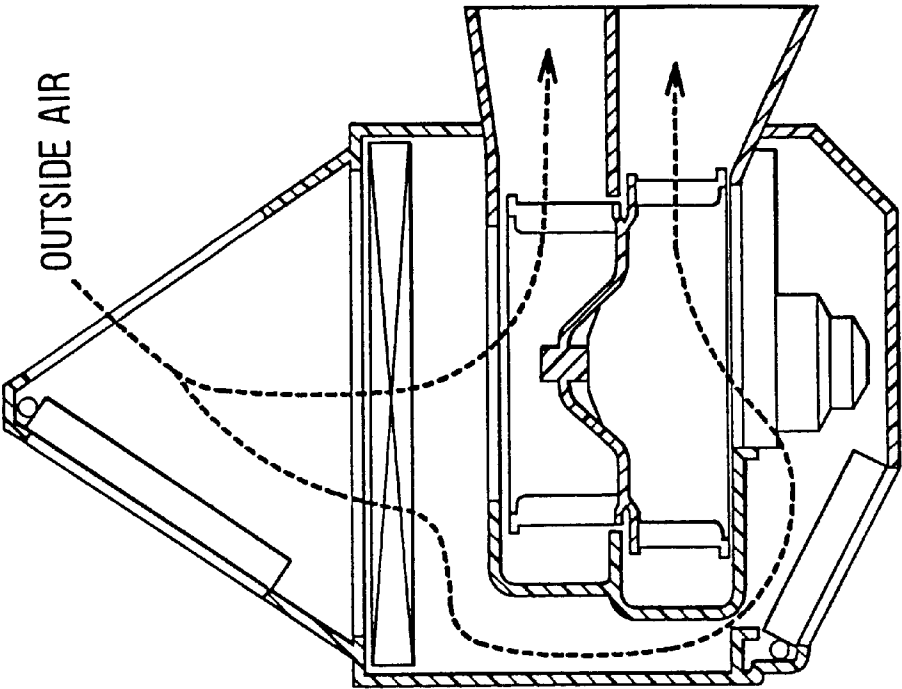


FIG. 20B

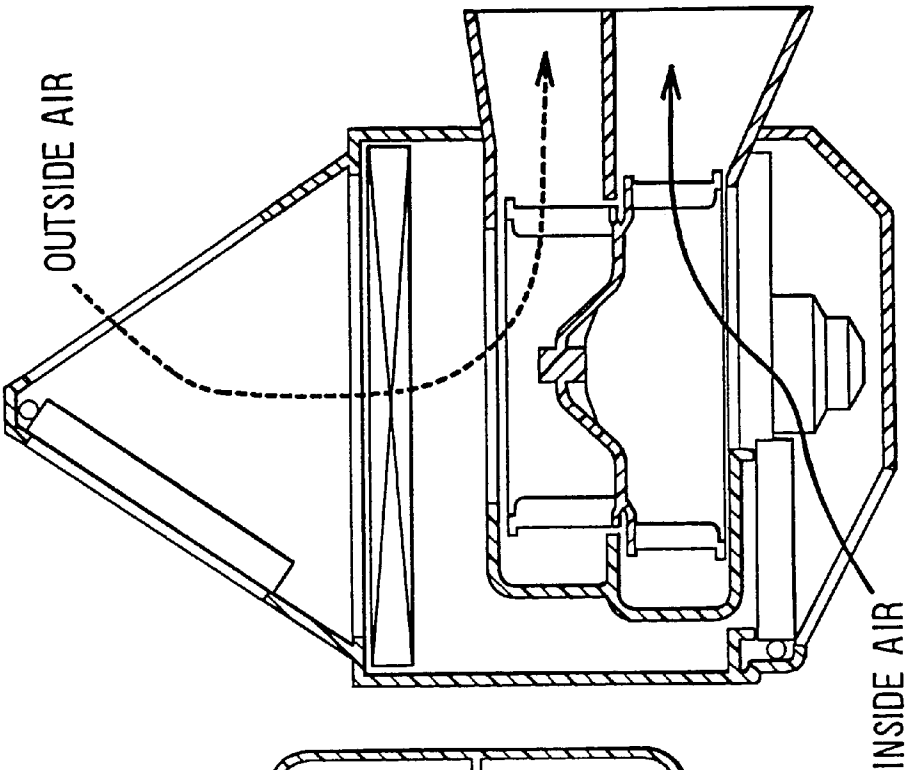


FIG. 20A

INSIDE AIR/OUTSIDE AIR
DOUBLE LAMINER MODE

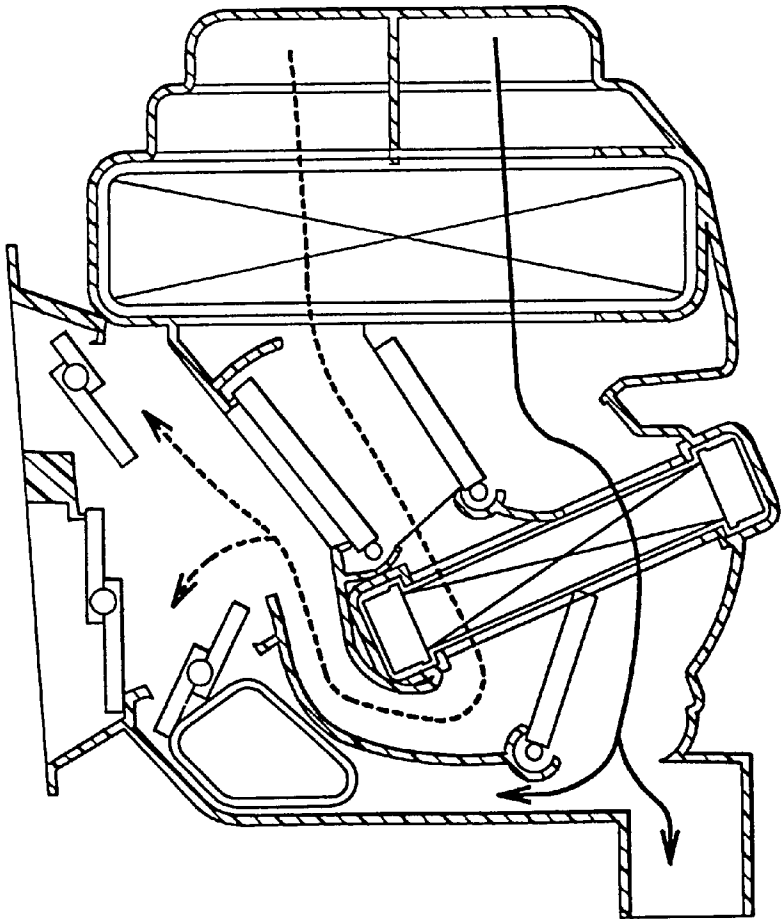


FIG. 21A

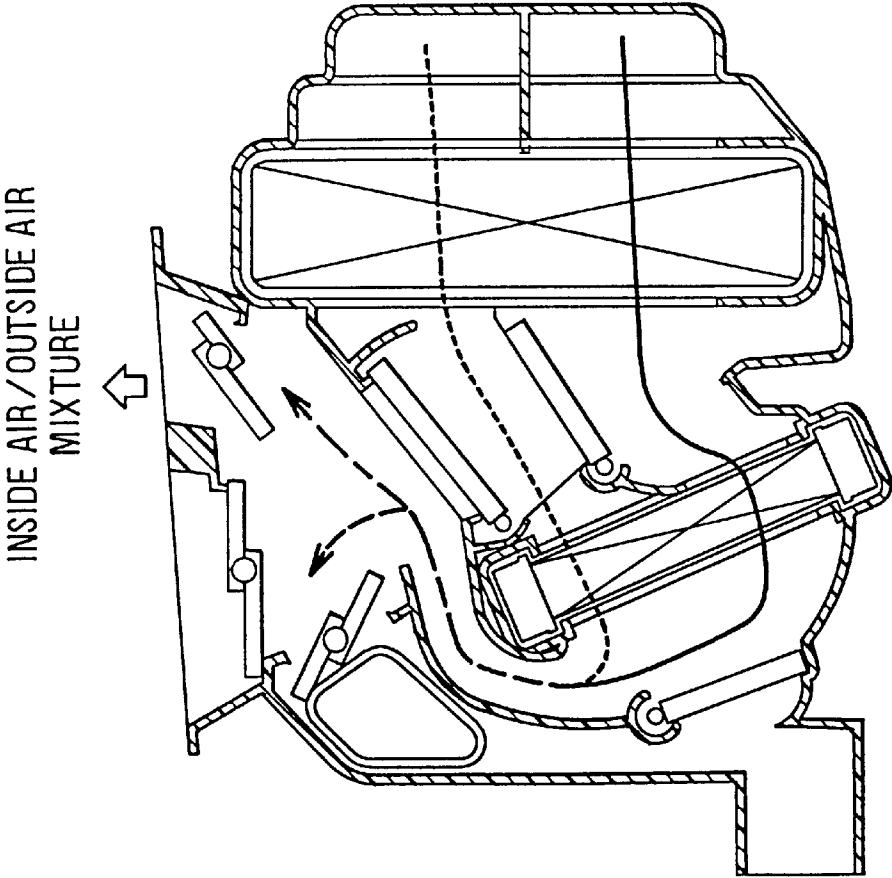


FIG. 21B

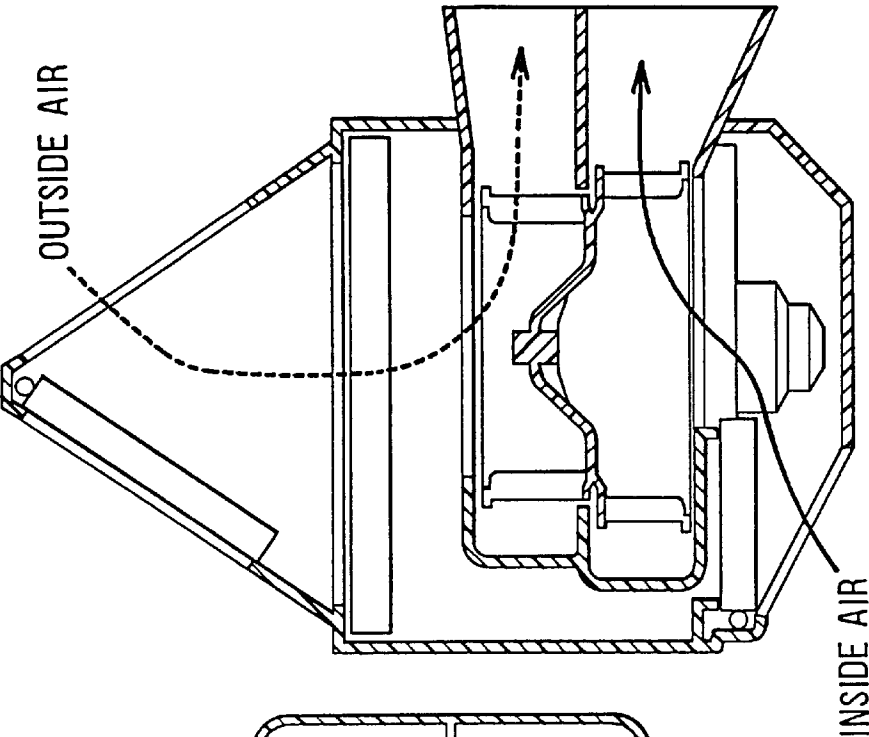


FIG. 22B

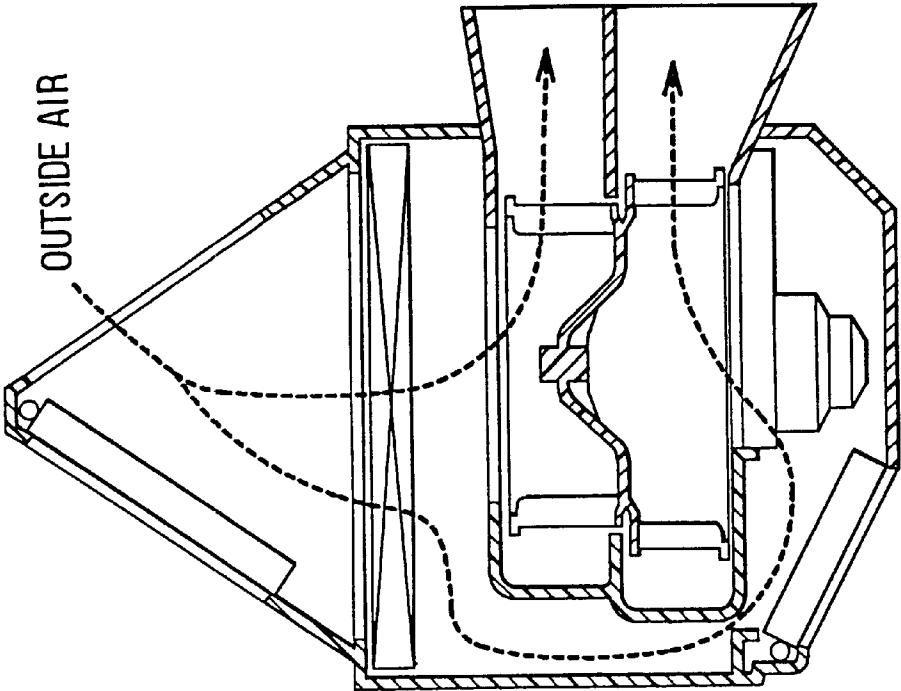


FIG. 22A

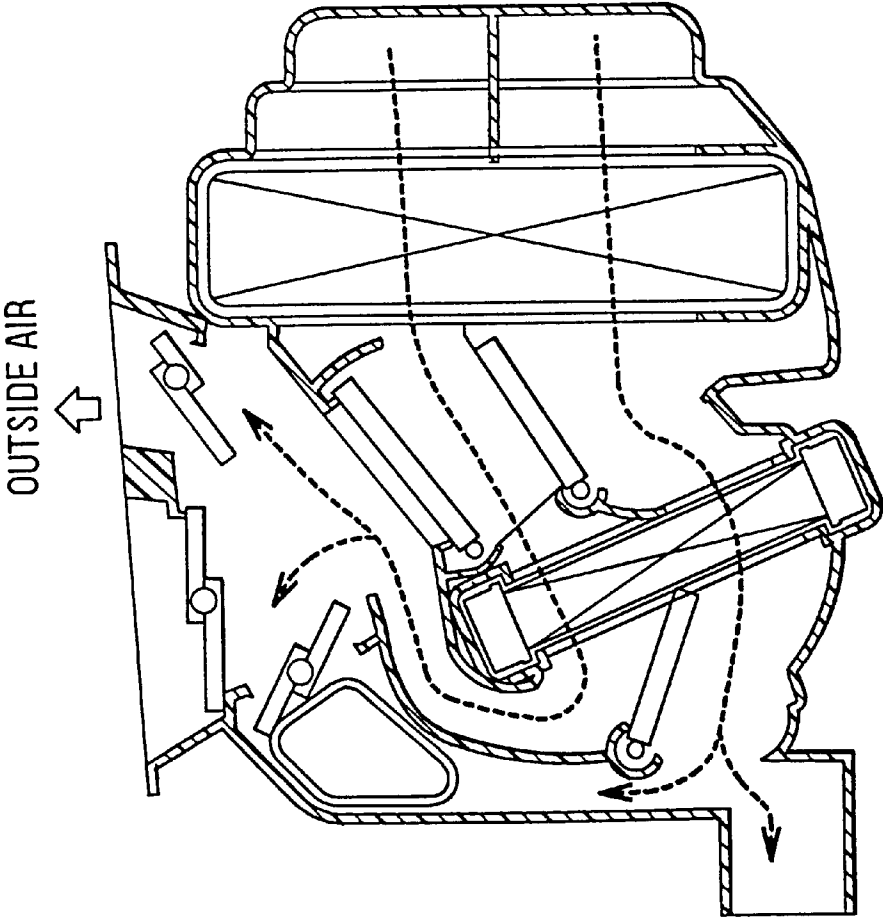


FIG. 23

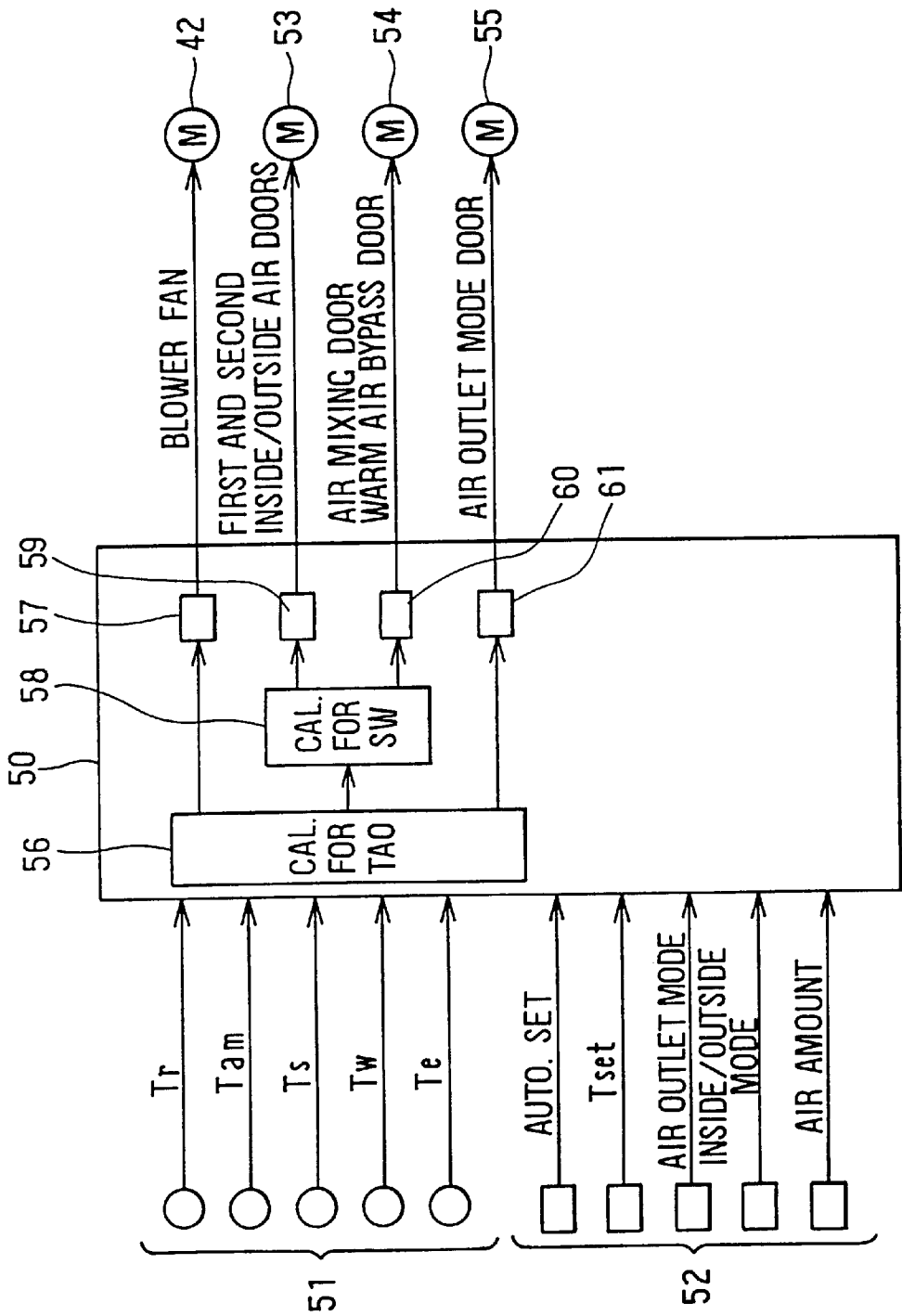


FIG. 24

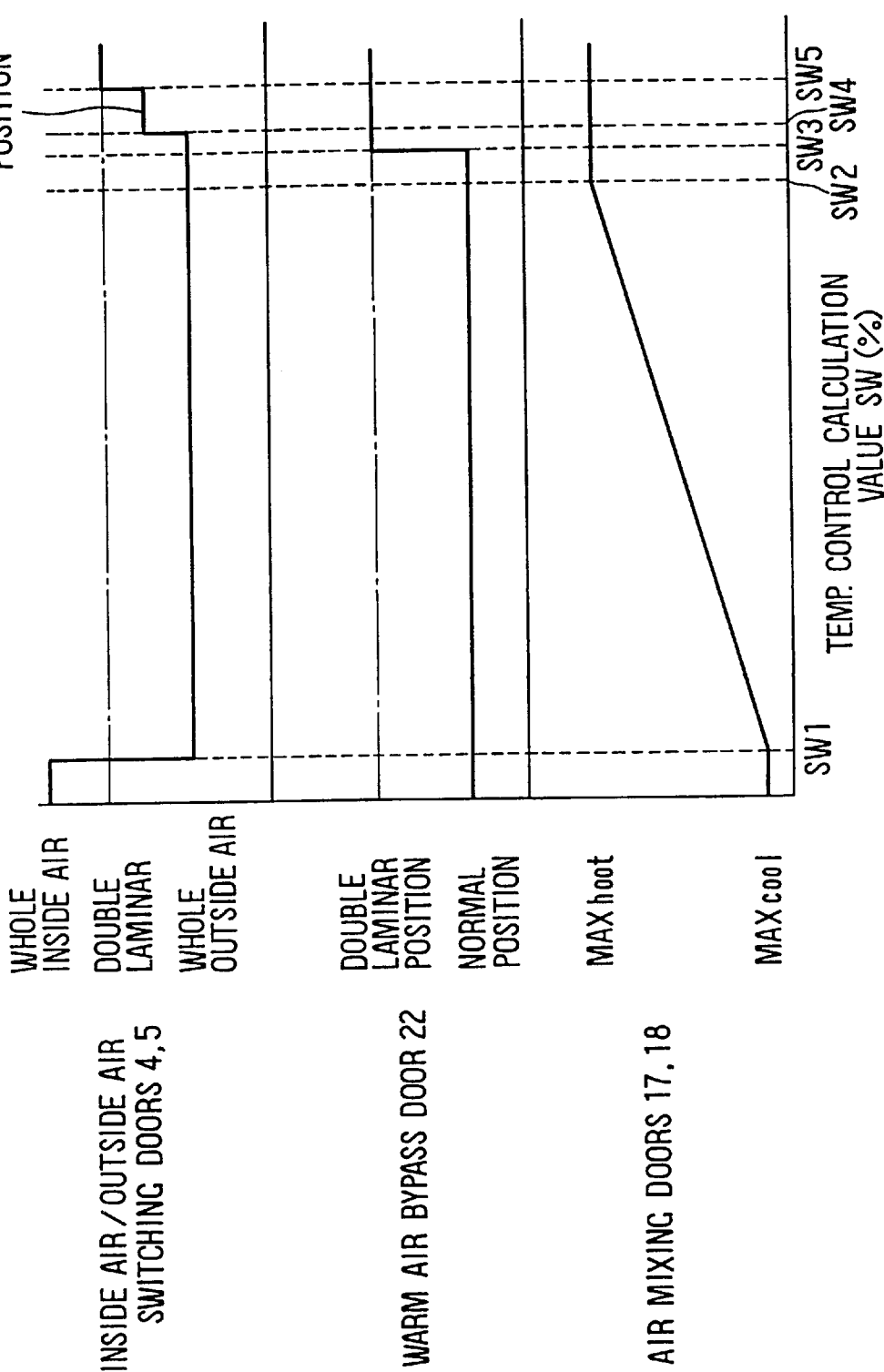


FIG. 25

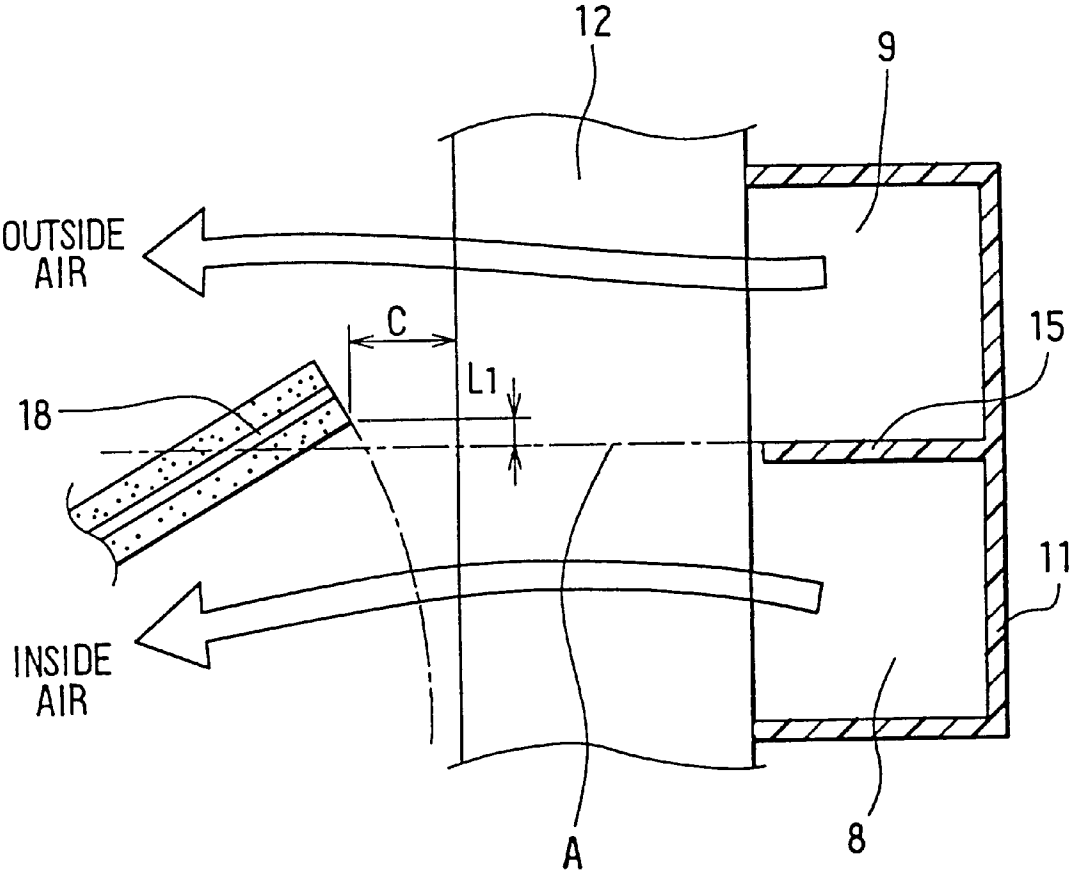


FIG. 26

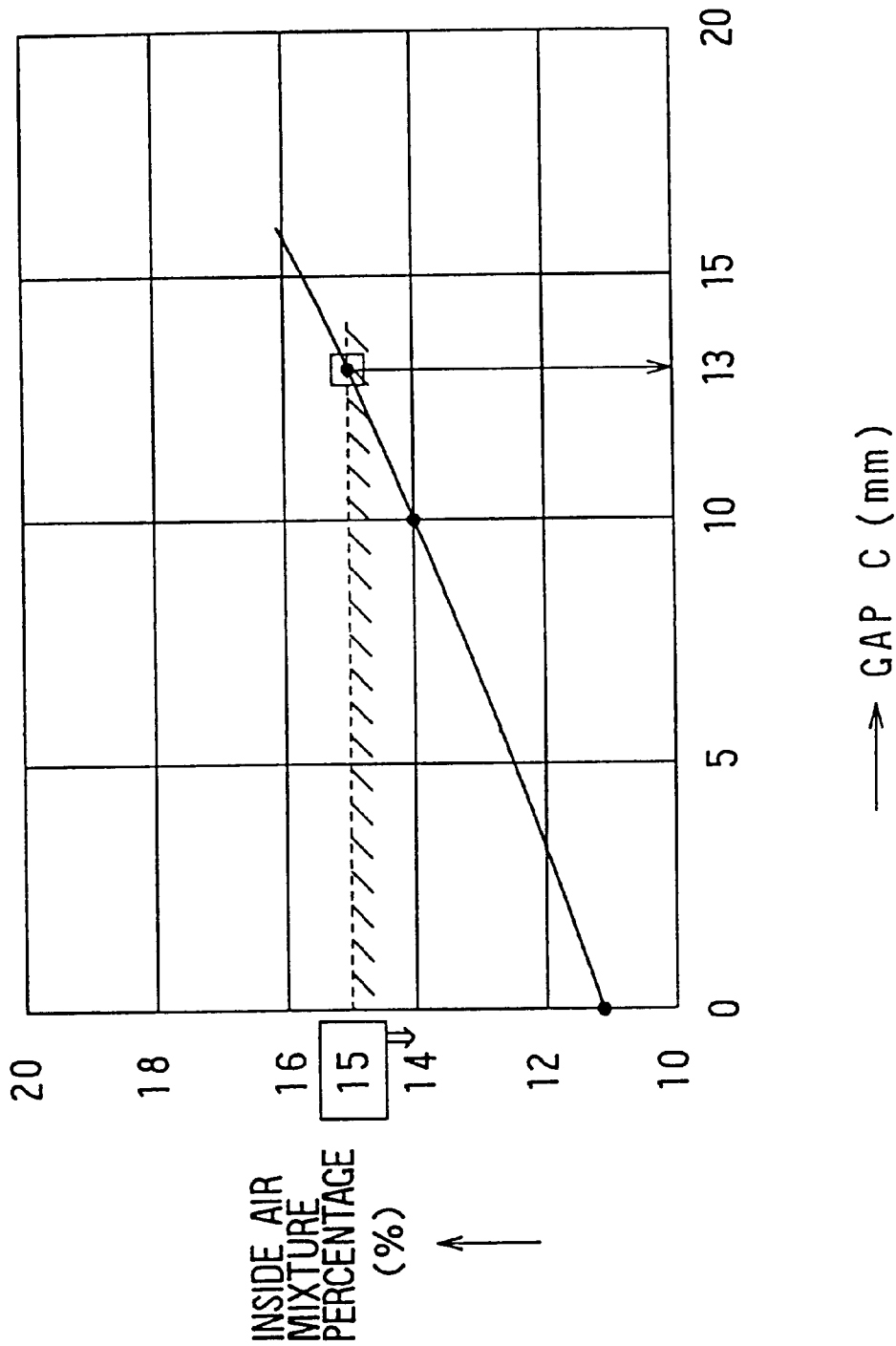


FIG. 27

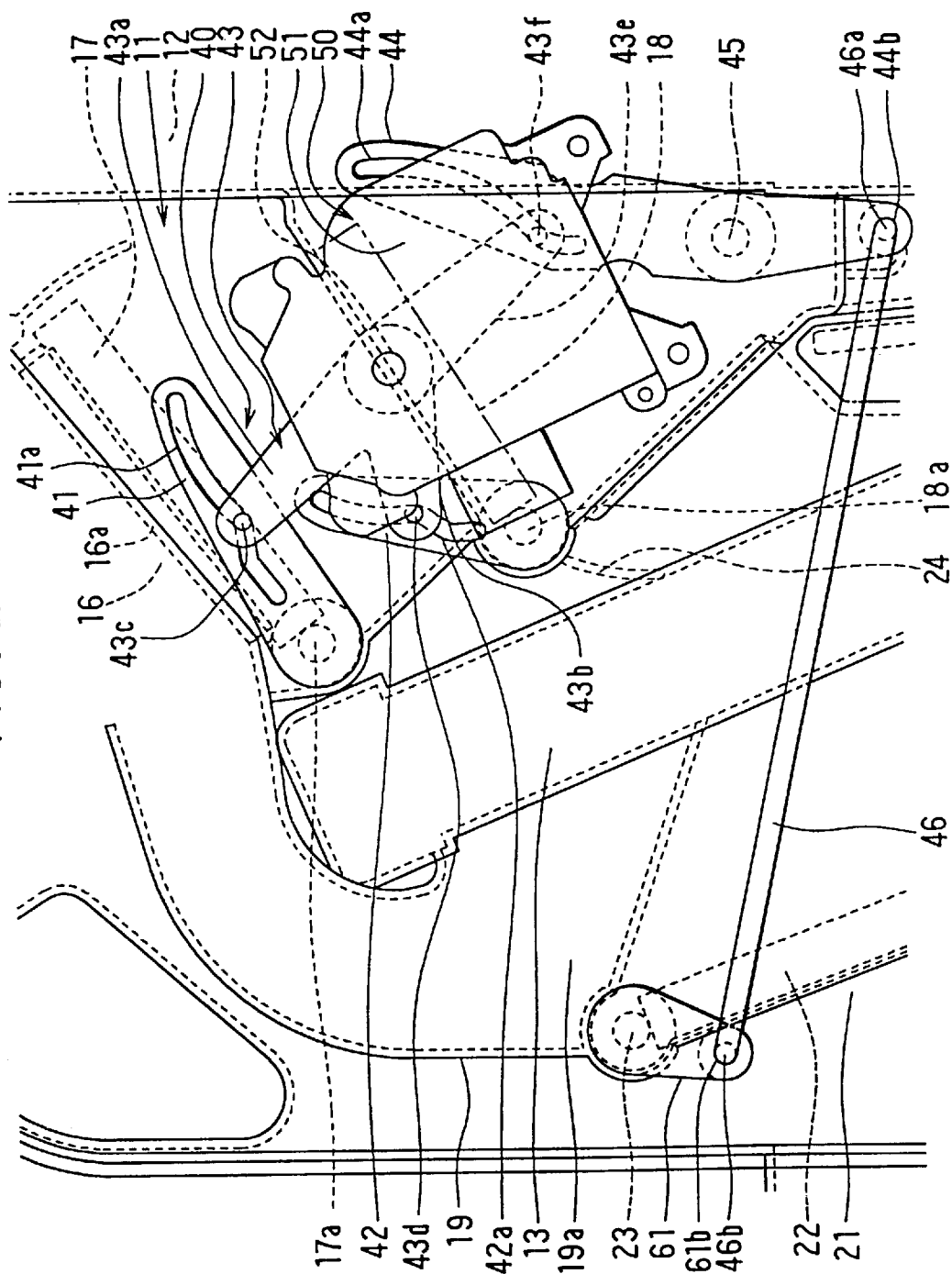


FIG. 28

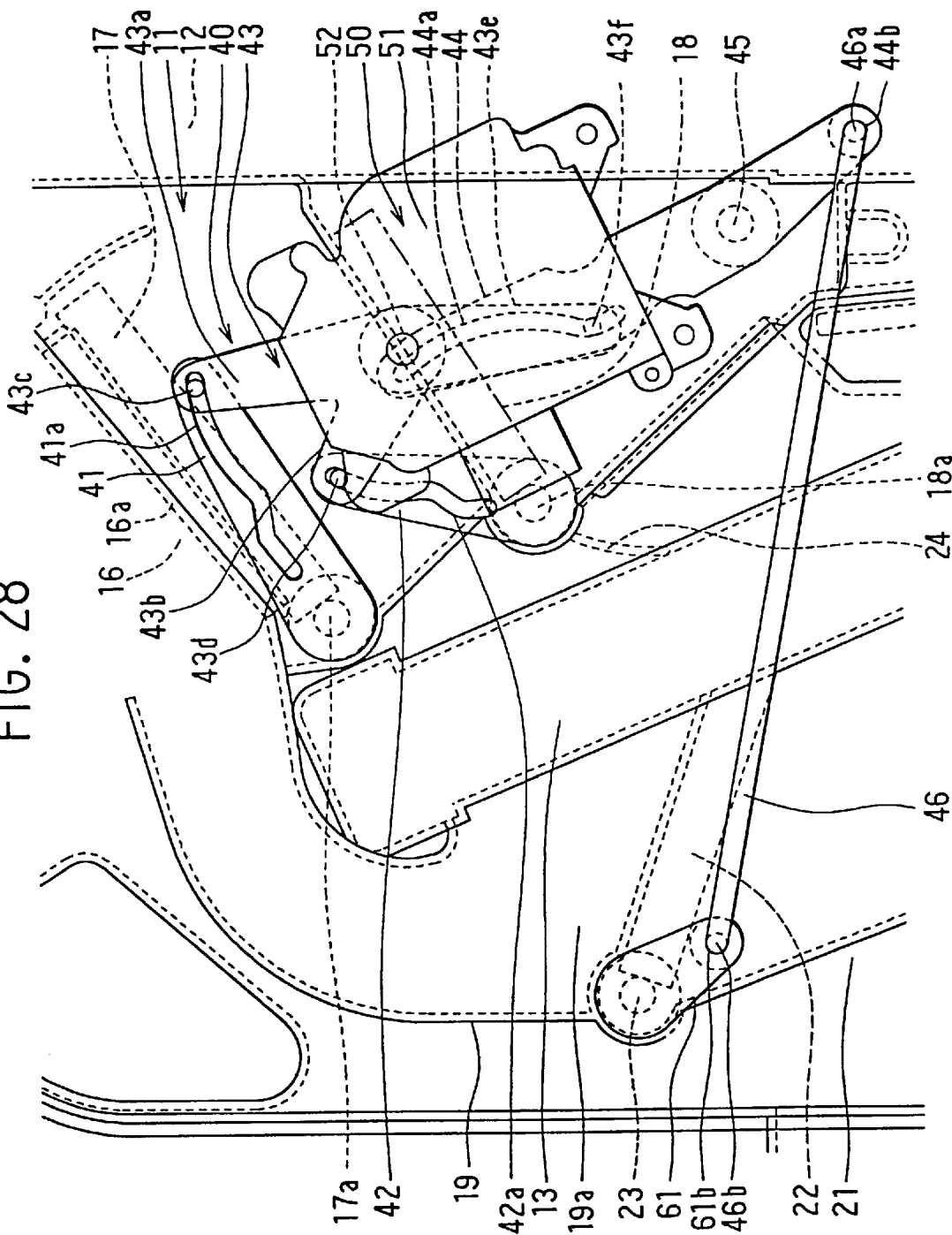
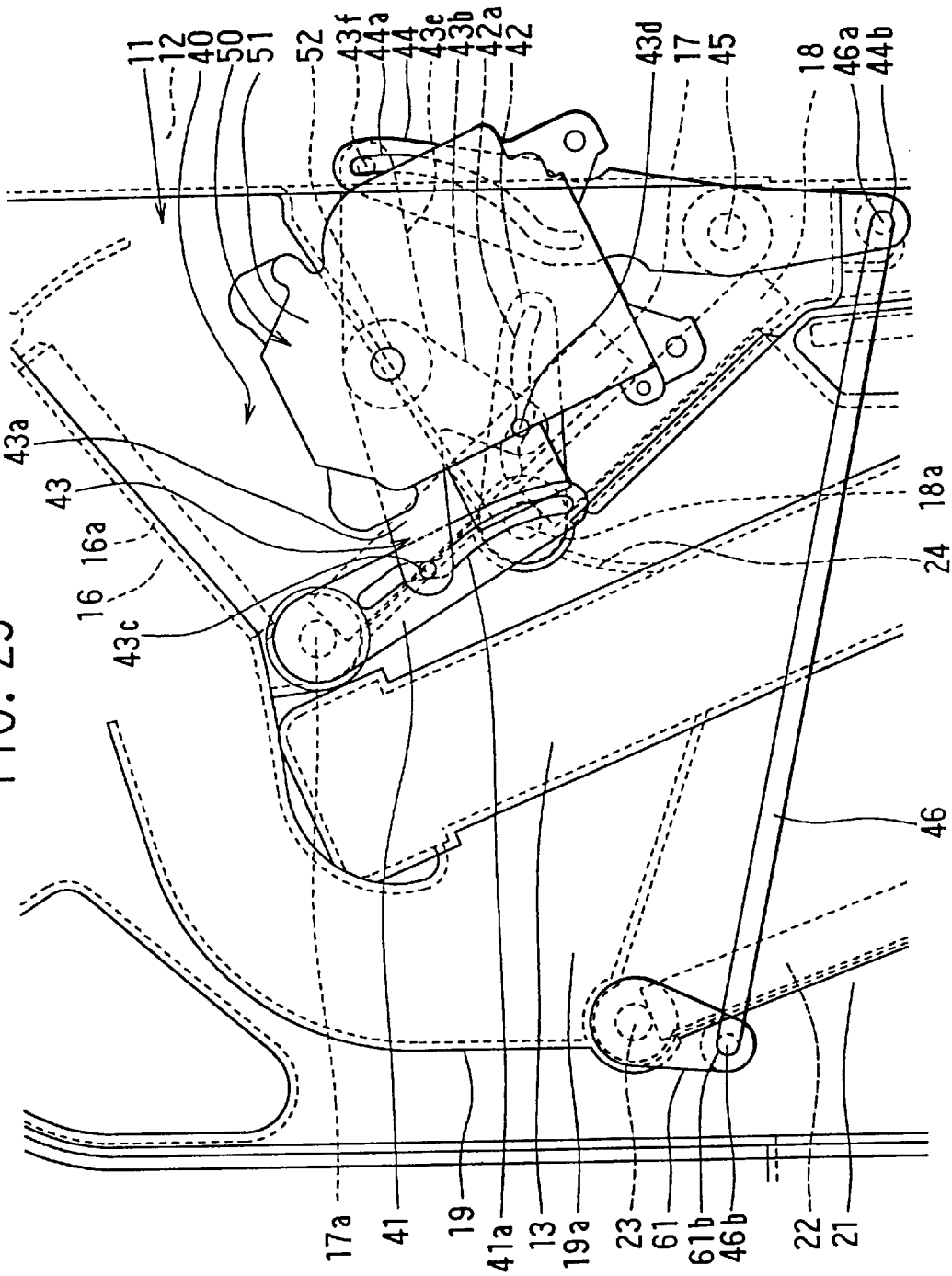


FIG. 29



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AIR CONDITIONING APPARATUS FOR VEHICLE

CROSS-REFERENCE TO RELATED APPLICATIONS

This application is based on and claims priority of Japanese Patent Applications of Nos. Hei. 8-266376 filed on Oct. 7, 1996, Hei. 8-287016, filed on Oct. 29, 1996, Hei. 8-343410 filed on Dec. 24, 1996, and Hei. 9-91003 filed on Apr. 9, 1997, the contents of which are incorporated herein by reference.

BACKGROUND OF THE INVENTION

1. Field of the Invention

This invention relates to an air conditioning apparatus, in which a passage of conditioned air is partitioned into a first air passage at a side of inside air and a second air passage at a side of outside air. A high-temperature inside air having been heated re-circulates and is blown out from a foot air outlet, and outside air having a low-humidity is blown out from a defroster air outlet, to improve a heating capacity and to defrost a windshield.

2. Description of Related Art

In a conventional air conditioning apparatus as disclosed in JP-A-5-124426, there are formed at one end side of an air conditioning case an inside-air inlet for introducing inside air and an outside-air inlet for introducing outside air and at the other end side of the air conditioning case a foot air outlet for blowing air toward a foot area, a defroster air outlet for blowing air toward an inner surface of a windshield, and a face air outlet for blowing air toward a face area.

In the air conditioning case, there is provided a partition plate for partitioning an interior of the air conditioning case into a first air passage extending from the outside-air suction port to the face air outlet and the foot air outlet and a second air passage extending from the outside-air suction port to the defroster air outlet.

Further, in each of the first and the second air passages, there are provided a heating heat exchanger, a bypass passage bypassing the heating heat exchanger, and an air mixing door. The air mixing door includes a door at a side of the first air passage, and another door at a side of the second air passage, which are integrally provided with a single rotary shaft rotatably disposed over both air passages.

When any one of the face mode, the bi-level mode, and the foot mode is selected as air outlet mode, if the inside air/outside air introduction mode is set to the inside-air circulation mode, the inside air is introduced into both air passages, whereas if the mode is set to the outside-air introduction mode, the outside air is introduced into both air passages.

Further, when the foot/defroster mode is selected as the air outlet mode, there is set an inside air/outside air double laminar mode in which the inside air is introduced into the first air passage and the outside air is introduced into the second air passage. In this way, because a passenger compartment is heated by a recirculation of the inside air which has been already heated, the heating performance is improved. Further, because the outside air having a low humidity is blown toward the windshield, it is possible to secure the defrosting performance of the windshield certainly.

In recent years, as for an air conditioning apparatus for a vehicle, in view of a limited space for being mounted on a vehicle as well as a cost reduction, a compactness of an air

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conditioning unit mounted on a vehicle and a simplicity of the construction have become significant problems. In the above-described prior art, it is necessary to additionally provide the partition plate for partitioning the first air passage and the second air passage in addition to the general construction elements of the air conditioning unit. As a result, the construction is complicated, the cost increases, and the air conditioning unit is large-sized to avoid an interference of the partition plate and the door members.

SUMMARY OF THE INVENTION

In view of the above-described problem, an object of the present invention is to downsize an air conditioning unit and to simplify a construction thereof, in the air conditioning apparatus for a vehicle, in which there is set a double laminar mode for partitioning inside air and outside air in an air conditioning case.

According to a first aspect of the present invention, in an air outlet mode where both of a first opening portion for blowing air toward a lower portion of a passenger compartment and a second opening portion for blowing air toward an inner surface of a windshield are opened, when the maximum heating state is set, an air passage is partitioned into a first air passage through which inside air flows and a second air passage through which outside air flows, in such a manner that the first air passage is communicated with the first opening portion, the second air passage is communicated with the second opening portion. Further, there are formed a first warm air passage for introducing the warm air having passed through a heating heat exchanger into the first and the second air opening portions and a second warm air passage for introducing the warm air having passed through the heating heat exchanger directly into the first air opening portion while bypassing the first warm air passage. An inlet portion of the second warm air passage is opened and closed by the warm air bypass door. In a double laminar mode where the first air passage and the second air passage are partitioned, the warm air bypass door is operated to a position where the inlet portion is opened and the warm air passage is partitioned into the first air passage and the second air passage.

In this way, because the warm air bypass door itself functions as a movable partition member for partitioning the inside air and the outside air, a space for providing a stationary partition member can be reduced, so that an air conditioning unit can be downsized, and a construction thereof can be simplified.

According to a second aspect of the present invention, as an air mixing door for adjusting a ratio of an amount of air passing through a heating heat exchanger and an amount of air passing through a cool air bypass passage, there are provided a main air mixing door and an auxiliary air mixing door, both of which are disposed at an air upstream side of the heating heat exchanger and operated with each other. Further, in a double laminar mode where a first air passage and a second air passage are partitioned, the main air mixing door fully closes the cool air bypass passage, and the auxiliary air mixing door partitions a space at an air upstream side of the heating heat exchanger into the first air passage and the second air passage.

In this way, because the air mixing door itself functions as a movable partition member for partitioning the inside air and the outside air, a space for providing a stationary partition member can be reduced. Further, a space for rotating two air mixing doors can be commonly used. Therefore, an air conditioning unit can be downsized, and a construction thereof can be simplified.

BRIEF DESCRIPTION OF THE DRAWINGS

Additional objects and advantages of the present invention will be more readily apparent from the following detailed description of preferred embodiments thereof when taken together with the accompanying drawings in which:

FIG. 1 is a view showing an entire construction of a ventilation system according to a first embodiment of the present invention;

FIG. 2 is a cross sectional view of an air conditioning unit of FIG. 1;

FIG. 3 is a cross sectional view showing a state of a double laminar mode in a foot air outlet mode according to the first embodiment;

FIG. 4 is a cross sectional view showing a state of a normal mode in the foot air outlet mode according to the first embodiment;

FIG. 5 is a cross sectional view showing a state of a double laminar mode in a foot/defroster air outlet mode according to the first embodiment;

FIG. 6 is a cross sectional view showing a state of a normal mode in the foot/defroster air outlet mode according to the first embodiment;

FIG. 7 is a cross sectional view showing a state of a face air outlet mode according to the first embodiment;

FIG. 8 is a cross sectional view showing a state of a bi-level air outlet mode according to the first embodiment;

FIG. 9 is a cross sectional view showing a state of a defroster air outlet mode according to the first embodiment;

FIG. 10 is an enlarged view showing a main portion of an air mixing door actuating mechanism in the maximum heating state according to the first embodiment;

FIG. 11 is an enlarged view showing a main portion of an air mixing door actuating mechanism in the maximum cooling state according to the first embodiment;

FIG. 12 shows characteristics of an operation of the air mixing door mechanism according to the first embodiment;

FIG. 13 is an enlarged view showing a main portion of a warm air bypass mixing door actuating mechanism in a state where a warm air bypass inlet portion is closed, according to the first embodiment;

FIG. 14 is an enlarged view showing a main portion of the warm air bypass mixing door actuating mechanism in a double laminar mode, according to the first embodiment;

FIG. 15A is an enlarged view showing an auxiliary air mixing door according to the first embodiment, and FIG. 15B is an enlarged view showing a warm air bypass door according to the first embodiment;

FIG. 16 is a cross sectional view taken along the line XVI—XVI of FIG. 2;

FIG. 17 is a cross sectional view showing a blower unit according to a second embodiment of the present invention;

FIG. 18 is a cross sectional view showing an air conditioning unit according to the second embodiment;

FIG. 19A is a cross sectional view of the air conditioning unit in a whole outside air mode according to the second embodiment, and FIG. 19B is a cross sectional view of the blower unit in a whole outside air mode according to the second embodiment;

FIG. 20A is a cross sectional view of the air conditioning unit in a double laminar mode according to the second embodiment, and FIG. 20B is a cross sectional view of the blower unit in a double laminar mode according to the second embodiment;

FIG. 21A is a cross sectional view of the air conditioning unit in a switching process between the whole outside air mode and the double laminar mode according to a comparison sample, and FIG. 21B is a cross sectional view of the blower unit in a switching process between the whole outside air mode and the double laminar mode according to a comparison sample;

FIG. 22A is a cross sectional view of the air conditioning unit in a switching process between the whole outside air mode and the double laminar mode according to the second embodiment, and FIG. 22B is a cross sectional view of the blower unit in a switching process between the whole outside air mode and the double laminar mode according to the second embodiment;

FIG. 23 is a diagram showing a system for electric control according to the second embodiment;

FIG. 24 is a graph showing control characteristics of door control according to the second embodiment;

FIG. 25 is an enlarged view showing a gap at a top end portion of the auxiliary air mixing door according to a third embodiment of the present invention;

FIG. 26 is a graph showing a relationship between the gap at the top end portion of the auxiliary air mixing door and an inside air mixture percentage, according to the third embodiment;

FIG. 27 is an enlarged view showing an actuating mechanism of an air mixing door and a warm air bypass door in the normal maximum heating state, according to a fourth embodiment of the present invention;

FIG. 28 is an enlarged view showing the door actuating mechanism of FIG. 27 in the double laminar mode, according to the fourth embodiment; and

FIG. 29 is an enlarged view showing the door actuating mechanism of FIG. 27 in the maximum cooling state, according to the fourth embodiment.

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS

Referring to the drawings, preferred embodiments of the present invention will be described.

A first embodiment of the present invention will be described with reference to FIGS. 1 and 2. In the first embodiment, the present invention is applied to an air conditioning apparatus mounted on a vehicle in which it is difficult to secure a sufficient heat source for heating operation, such as a vehicle having a diesel engine mounted thereon, an electric vehicle, and a hybrid vehicle.

FIG. 1 is a schematic view showing an entire construction of a ventilation system of an air conditioning apparatus, and FIG. 2 is a vertical cross section of an air-conditioning unit.

In FIG. 1, the ventilation system of the air conditioning apparatus is mainly divided into two parts of a blower unit 1 and an air conditioning unit 100. First, the blower unit 1 will be described. The blower unit 1 is disposed at a lower portion of a dashboard in a passenger compartment and at an offset position from a center portion toward a passenger's seat next to a driver's seat. The blower unit 1 is provided with a first and a second inside air introduction ports 2 and 2a for introducing inside air (air in the passenger compartment) and an outside air introduction port 3 for introducing outside air (air outside the passenger compartment). Those introduction ports 2, 2a and 3 are opened and closed by a first and a second inside air/outside air switching doors 4 and 5.

Those first and second inside air/outside air switching doors 4 and 5 are rotated with rotary shafts 4a and 5a,

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respectively, as a center thereof, and are operated by a link mechanism (not shown) and an actuator such as a servomotor in accordance with an inside air/outside air introduction mode control signal of the air conditioning apparatus.

In the blower unit 1, there are disposed a first (inside air) fan 6 and a second (outside air) fan 7 for blowing air introduced from the introduction ports 2, 2a, and 3. Both of fans 6 and 7 are composed of a centrifugal multi-blades fan (sirocco fan) and are rotated simultaneously by a single common electric motor (not shown).

FIG. 2 shows a state of a double laminar mode (described later). Because the first inside air/outside air switching door 4 opens the first inside air introduction port 2 and closes the outside air passage 3a from the outside air introduction port 3, the inside air is sucked into a suction port 6a of the first (inside air) fan 6. On the other hand, because the second inside air/outside air switching door 4 closes the second inside air introduction port 2a and opens the outside air passage 3b from the outside air introduction port 3, the outside air is sucked into a suction port 7a of the second (outside air) fan 7.

Therefore, in this state, the first fan 6 blows the inside air from the inside air introduction port 2 into a first (inside air) passage 8, and the second fan 7 blows the outside air from the outside air introduction port 3 into the second (outside air) passage 9. The first passage 8 and the second passage 9 are partitioned by partitioning means in the form of a partition plate 10 disposed between the first fan 4 and the second fan 5. The partition plate 10 may be integrally formed with a scroll casing 10a made of resin, for accommodating both fans 6 and 7.

In this embodiment, to secure both of an improvement of the heating capacity and a defrosting performance of the windshield in the double laminar mode, an amount of the inside air blown by the first fan 6 in the double laminar mode is set to be larger than an amount of the outside air blown by the second fan 7 in the double laminar mode.

That is, considering an air flow resistance (pressure loss) in the first passage 8 and an air flow resistance (pressure loss) in the double laminar mode, a blowing capacity of the first fan 6 and a blowing capacity of the second fan 7 are set in such a manner that the amount of the inside air blown by the first fan 6 is set to be larger than the amount of the outside air blown by the second fan 7.

More specifically, a cross sectional area of the first passage 8 is set to be larger than that of the second passage 7 to reduce the air flow resistance (pressure loss) in the second passage 9 as compared with the first passage 8, a blowing capacity of the second fan 7 in a state of a single fan unit is set to be larger than that of the first fan 6, or a ratio of the amount of the inside air is set to be larger than that of the outside air in the double laminar mode by a combination of adjustments of the air flow resistance and the blowing capacity.

As a result of experiments and examinations by the inventors, it is preferable that a ratio of the amount of the inside air and the outside air is specifically set to approximately 4.5:5.5 to secure both of the heating performance and the defrosting performance of the windshield.

Next, the air conditioning unit 100 is of a type in which both of an evaporator (cooling heat exchanger) 12 and a heater core (heating heat exchanger) 13 are integrally accommodated in a single air conditioning case 11. Hereinafter, a structure of the air conditioning unit 100 will be described specifically with reference to FIG. 2.

The air conditioning case 11 is made of resin which has an elasticity to some degree and is superior in a strength,

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such as polypropylene, and is composed of right and left division cases each having a division surface in a vertical direction (up-and-down direction of the vehicle) in FIG. 2. The right and left division cases are integrally connected by fastening means such as a metal spring clip and a screw, after the heat exchangers 12 and 13, and components such as a door (described later) are accommodated therein, to construct the air conditioning case 11.

The air conditioning unit 100 is disposed at a lower side of the dashboard in the passenger compartment and substantially at a center portion in the right-and-left direction of the vehicle. In a portion of the air conditioning case 11, at the most front side of the vehicle, there is formed an air inlet 14 into which conditioned air blown by the blower unit 1 flows. The air inlet 14 is opened at a side surface of the passenger's seat to be connected to an air outlet portion of the blower unit 1 disposed in a portion at a front side of the passenger seat.

In the air conditioning case 11, there is disposed the evaporator 12 at a portion immediately after the air inlet 14 to cross whole areas of the first and second air passages 8 and 9. As being known well, the evaporator 12 is for cooling the conditioned air while absorbing an evaporation latent heat of a refrigerant of a refrigeration cycle from the conditioned air. As shown in FIG. 2, the evaporator 12 is thin in the front-and-rear direction of the vehicle and is disposed in the air conditioning case 11 in such a manner that a longitudinal direction thereof extends in the up-and-down direction of the vehicle.

An air passage extending from the air inlet 14 to the evaporator 12 is partitioned by partitioning means in the form of a partition plate 15 into the first air passage 8 at a lower side of the vehicle and the second air passage 9 at an upper side of the vehicle. The partition plate 15 is integrally formed with the air conditioning case 11 by using resin and is a stationary partition member extending in a horizontal direction.

At an air downstream side (vehicle rear side) of the evaporator 12, there is adjacently disposed a heater core 13 to form a predetermined distance therebetween. The heater core 13 is for re-heating cool air having passed through the evaporator 12. In the heater core 13, high-temperature cooling water (hot water) for cooling an engine of the vehicle flows, and the heater core 13 heats the air by using the cooling water as heat source. Similar to the evaporator 12, the heater core 13 is thin in the front-and-rear direction of the vehicle and is disposed in the air conditioning case 11 in such a manner that a longitudinal direction thereof extends in the up-and-down direction of the vehicle. However, the heater core 13 is inclined from a vertical direction to the front side of the vehicle with a slight angle.

In the air conditioning case 11, there is formed a cool air bypass passage 16 at an upper portion of the heater core 13, through which the air (cool air) flows while bypassing the heater core 13.

In the air conditioning case 11, between the heater core 13 and the evaporator 12, there is provided temperature control means in the form of a plate-like main air mixing door 17 and a plate-like auxiliary air mixing door 18 for adjusting an amount ratio of warm air to be heated in the heater core 13 and the cool air bypassing the heater core 13 (i.e., cool air are flowing in the bypass passage 16). Both air mixing doors 17 and 18 are integrally connected to rotary shafts 17a and 18a, respectively, and are rotatable with the rotary shafts 17a and 18a in the up-and-down direction of the vehicle.

The rotary shafts 17a and 18a are rotatably supported in the air conditioning case 11, and each end portion of the

rotary shafts **17a** and **18a** protrudes outside and is connected to a link mechanism (not shown). Both air mixing doors **17** and **18** are interlockingly operated by the link mechanism and an actuator such as a servomotor in accordance with a blown-air temperature control signal of the air conditioning apparatus.

The rotary shaft **17a** of the main air mixing door **17** is disposed at an upper side of the rotary shaft **18a** of the auxiliary air mixing door **18** to form a predetermined distance therebetween, and both air mixing doors **17** and **18** are rotated to any rotational positions not to interfere with each other. In the maximum cooling operation, the air mixing doors **17** and **18** are rotated to positions where both are overlapped with each other, shown by a two-dot chain line in FIG. 2, in such a manner that both air mixing doors **17** and **18** are attached by pressure to a protrusion rib at a side of the air conditioning case **11** to fully close an air inflow passage into the heater core **13**.

In the maximum heating operation, both air mixing doors **17** and **18** are rotated to positions shown by solid lines, and the main air mixing door **17** closes an inlet hole **16a** of the cool air bypass passage **16**. Simultaneously, a top end portion of the auxiliary air mixing door **18** is located at a position immediately after the evaporator **12** and in the vicinity of an extension line A of the partition plate **15** so that the auxiliary air mixing door **18** functions as partitioning means in the form of a movable partition member for partitioning an air passage between the evaporator **12** and the heater core **13** into the first air passage **8** and the second air passage.

Especially, in this embodiment, the top end portion of the auxiliary air mixing door **18** is shifted from the extension line A of the partition plate **15** toward the second air passage **9** by a predetermined distance. More specifically, as shown in FIG. 15A, the topmost end portion of the auxiliary air mixing door **18** is shifted from the extension line A of the partition plate **15** toward the second air passage **9** by a predetermined distance L1 (e.g., 2 mm). Therefore, the whole of the top end portion (the whole of the thickness of the top end portion including an elastic sealing material portion affixed on a door base plate) of the auxiliary air mixing door **18** can enter an area at a side of the second air passage **9**.

Especially, in this embodiment, the top end portion of the auxiliary air mixing door **18** is set to be shifted from the extension line A of the partition plate **15** toward the second air passage **8** by a predetermined distance.

Here, rotating areas of both air mixing doors **17** and **18** are partially overlapped with each other, as shown by circular areas **17b** and **18b** of FIGS. 1 and 2. In this way, the rotating areas are commonly used, and the space for installing both air mixing doors **17** and **18** can be reduced.

Further, in the layout where the rotating areas **17b** and **18b** of both air mixing doors **17** and **18** are partially overlapped with each other, to improve the function for partitioning the passage by the auxiliary air mixing door **18**, the following construction is employed. That is, of the inner wall surface (wall surfaces of the air passages **8** and **9**) of the air conditioning case **11**, at a portion where the rotating areas **17b** and **18b** are overlapped, there is provided a level difference **11b** which contacts only the auxiliary air mixing door **18** without contacting the main air mixing door **17**.

As shown in FIG. 16, the level difference **11b** is integrally formed with the air conditioning case **11** made of resin. The level difference **11b** is formed by enlarging the wall surface of the rotating area **18b** of the auxiliary air mixing door **18**

outwardly of the case, from the wall surface of the rotating area **17b** of the main air mixing door **17**.

Because the level difference **11b** is formed on each wall surface of two cases opposed to each other in a vertical direction of the sheet (the vehicle left-and-right direction), of the auxiliary air mixing door **18**, both end portions in the direction of the rotary shaft **18a** contact the level difference **11b**.

Further, the level difference **11b** is formed entirely from an end portion at the side of the rotary shaft **18a**, of the auxiliary air mixing door **18**, to an end portion at the opposite side of the rotary shaft **18a**, so as to contact the auxiliary air mixing door **18**.

The level difference **11b** forms a sealing surface when the auxiliary air mixing door **18** is stopped to a position of the maximum heating operation.

In FIG. 16, a sealing surface **11a**, to which the main air mixing door **17** in the maximum heating operation, is formed to protrude from the inner wall surface of the air conditioning case **11** at a peripheral portion of the inlet hole **16a** of the cool air bypass passage **16**. The sealing surface **11a** secures a sealing effect when the main air mixing door **17** is stopped to a position of the maximum heating operation.

The evaporator **12** is of a laminated type in which a plurality of flat tubes, each of which is formed by connecting two metal thin plates made of aluminum or the like to each other, are laminated to sandwich a corrugated fin between the adjacent flat tubes, and then integrally brazed. In the evaporator **12**, there is formed an air passage on the extension line A by a fin surface of the corrugated fin or a flat surface of the flat tube, so that the first air passage **8** and the second air passage are partitioned in the evaporator **12**.

In the air conditioning case **11**, there is provided, at an air downstream side (portion at the rear side of the vehicle), partitioning means in the form of a partition wall **19** integrally with the air conditioning case **11** and extending in the up-and-down direction to form a predetermined distance with the heater core **13**. By the partition wall **19**, there is formed a first warm air passage **19a** extending upwardly from a position immediately after the heater core **13**. At a downstream side (upper side) of the warm air passage joins to the cool air bypass passage **16** at an upper side of the heater core **13** to form a cool air/warm air mixing space **20** for mixing the cool air and the warm air.

At a lower end portion of the partition wall **19**, there is opened a warm air bypass inlet portion **21** opposed to a surface at the air downstream side, of the heater core **13** the warm air bypass inlet portion **21** is opened and closed by a warm air bypass door **22**. The warm air bypass door **22** is connected to a rotary shaft **23** rotatably disposed at an upper end portion of the warm air bypass inlet portion **21**, and is rotated integrally with the rotary shaft **23** between a position shown by the solid line and a position shown by the two-dot chain line in FIG. 2. In this embodiment, the warm air bypass door **22** is operated by a link mechanism and an actuator such as a servomotor, as shown in FIGS. 13 and 14, in accordance with a blown-air temperature control signal and an air outlet mode control signal of the air conditioning apparatus.

In a foot air outlet mode (described later) and a foot/defroster outlet mode (described later), when the maximum heating operation (double laminar mode) is set, the warm air bypass door **22** is operated to a position shown by the solid line of FIG. 2 (position in the vicinity of a partition line B of the heater core **13**) so that the warm air bypass door **22**

functions as partitioning means in the form of a movable partition member for partitioning the first warm air passage **19a** immediately after the heater core **13** into the first air passage **8** and the second air passage **9**. A stop position of the warm air bypass door **22** is set such that a top end portion of the door **22** is shifted from the partition line B toward the second air passage **9** by a predetermined distance, similar to the auxiliary air mixing door **18**.

More specifically, as shown in FIG. **15B**, the topmost end portion of the warm air bypass door **22** is shifted from the partition line B of the heater core **13** toward the second air passage **9** by a predetermined distance **L2** (e.g., 3 mm).

The heater core **13** is of a type in which a plurality of flat tubes, each of which is formed by connecting metal thin plates made of aluminum or the like to have a flat cross section, to sandwich a corrugated fin between the adjacent flat tubes, and then integrally brazed. In the heater core **13**, there is partitioned an air passage on the partition line B by a fin surface of the corrugated fin or a flat surface of the flat tube, so that the first air passage **8** and the second air passage are partitioned in the heater core **13**.

At an air upstream side of the heater core **13**, there is formed partitioning means in the form of a stationary partition plate **24** integrally with the air conditioning case **11**, for partitioning between the partition line B and the rotary shaft **17a** of the auxiliary air mixing door **18**.

In an upper surface portion of the air conditioning case **11**, there is opened a defroster opening portion **25** at a portion of the vehicle front side. The conditioned air from the cool air/the warm air mixing space **20**, a temperature of which has been controlled, flows into the defroster opening portion **25**, and then is blown out toward an inner surface of the windshield of the vehicle, through a defroster duct (not shown) and a defroster air outlet (not shown). An inlet hole **25a** provided in a passage leading to the defroster opening portion **25** is opened and closed by the defroster door **26**. The defroster door **26** is rotatably disposed by a rotary shaft **27**.

In the upper surface portion of the air conditioning case **11**, there is opened a face opening portion **28** at a portion of the vehicle rear side (at a side of the passenger). The conditioned air from the cool air/the warm air mixing space **20**, a temperature of which has been controlled, flows into the face opening portion **25** through a communication path **36**, and then is blown out from a face air outlet at an upper portion of the dashboard, through a face duct (not shown).

At an upper portion of a side surface at the vehicle rear side, of the air conditioning case **11**, there is opened a front-seat foot opening portion **29**. The conditioned air from the cool air/the warm air mixing space **20**, a temperature of which has been controlled, flows into the front-seat foot opening portion **29** through a communication path **36**, and in the maximum heating operation, the warm air from the warm air bypass inlet portion **21** through an opening thereof flows into the front-seat foot opening portion **29** through a second warm air passage **30**. The warm air is blown out from the front-seat foot air outlet toward feet of the passenger at the front seat, through a front-seat foot duct (not shown).

Between an inlet hole **29a** of the front-seat foot opening portion **29** and the face opening portion, there is disposed a foot/face switching door **31** rotatably supported by a rotary shaft **32**. The inlet hole **29a** of the front-seat foot opening portion **29** and the face opening portion **28** are selectively opened and closed by the door **31**.

At a lower portion of a side surface at the vehicle rear side (at a side of the passenger), of the air conditioning case **11**,

there is opened a rear-seat foot opening portion **33** opposed to a position immediately after the warm air bypass inlet portion **21**. The warm air from the warm air bypass inlet portion **21** and the second warm air passage **30** flows into the rear-seat foot opening portion **33**, and is blown out from a rear-seat foot air outlet (not shown) toward feet of the passenger at the rear seat, through a rear-seat foot duct (not shown).

Further, at a lower end portion of the warm air bypass inlet portion **21**, there is provided a warm air guide plate for guiding the warm air toward the second warm air passage **30**.

In this embodiment, in the double laminar mode during the foot air outlet mode, at an air downstream side of the heater core **13**, the warm air bypass door **22** is operated to a position shown by the solid line to partition the first and the second air passages **8** and **9** and to open the communication path **36**. In this way, the first and the second air passages **8** and **9** are communicated with each other at a position in the vicinity of the front-seat foot opening portion **29**.

The defroster door **26** and the foot/face switching door **31** are connected to a link mechanism (not shown) and are interlockingly operated by an actuator such as a servomotor in accordance with an air outlet mode control signal of the air conditioning apparatus.

Each of the above-described doors **4**, **5**, **17**, **18**, **22**, **26**, and **31** has the same structure in a state of the single unit, and has a door base plate made of resin or metal, which is integrally connected to the respective rotary shafts **4a**, **5a**, **17a**, **18a**, **23**, **27**, and **32** and is covered with elastic sealing materials at both sides.

In this embodiment, the warm air bypass door **22** and the foot/face switching door **31** construct foot-side door means.

Next, an operation of the first embodiment having the above-described construction will be described.

The air conditioning apparatus is provided with an electronic control unit (not shown) to which an operation signal from each of various operation members and a sensor signal from each of various air conditioning sensors are input, and each position of the doors **4**, **5**, **17**, **18**, **22**, **26**, and **31** is controlled by output signals of the control unit.

FIGS. **1** to **3** show a state in which the double laminar mode is set when the maximum heating state is set in the foot air outlet mode. In this state, the first inside air introduction port **2** is communicated with the suction port **6a** of the first (inside air) fan **6** and further the outside air introduction port **3** is communicated with the suction port **7a** of the second (outside air) fan **7** in the blower unit **1**. Therefore, in this state, the first fan **6** blows the inside air from the inside air introduction port **2** into the first (inside air) passage **8**, and the second fan **7** blows the outside air from the outside introduction port **3** into the second (outside air) passage **9**.

Further, in the air conditioning unit **100**, both air mixing doors **17** and **18** are rotated to the positions shown by the solid lines, so that the main air mixing door **17** fully closes the inlet port **16a** of the cool air bypass passage **16**, and simultaneously the top end portion of the auxiliary air mixing door **18** is located at a position immediately after the evaporator **12** and is shifted from the extension line A of the partition plate **15** toward the second air passage **9** by the predetermined distance. In this way, the auxiliary air mixing door **18** functions as the movable partition member for partitioning the air passage between the evaporator **12** and the heater core **13** into the first air passage **8** and the second air passage **9**.

The warm air bypass door **22** is operated to the position shown by the solid line, so that the warm air bypass door **22**

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functions as the movable partition member for partitioning the first warm air passage 19a immediately after the first warm air passage 19a into the first air passage 8 and the second air passage 9 and opens the warm air bypass inlet portion 21.

The defroster door 26 is operated to an intermediate position between the communication path 36 and the inlet hole 25a of the defroster opening portion 25 to open both of the communication path 36 and the defroster opening portion 25. The foot/face switching door 31 closes the face opening portion 28 and opens the front-seat foot opening portion 29.

Therefore, by operating fans 6 and 7, the inside air from the inside air introduction port 2 and the outside air from the outside air introduction port 3 flow separately into the first air passage 8 and the second air passage 9, respectively, while being partitioned by the partition members 10, 15, 18, and 22. All of the inside air and the outside air passes through the heater core 13 and are heated maximally.

After being heated in the heater core 13, the inside air passes through the warm air bypass inlet portion 23 and the second warm air passage 30 and leads to the front-seat and the rear-seat opening portions 29 and 33. On the other hand, after being heated in the heater core 13, the outside air passes through the first warm air passage 19a at an upper side of the warm air bypass door 22, leads to the cool air/warm air mixing space 20, and further is divided into two flows therefrom. One of the outside air flows into the defroster opening portion 25, and the other outside air flows into the front-seat foot opening portion 29 through the communication path 36.

As a result, the warm air produced by heating the outside air having a low-humidity flows into the defroster opening portion 25, and the warm air having a low-humidity is blown out toward the inner surface of the windshield, so that the defrosting performance of the windshield can be secured sufficiently. Further, the high-temperature warm air produced by heating the inside air mainly is blown out into the front-seat and the rear-seat foot opening portions 29 and 33, so that the heating effect can be improved. In FIG. 2, an arrow C shows a flow of the inside air, whereas an arrow D shows a flow of the outside air.

At this time, a ratio of an amount of air blown into the defroster opening portion 25 and an amount of air blown into the foot opening portions 29 and 33 is controlled by operating the defroster door 26 into the intermediate position of the defroster door 26 to introduce the outside air in the second air passage 9 into the front-seat opening portion 29, so that the amount of the air blown into the foot opening portions 29 and 33 is set approximately 80% and the amount of the air blown into the defroster opening portions 25 is set approximately 20%.

Further, a noticeable point in the double laminar mode is that, although there is formed the communication path 36 communicating the first air passage 8 and the second air passage 9 at a downstream side of the heater core 13, the inside air is effectively prevented from flowing into the defroster opening portion 25.

That is, as described above, the ratio of the amount of the outside air is larger than that of the inside air (more specifically, there is set the ratio of approximately 4.5:5.5), further, the outside air is guided by the defroster door 26 to direct a dynamic pressure of the outside air having reached the position of the cool air/warm air mixing space 20 toward the communication path 36, and the air flow resistance in the front-seat and the rear-seat opening portions 29 and 33 is

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sufficiently smaller than that in the defroster opening portion 25. Therefore, the dynamic pressure of the inside air having reached the portion of the front-seat foot opening portion 29 is released into the front-seat foot opening portion 29 and lowers, and it is prevented that the inside air flows backward in the communication path 36 and is mixed into the outside air in the defroster opening portion 25.

In addition, because the auxiliary air mixing door 18 contacts the level difference 11b when the auxiliary air mixing door 18 is stopped at the position of the maximum heating operation to construct the sealing surface, the function for partitioning the first air passage 8 and the second air passage 9 can be improved. Therefore, the separating performance between the outside air and the inside air can be further improved.

In this way, it is possible to suppress the inside air from flowing into the area of the outside air laminar through the gap at the top end portion of the warm air bypass door 22.

Further, in the double laminar mode, the top end portion of the auxiliary air mixing door 18 is shifted from the extension line A of the partition plate 15 toward an area of the second air passage 9 (the outside air laminar area) by the predetermined distance L1, and the top end portion of the warm air bypass door 22 is shifted from the partition line B inside the heater core 13 toward an area of the second air passage 9 (the outside air laminar area) by the predetermined distance L2.

A dynamic pressure of the outside air in the second air passage 9 is applied to gaps at the top end portions of the auxiliary air mixing door 18 and the warm bypass door 22, and the outside air may flow into the gap. However, according to the above construction, it is possible to satisfactorily suppress the inside air in the first air passage 8 from leaking into the outside air laminar area through the gap.

Next, in the foot air outlet mode, when both mixing doors 17 and 18 are operated from the maximum heating state to an intermediate opening position to control a temperature of the blown-air, the air conditioning unit 100 becomes in a state of a normal mode as shown in FIG. 4. In this state of the normal mode, both air mixing doors 17 and 18 are operated to the intermediate opening position, and the main air mixing door 17 opens the cool air bypass passage 16. Therefore, the cool air passes through the cool air bypass passage 16 and flows directly into the cool air/warm air mixing space 20 while bypassing the heater core 13.

Interlocked with the operations of both air mixing doors 17 and 18, the warm air bypass door 22 is operated to the position shown by the solid line in FIG. 4, in which the warm air bypass inlet portion 21 is closed and the partitioning function for the first warm air passage 19a immediately after the heater core 13 is eliminated.

Therefore, all of the warm air heated while passing through the heater core 13 rises in the first warm air passage 19a and then is mixed with cool air from the cool air bypass passage 16 in the space 20, so that warm air having a desired temperature can be obtained. Most of the warm air passes through the communication path 36, leads toward the front-seat and the rear-seat foot opening portions 29 and 33, and is blown out toward feet of the passenger.

The remaining warm air in the space 20 leads toward the defroster opening portion 25 and is blown out toward the inner surface of the windshield.

In the foot air outlet mode having a temperature control area shown in FIG. 4, the maximum heating capacity is not necessary. Therefore, in the inside air/outside air introduction mode, generally, there is set a whole outside air mode

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in which both of the first and the second inside air introduction ports **2** and **2a** are closed and only the outside air introduction port **3** is opened. However, temperature control means is provided by a manual operation of the passenger, there may be set a whole inside air mode in which the outside air introduction port **3** is closed and both of the first and the second inside air introduction ports **2** and **2a** are opened, or an inside air/outside air mixture mode in which the inside air and the outside air are simultaneously introduced, as described above.

In the foot air outlet mode having this temperature control area, because the amount of the air blown toward the front-seat and the rear-seat foot opening portions **29** and **33** may lower by closing the warm air bypass inlet portion **21**, the defroster door **26** is displaced from the position in FIG. **3** to the position in FIG. **4**, in which an opening area of the communication path **36** is increased to prevent a decrease of the amount of the blown-air.

Next, FIG. **5** shows a state in which the double laminar mode is set when the maximum heating state is set in the foot/defroster air outlet mode in which an amount of air blown from the front-seat and the rear-seat foot opening portions **29** and **33** is approximately equal to that from the defroster opening portion **25**. In the double laminar mode during the foot/defroster air outlet mode, as being understood from a comparison with FIG. **3**, the defroster door **26** is operated to a position to close the communication path **36**.

In this way, because there is no flow of air flowing from the communication path **36** toward the front-seat foot opening portion **29**, it is possible to set the amount of the air blown from the front-seat and the rear-seat opening portions **29** and **33** to be approximately equal to that from the defroster opening portion **25**. The other points are similar to those in the double laminar mode in the foot air outlet mode.

Because air flow resistance of each portion in the air conditioning unit **100** varies for every product, the defroster door **26** may be operated to a position where the communication path **36** is opened slightly. In this way, in the double laminar mode, not only in the foot air outlet mode but also the foot defroster air outlet mode, the outside air from the second air passage **9** flows into the front-seat foot opening portion **29** through the communication passage **9**.

Next, FIG. **6** shows a state of a normal mode when both mixing doors **17** and **18** are operated from the maximum heating state to an intermediate opening position in the foot/defroster air outlet mode, to control a temperature of the blown-air. In this state of the normal mode, interlocked with the operations of both air mixing doors **17** and **18**, the warm air bypass door **22** is operated to the position shown by solid line in FIG. **6**, to close the warm air bypass inlet portion **21**. To secure an air flow passage toward the front-seat and the rear-seat foot opening portions **29** and **33**, the defroster door **26** is operated to an intermediate position shown in FIG. **6** to maintain an air amount ratio in which an amount of the air blown toward the foot opening portions **29** and **33** is approximately equal to that toward the defroster opening portion **25**.

FIG. **7** shows a state of a face air outlet mode, in which the doors **22**, **26** and **31** is operated to positions shown by the solid lines, respectively, to open only an air passage into the face opening portion **28**. Both air mixing doors **17** and **18** are operated in the maximum cooling state to fully close an air inflow passage into the heater core **13**. Therefore, all of cool air cooled in the evaporator **12** passes through the bypass passage **16** and is blown out toward the face opening portion **28**.

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By operating both air mixing doors **17** and **18** from the maximum cooling state toward the maximum heating operation, a temperature of the blown-air in the face air outlet mode can be adjusted freely.

FIG. **8** shows a state of a bi-level air outlet mode, in which the foot/face switching door **32** is operated from the position of the face air outlet mode to an intermediate position where an air passage toward the face opening portion **28** and an air passage toward the foot opening portions **29** and **33** are opened simultaneously. In this way, the cool air from the cool air bypass passage **16** is mainly blown toward the face opening portion **28**, and the warm air from the first warm air passage **19a** is mainly blown into the foot opening portions **29** and **33**. Therefore, the temperature of the air blown toward the face opening portion **28** is lower than that toward the foot opening portions **29** and **33**, so that a temperature distribution of the blown-air, for cooling a head portion and heating a foot portion can be obtained.

FIG. **9** shows a state of a defroster air outlet mode, in which the doors **22**, **26** and **31** is operated to positions shown by the solid lines, respectively, to open only an air passage into the defroster opening portion **25**. Both air mixing doors **17** and **18** are operated in the maximum heating state to fully close the cool air bypass passage **16**; however, by rotating both air mixing doors **17** and **18** from the maximum heating state toward the maximum cooling operation, a temperature of the blown-air in the defroster air outlet mode can be adjusted freely. Further, in the defroster air outlet mode, the warm air bypass door **22** is operated to a position where the warm air bypass inlet portion **21** is closed, so that the warm air is prevented from flowing toward the second warm air passage **30**.

Next, FIGS. **10** and **11** specifically show a link mechanism **40** for interlockingly operating two plates of the air mixing doors **17** and **18** and a servomotor (actuator) **50** for actuating the doors. Each of the rotary shaft **17a** and **18a** of the air mixing doors **17** and **18** protrudes outside the air conditioning case **11**, and the protruding end portions are integrally connected to one end portions of driven link levers **41** and **42**, respectively. In the driven link levers **41** and **42**, there are formed cam grooves **41a** and **42a**, respectively.

On the other hand, the servomotor **50** is fixed onto a portion at a vehicle front side from the rotary shafts **17a** and **18a**, of the outer wall surface of the air conditioning case **11**, by fastening means such as a screw. In a motor case **51** of the servomotor **50**, there are accommodated a motor mechanism portion (not shown) and an output shaft **52**. The axial direction of the output shaft **52** extends in a vertical direction of the sheet (the vehicle left-and-right direction) of FIGS. **10** and **11**.

A rotation center portion of an actuating link lever **43** integrally formed in a forked shape is integrally connected to the output shaft **52** so that the output shaft **52** and the actuating link lever **43** rotate integrally. Pin portions **43c** and **43d** are integrally provided on top ends of a first lever portion **43a** and a second lever portion **43b** of the actuating link lever **43**, respectively. The pin portions **43c** and **43d** are slidably fitted in the cam grooves **41a** and **42a** of the driven link levers **41** and **42**, respectively.

FIG. **10** shows rotational positions of the air mixing doors **17** and **18** in the maximum heating state (including the double laminar mode), and FIG. **11** shows rotational positions of the air mixing doors **17** and **18** in the maximum cooling state.

Since the link mechanism **40** and the servomotor **50** for actuating the doors are constructed as described above, when

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the output shaft **52** of the servomotor **50** rotates, the rotary shafts **17a** and **18a** of both air mixing doors **17** and **18** rotate through the actuating link lever **43** and the driven link levers **41** and **42**, respectively, and both air mixing doors **17** and **18** interlockingly rotate.

FIG. **12** shows characteristics of door operation, in which the horizontal axis indicates a rotational angle, i.e., an actuating angle, of the servomotor **50** and the vertical axis indicates rotational angles, i.e., driven angles, of the air mixing doors **17** and **18**. According to the characteristics of door operation of FIG. **12**, first, the main air mixing door **17** rotates from the maximum cooling position (the actuating angle is equal to 0°), and slightly thereafter the auxiliary air mixing door **18** starts to rotate. At a side of the maximum heating operation, first, the auxiliary air mixing door **18** finishes rotating, and thereafter the main air mixing door **17** finishes rotating.

According to the characteristics of door operation of FIG. **12**, in the maximum heating operation, the auxiliary air mixing door **18** is rotated to the predetermined partition position (the position where the auxiliary air mixing door **18** is shifted from the partition line A of the evaporator **12** toward the outside air by the predetermined distance), as shown in FIG. **15A**.

FIGS. **13** and **14** specifically show a link mechanism **60** for rotating the warm air bypass door **22** and a servomotor (actuator) for actuating the door. The rotary shaft **23** of the warm air bypass door **22** protrudes outside the air conditioning case **11**, and the protruding end portion is integrally connected to one end portion of a driven link lever **61**. In the driven link lever **61**, there is formed cam groove **61a**.

On the other hand, the servomotor **70** is fixed onto a portion at a vehicle front side from the rotary shaft **23**, of the outer wall surface of the air conditioning case **11**, by fastening means such as a screw. In a motor case **71** of the servomotor **50**, there are accommodated a motor mechanism portion (not shown) and an output shaft **72**. The axial direction of the output shaft **72** extends in a vertical direction of the sheet (the vehicle left-and-right direction) of FIGS. **13** and **14**.

A rotation center portion of an actuating link lever **73** is integrally connected to the output shaft **72** so that the output shaft **72** and the actuating link lever **73** rotate integrally. A pin portion **73a** is integrally provided on a top end of the actuating link lever **73**. The pin portions **73a** is slidably fitted in the cam groove **61a** of the driven link lever **61**.

FIG. **13** shows a state where the warm air bypass door **22** closes the warm air bypass inlet portion **21**, and FIG. **14** shows a state where the warm air bypass door **22** partitions the first warm air passage **19a** at a position immediately after the heater core **13** into the first air passage **8** and the second air passage **9**.

Since the link mechanism **60** and the servomotor **70** for actuating the door are constructed as described above, when the output shaft **72** of the servomotor **70** rotates, the rotary shaft **23** of the warm air bypass door **22** rotates through the actuating link lever **73** and the driven link lever **61**, and the warm air bypass door **22** rotates between the position shown in FIG. **13** and the position shown in FIG. **14**.

In the first embodiment, the warm air bypass door **22** is actuated by the actuating mechanism provided independently of the actuating mechanism (link mechanism and the actuating servomotor) of the air mixing doors **17** and **18**; however, the warm air bypass door **22** may be actuated commonly by the actuating mechanism of the air mixing doors **17** and **18**.

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For example, the rotary shaft **23** of the warm air bypass door **22** is connected to an output shaft of the actuating servomotor of the air mixing doors **17** and **18** through a link mechanism. Further, in the other air outlet modes than the foot air outlet mode and the foot/defroster air outlet mode, e.g., in the defroster mode, even if both air mixing doors **17** and **18** are rotated to the maximum heating state, the warm air bypass door **22** is maintained at the closing position (shown by the two-dot chain line in FIG. **2**) where the warm air bypass inlet portion **21** is closed, and only in the maximum heating operation in the foot air outlet mode and the foot/defroster air outlet mode, the warm air bypass door **22** is switched from the closing position to a partitioning position for partitioning between the first air passage **8** and the second air passage **9**.

For this operation, in the maximum heating operation in the foot air outlet mode and in the foot/defroster air outlet mode, a rotational amount of the actuating servomotor of the air mixing door is increased as compared with the maximum heating operation in the other air outlet modes, and by the increase of the rotational amount of the servomotor, the warm air bypass door **22** may be switched from the closing position to the partitioning position between the first air passage **8** and the second air passage **9**, while both air mixing doors **17** and **18** are maintained at the maximum heating state.

The present invention may be employed in an air conditioning apparatus in which the rear-seat opening portion **33** is eliminated from the first embodiment.

Further, in the first embodiment, the single warm air bypass door **22** has not only a function as the movable partition member in the double laminar mode but also a function for opening and closing the warm bypass inlet portion **21**; however, according to a modification of the second warm air passage **30** in the air conditioning unit, the warm air bypass door **22** may be divided into a door having a function as the movable partition member and a door for opening and closing the warm bypass inlet portion **21**.

A second embodiment of the present invention will be described with reference to FIGS. **17** to **24**.

The inventors of the present invention have specifically examined and studied the switching order of the inside air/outside air switching doors **4** and **5** in the blower unit **1** and the warm air bypass door **22** in the air conditioning unit **100** in the first embodiment. As a result, it turns out that, by selectively operating the inside air/outside air switching doors **4** and **5** and the warm air bypass door **22** based on the predetermined switching order, the windshield can be effectively prevented from being frosted or clouded in a switching process between the whole outside air mode and the inside air/outside air double laminar mode. In the second embodiment, the switching order is defined.

FIGS. **17** and **18** show the blower unit **1** and the air conditioning unit **100**, respectively. The constructions of the blower unit **1** and the air conditioning unit **100** are basically identical or equivalent to those in the first embodiment, and shown by the same reference numerals.

In the blower unit **1** shown in FIG. **17**, the outside air introduction port **3** and the second inside air introduction port **2a** are adjacently disposed in an upper portion of the unit, and the outside air introduction port **3** and the second inside air introduction port **2a** are selectively opened and closed by the second inside air/outside air switching door **5**. At a downstream side of the second inside air/outside air switching door **5**, there is disposed an air filter for purifying (removing dusts in or adsorbing an malodor of) an air

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introduced from the outside air introduction port **3** and the second inside air introduction port **2a**.

Further, at a lower side of the first fan **6** at the side of the inside air, there is disposed an actuating electric motor **42**. By the actuating electric motor **42**, the first fan **6** at the side of the inside air and the second fan **7** at the side of the outside air are rotated. The suction port **6a** of the first fan **6** can be in communication with the first inside air introduction port **2** and also with a space **44** at a downstream side of the air filter through a communication path **43**. The first inside air/outside air switching door **4** selectively opens and closes the first inside air introduction port **2** and the communication path **43**.

In the air conditioning unit **100** shown in FIG. **18**, a door mechanism for switching the air outlet mode is modified from the first embodiment. That is, the defroster door **26** is a butterfly-shaped door which rotates with the rotary shaft **27** as a center thereof to open and close the inlet hole **25a** of the defroster opening **25**. In this embodiment, the defroster door **26** does not contribute to open and close the communication path **36**.

The foot/face switching door **31** of the first embodiment is divided into a face door **31** and a foot door **310**. The face door **31** and the foot door **310** are butterfly-shaped doors which rotate, respectively, with the rotary shafts **32** and **311** as a center thereof to open and close the face opening portion **28** and the inlet hole **29a** of the front-seat foot opening portion **29**.

The defroster door **26**, the face door **31**, and the foot door **310**, which are door means for switching the air outlet mode, are connected to a link mechanism (not shown) and are interlockingly operated by an actuator such as a servomotor in accordance with an air outlet mode control signal of the air conditioning apparatus. The other points than the above-described features are same as in the first embodiment.

FIG. **19** shows a state of the whole outside air mode in the foot/defroster air outlet mode, and FIG. **20** shows a state of the double laminar mode in the foot/defroster air outlet mode. Also in the foot air outlet mode, it is only different therefrom that the opening degree of the defroster door **26** decreases and the opening degree of the foot door **10** increases, and the other points are same as in the foot/defroster air outlet mode.

In a process for switching from the whole outside air mode in FIG. **19** to the inside air/outside air double laminar mode in FIG. **20**, the inventors have examined that a state shown in FIG. **21** is set at first. FIG. **21** shows a state where the first inside air/outside air **4** is shifted to a position to open the inside air/outside air introduction port **4** prior to the warm air bypass door **22**, from the state of the whole outside air mode in FIG. **19**. Subsequently, the warm air bypass door **22** is shifted from this state to a position for partitioning the inside air and the outside air, as shown in FIG. **20**.

However, in the switching process, if there is set the state of FIG. **21**, the inside air and the outside air which are partitioned to the upstream side of the heater core **13** are mixed in the first warm air passage **19a** at the downstream side of the heater core **13**. Therefore, it turns out that, the air having a high-humidity, which is produced by mixing the inside air and the outside air, flows into the defroster opening portion **25** temporarily, with the result that the windshield may be frosted or clouded.

Conversely, in the switching process from the inside air/outside air double laminar mode in FIG. **19** to the whole outside air mode in FIG. **20**, when the warm bypass door **22** is shifted to the normal position (the position where the inlet

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portion **21** of the second warm air passage **30** is closed) prior to the first inside air/outside air switching door **4**, similarly, the air having a high-humidity, which is produced by mixing the inside air and the outside air, flows into the defroster opening portion **25** temporarily.

In the second embodiment, when the state of the whole outside air mode in FIG. **19** is switched to the state of the inside air/outside air double laminar mode in FIG. **20**, the warm air bypass door **22** is always shifted to the position for partitioning the inside air and the outside air before the first inside air/outside air switching door **4**, as shown in FIG. **22**. In this way, because the outside air flows into each of the defroster opening portion **25**, and the foot opening portions **29** and **33**, there is no possibility that the windshield is frosted or clouded.

Conversely, when the state of the inside air/outside air double laminar mode in FIG. **20** is switched to the state of the whole outside air mode in FIG. **19**, the first inside air/outside air switching door **4** is always shifted to the position where the first inside air introduction port **2** is closed before the warm air bypass door **22** is shifted, as shown in FIG. **22**. In this way, because the outside air flows into each of the opening portions **25**, **29** and **33**, there is no possibility that the windshield is frosted or clouded.

As described above, when the whole outside air mode and the inside air/outside air double laminar mode are switched, by defining the switching order of the first inside air/outside air switching door **4** and the warm air bypass door **22**, it is possible to certainly prevent the windshield from being frosted or clouded in the switching process.

The above-described door switching order is defined specifically by electrically controlling an operation of the servomotor (actuator) for actuating each door.

FIG. **23** is a system view showing briefly an electric control in the second embodiment. Temperature control means in the form of an electronic control apparatus **50** is constructed by a microcomputer and the like, and each of various sensor signals from a group of sensors **51** and each of various operation signals from a group of operation members **52** provided on an air conditioning operation panel (not shown) are input to the electronic control apparatus **50**.

Each of various air conditioning components including a servomotor **53** for actuating the first inside air/outside air switching door **54** and the inside air/outside air switching door **5** through a link mechanism, and a servomotor **54** for interlockingly actuating the main air mixing door **17** and the auxiliary air mixing door **18** are controlled by the electronic control apparatus **50**. The servomotor **54** actuates the warm air bypass door **22** in an area at a side of the maximum heating operation, which exceeds a range of a predetermined rotational angle for actuating both of the air mixing doors **17** and **18**. Such a door actuating mechanism has been described as a modification of the door actuating mechanism in the first embodiment. A servomotor **55** is for interlockingly actuating the defroster door **26**, the face door **31**, and the foot door **310**.

The air conditioning electronic control apparatus **50** performs calculation processes in accordance with each of input signals based on a preset program stored in the ROM to control an operation of each of various components (the above-described doors and the like), and only the control portion relative to the second embodiment will be described. First calculating means **56** is for calculating a target air temperature TAO of the air blown into the passenger compartment, which is necessary to maintain the passenger compartment at a set temperature. The first calculating

means 56 calculates the target air temperature TAO based on the following formula (1) pre-stored in the ROM.

$$TAO = K_{set} \times T_{set} - K_r \times T_r - K_{am} \times T_{am} - K_s \times T_s + C \quad (1)$$

wherein, T_{set} is a set temperature set by an operation member of the air conditioning operation panel, for setting a temperature, T_r is a temperature of the inside air, detected by an inside air temperature sensor, T_{am} is a temperature of the outside air, detected by an outside air temperature sensor, and T_s is an amount of sunlight, detected by a sunlight sensor. Further, K_{set} , K_r , K_{am} , and K_s are gains, and C is a constant for correction.

Control means 57 is for controlling the electric motor 42 for the blower fan. The control means 57 determines a fan motor voltage in corresponding with TAO, based on a map pre-stored in the ROM, and applies the voltage to the motor 42. Second calculating means 58 is for calculating a temperature control calculation value SW to determine each target opening degree of the air mixing doors 17 and 18. The second calculating means 58 calculates the temperature control calculation value SW based on the following formula (2) pre-stored in the ROM.

$$SW = [(TAO - T_e) / (T_w - T_e)] \times 100(\%) \quad (2)$$

wherein, T_w is a temperature of the cooling water flowing into the heater core 13, detected by a water temperature sensor, and T_e is a temperature of air blown out from the evaporator 12, detected by an evaporator blown-air temperature sensor.

Control means 59 is for controlling the servomotor 53 for actuating the inside air/outside air switching doors 4 and 5. The control means 59 controls an electric current supplied to the servomotor 53 in correspondence with the temperature control calculation value SW, based on a map pre-stored in the ROM, to determine a rotational angle of the motor.

Control means 60 is for controlling the servomotor 54 for actuating the air mixing doors 17 and 18 and the warm air bypass door 22. The control means 60 controls an electric current supplied to the servomotor 54 in correspondence with the temperature control calculation value SW, based on a map pre-stored in the ROM, to determine a rotational angle of the motor.

Control means 61 is for controlling the servomotor 55 for actuating the air outlet mode doors 26, 31, and 310. The control means 61 controls an electric current supplied to the servomotor 55 in correspondence with the temperature control calculation value SW, based on a map pre-stored in the ROM, to determine a rotational angle of the motor.

FIG. 24 shows a content of the door control in correspondence with the temperature control calculation value SW in the second embodiment. Each of the doors is controlled as in FIG. 24 in any air outlet modes other than the defroster air outlet mode. In the face air outlet mode and the bi-level mode, because the maximum heating mode is not necessary, each of doors is controlled in an area where the temperature control calculation value SW is less than SW3 ($SW < SW3$).

The door control of FIG. 24 will be described specifically. When the temperature control calculation value SW is equal to or less than SW1, the air mixing doors 17 and 18 are operated to the maximum cooling position where the cool air bypass passage 16 is fully opened and the air passage into the heater core 13 is fully closed. Simultaneously, the warm air bypass door 22 is operated to the normal position where the inlet portion 21 is closed, and the inside air/outside air 4 and 5 are operated to the position of the whole inside air mode, where the inside air introduction ports 2 and 2a are opened and the outside introduction port 3 is closed.

When the temperature control calculation value SW exceeds the SW1 which is a sufficient small value, the first inside air/outside air switching door 4 is operated to the position of the whole outside air mode where the first inside air introduction port 2 is closed and the communication path 43 is opened, and the second inside air/outside air switching door is also operated to the position of the whole outside air mode where the second inside air introduction port 2a is closed and the outside air introduction port 3 is opened. Simultaneously, the air mixing doors 17 and 18 are operated to positions where the air passage into the heater core 13 is gradually opened.

Until the temperature control calculation value SW reaches SW2 (which is a sufficient large value as compared with the SW1) after having exceeded SW1, the warm air bypass door 22 is maintained at the normal position where the inlet portion 21 is closed, and the first and second inside air/outside air switching doors 4 and 5 are maintained at the positions of the whole outside air mode. Meanwhile, the air mixing doors 17 and 18 continuously increases the opening degree of the air passage into the heater core 13 to increase the temperature of the blown-air.

When the temperature control calculation value SW reaches SW2, the air mixing doors 17 and 18 are operated to the maximum heating positions where the cool air bypass passage 16 is fully closed and the air passage into the heater core 13 is fully opened. However, the warm air bypass door 22 is still maintained at the normal position where the inlet portion 21 is closed, and the first and second inside air/outside air switching doors 4 and 5 are also still maintained at the positions of the whole outside air mode.

When the temperature control calculation value SW reaches SW3 which is slightly larger than SW2, first, the warm air bypass door 22 is shifted from the normal position to the double laminar position. That is, the warm air bypass door 22 is operated to a position where the inlet portion 21 is opened and the first warm air passage 19a is partitioned into the first air passage 8 and the second air passage 9. At this time, because the first and second inside air/outside air switching doors 4 and 5 are still maintained at the positions of the whole outside air mode, there is no possibility that the inside air and the outside air are temporarily mixed at the downstream side of the heater core 13.

Further, when the temperature control calculation value SW exceeds SW3 and reaches SW4, the first inside air/outside air switching door 4 is shifted to the intermediate opening position to open each of the first inside air introduction port 2 and the communication path 43 by the intermediate opening degree.

When the temperature control calculation value SW exceeds SW4 and reaches SW5, the first inside air/outside air switching door 4 fully opens the first inside air introduction port 2 and fully closes the communication path 43. Because the second inside air/outside air switching door 4 is maintained in a range of $SW > SW1$ to always open the outside air introduction port 3, the inside air/outside air introduction portion of the blower unit 1 is not shifted to the position of the inside air/outside air double laminar until $SW > SW5$. That is, the timing (SW5) when the inside air/outside air introduction port of the blower unit 1 is operated to the double laminar position is behind the timing (SW3) when the warm air bypass door 22 is switched from the normal position to the double laminar position.

Conversely, when the inside air/outside air double laminar mode is switched to the whole outside air mode, at the time of SW5, first, in the inside air/outside air introduction port of the blower unit 1, the first inside air/outside air switching

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door 4 is shifted to the intermediate opening position. Subsequently, at the time of SW4, the first inside air/outside air switching door 4 fully closes the first inside air introduction port 2 and fully opens the communication path 43, so that the inside air/outside air introduction portion of the blower unit 1 is switched to the whole outside air mode.

However, at this time, because the warm air bypass passage 22 is still maintained at the double laminar position, there is no possibility that the inside and the outside air are temporarily mixed at the downstream side of the heater core 13. Subsequently, at the time of SW3, the warm air bypass door 22 is recovered from the double laminar position to the normal position.

The reason why the intermediate opening position where the first inside air/outside air switching door 4 opens each of the first inside air introduction port 2 and the communication path 43 is provided between the whole outside air mode and the double laminar mode in FIG. 24 is as follows. That is, when the inside air/outside air mode is switched directly between the whole outside air mode and the double laminar mode, the temperature, the amount, and the like, of the air blown into the passenger compartment, may vary suddenly. Therefore, by setting the intermediate opening position, it is possible to slacken the sudden variation of the temperature, the amount, and the like, due to the switching of the inside air/outside air mode.

The necessity for setting the intermediate opening position varies dependent on the specification of the ventilation system of the air conditioning apparatus; however, because the second embodiment has an object to prevent the windshield from being frosted or clouded when the inside air/outside air double laminar mode and the whole outside air mode are switched, it is not always necessary to set the intermediate opening position.

As described above, the door control in the other air outlet modes than the defroster air outlet mode is performed as shown in FIG. 24; however, in the defroster air outlet mode, independently of the temperature control calculation value SW, the whole outside air mode is always maintained to defrost the windshield.

A third embodiment of the present invention will be described with reference to FIGS. 25 and 26.

FIG. 25 is an enlarged view showing the top end portion of the auxiliary air mixing door 18 in FIG. 15A. The inventors of the present invention have variously examined and studied a correlation between a gap C generated between the top end portion of the auxiliary air mixing door 18 and the evaporator 12 at the position where the auxiliary air mixing door 18 is stopped in the double laminar mode and the defrosting performance of the windshield of the vehicle. As a result, the inventors found out that, by setting the gap C to a predetermined value, more specifically, 13 mm or less, the defrosting performance of the windshield can be secured satisfactorily.

That is, FIG. 26 shows data of an experiment result, showing a relationship between the gap C generated at the top end portion of the auxiliary air mixing door 18 and an inside air mixture percentage into the second air passage 9 at the side of the outside air. According to the examinations and studies by the inventors, it has been confirmed that, by suppressing the inside air mixture percentage into the second air passage 9 to be equal to 15% or less, a preferable field of vision of the windshield, which does not impede the driving of the vehicle (i.e., more than 90% of the area of the windshield is clear), can be secured.

The experiments have been performed on the following conditions:

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Outside air temperature: 0° C.

Outside air relative humidity: 90%

Inside air temperature: 25° C.

Number of passengers: 5 people

Total air amount in the double laminar mode: 200 m³/h

Outside air percentage: 55%

Inside air percentage: 45%

Here, the inside air mixture percentage is expressed the following formula (3)

$$\text{Inside air mixture percentage} = X/Y \quad (3)$$

wherein,

X = absolute humidity of the air blown out from the defroster air outlet - absolute humidity of the sucked outside air,

and

Y = absolute humidity of the sucked inside air - absolute humidity of the sucked outside air.

As being understood from the experiment result of FIG. 26, it has turned out that, by setting the gap C within 13 mm, the inside air mixture percentage can be suppressed to be equal to 15% or less.

By setting the inside air mixture percentage to be equal to 15% or less, more than 90% of the area of the windshield is made clear, so that the defrosting performance of the windshield can be secured satisfactorily.

In the third embodiment, the gap C at the top end portion of the auxiliary air mixing door 13 in the double laminar mode is set to be equal to 13 mm or less; however, a gap at the top end portion of the warm air bypass door 22 in the double laminar mode is specifically set to approximately 3 mm, because it is structurally easy to set the gap at the top end portion of the warm air bypass door 22 to be equal to 13 mm or less.

A fourth embodiment of the present invention will be described with reference to FIGS. 27 to 29.

In the first embodiment, as shown in FIGS. 10-14, the warm air bypass door 22 is actuated by the actuating mechanism (the link mechanism 60 and the actuating servomotor 70) provided independently of the actuating mechanism (the link mechanism 40 and the actuating servomotor 50) of both air mixing doors 17 and 18; however, the warm air bypass door 22 can be actuated by a common actuating servomotor with both air mixing doors 17 and 18.

FIGS. 27 and 28 show an actuating mechanism for actuating the warm air bypass door 22 and both air mixing doors 17 and 18 by the common servomotor 50. In FIGS. 27 and 28, parts and components similar or equivalent to those in the first embodiment are shown by the same reference numerals, and explanations thereof are omitted.

To the output shaft 52 of the common actuating servomotor 50, there is connected the link mechanism 40 which will be described below. That is, the rotation center portion of the actuating link lever 43 is integrally connected to the output shaft 52, and the lever 43 is integrally formed with three of first to three lever portions 43a, 43b, and 43c. At the top end portions of the first to third lever portions 43a, 43b, and 43c, there are integrally provided pin portions 43d, 43e, and 43f, respectively.

The pin portions 43c and 43d are slidably fitted in cam grooves 41a and 42a of the driven link levers 41 and 42 connected to the rotary shafts 17a and 18a of the air mixing doors 17 and 18, respectively.

The pin portion 43f is slidably fitted in a cam groove 44a provided at one end side of the a middle link lever 44. The

middle link lever **44** is rotatably provided onto an outer wall surface of the air conditioning case **11** with a rotary shaft **45** as a center thereof. At the other end side of the middle link lever **44**, there is provided a connection hole **44b**.

Into the connection hole **44b**, there is rotatably fitted a folded end portion **46a** provided at one end of a middle connection stick **46**. A folded end portion **46b** provided at the other end of the middle connection stick **46** is rotatably fitted in a connection hole **61b** provided in the driven link lever **61** of the warm air bypass door **22**.

FIG. **27** shows the maximum heating state (the normal maximum heating state) in the air outlet modes (e.g., the defroster air outlet mode) other than the foot air outlet mode and the foot/defroster air outlet mode. In this state, by the rotation of the output shaft **52** of the servomotor **50**, both air mixing doors **17** and **18** are rotated to the maximum heating position through the link mechanism **40**.

In contrast, the warm air bypass door **22** is maintained to close the warm air bypass inlet portion **21**.

FIG. **28** shows the maximum heating state in the double laminar mode, i.e., in the foot air outlet mode or the foot/defroster air outlet mode. In this state, the output shaft **52** of the servomotor **50** is further rotated by a predetermined amount from the normal maximum heating state in FIG. **27**. That is, even if the output shaft **52** of the servomotor **50** is rotated from the normal maximum heating state and the actuating link lever **43** is rotated, the fitting states between the pin portions **43c** and **43d** of the first and the second lever portions **43a** and **43b** of the actuating link lever **43** and the cam grooves **41a** and **42a** of the driven link levers **41** and **42** are in idling. Therefore, the driven link levers **41** and **42** do not rotate, and both air mixing doors **17** and **18** are maintained at the maximum heating position.

On the other hand, the third lever portion **43e** of the actuating link lever **43** rotates the middle link lever **44** to the position of FIG. **28**. Therefore, the driven link lever **61** is rotated to the position of FIG. **28** through the middle connection stick **46**, and the warm air bypass door **22** is rotated to the partition position between the first air passage **8** at the side of the inside air and the second air passage **9** at the side of the outside air. In this way, the double laminar mode can be set by using the single common servomotor **50**.

In FIG. **12**, the position of the actuating angle=90° corresponds to the normal maximum heating state in FIG. **27**, and the position of the actuating angle=120° corresponds to the maximum heating state (double laminar mode) in the foot air outlet mode or the foot/defroster air outlet mode in FIG. **28**.

FIG. **29** heater core **13** is fully closed and the warm air bypass door **22** closes the warm air bypass inlet portion **21**.

In the above-described embodiments, each of the doors **4**, **5**, **17**, **18**, **22**, **26**, **31**, **36**, **310**, and **312** is operated by an actuator such as a servomotor through a link mechanism; however, each door may be manually operated by a manual operation member provided on an air conditioning operation panel, such as an inside air/outside air introduction setting lever, a temperature control lever, and an air outlet mode lever.

In the above-described embodiments, the warm air bypass door **22** is actuated by an actuating mechanism independently of the air mixing doors **17** and **18**; however, the rotary shaft **23** of the warm air bypass door **22** is connected to the link mechanism of the air mixing doors **17** and **18**, so that the warm air bypass door **22** and the air mixing doors **17** and **18** can be actuated by the common actuator.

In this case, in the air outlet mode other than the foot air outlet mode and the foot/defroster air outlet mode, e.g., the

defroster air outlet mode, even if both air mixing doors **17** and **18** are in the maximum heating state, the warm air bypass door **22** is maintained at the position (two-dot chain line of FIG. **2**) where the warm air bypass inlet portion **21** is closed. That is, the rotating amount of the servomotor for actuating the air mixing door in the maximum heating operation during the foot air outlet mode and the foot/defroster air outlet mode may be changed from that during the other air outlet modes.

The maximum heating operation when the double laminar mode is set is not strictly limited to the case where the air mixing doors **17** and **18** are operated to the positions to completely prevent the cool air from bypassing; however, it may include a case where positions of the air mixing doors **17** and **18** allow a slight amount of the cool air to bypass.

In each of the first and the second embodiment, the evaporator (cooling heat exchanger) **12** is disposed in the air conditioning unit **100**; however, the present invention may be employed in an air conditioning apparatus without having the cooling heat exchanger.

Further, in each of the first and the second embodiments, as temperature adjusting means for adjusting a heating amount of conditioned air in the heater core **13** to adjust a temperature of air, the air mixing doors **17** and **18** for adjusting an air amount ratio of cool air and warm air are employed; however, instead of the air mixing doors **17** and **18**, by using a hot water valve for adjusting an amount of hot water flowing into the heater core **13** or a temperature of the hot water, the present invention can be also employed in an air conditioning apparatus in which a temperature of the air is adjusted by a function for adjusting the flow amount (or the temperature) of the hot water, of the hot water valve.

Still further, the present invention may be also employed in an air conditioning apparatus in which the rear-seat foot opening portion **33** in each of the first and the second embodiments is eliminated.

In each of the above-described embodiments, as for the air mixing doors **17** and **18** having the rotating areas **17b** and **18b** which are partially overlapped with each other, there is provided the level difference **11b** for forming a sealing surface of the auxiliary air mixing door **18**; however, the level difference **11b** is not limited to the case where the rotating areas of the air mixing doors **17** and **18** are overlapped, but may be provided in a case where rotating areas of doors such as the air outlet mode doors **26** and **31** are overlapped with each other.

Although the present invention has been fully described in connection with the preferred embodiments thereof with reference to the accompanying drawings, it is to be noted that various changes and modifications will become apparent to those skilled in the art. Such changes and modifications are to be understood as being included within the scope of the present invention as defined in the appended claims.

What is claimed is:

1. An air conditioning apparatus for a vehicle having a passenger compartment, said air conditioning apparatus comprising:

an air conditioning case for forming an air passage, said air conditioning case having a first opening portion for blowing air toward a lower portion of the passenger compartment and a second opening portion for blowing air toward an inner surface of a windshield;

a heating heat exchanger for heating air passing through said air passage, said heating heat exchanger being disposed in said air conditioning case approximately vertically;

temperature control means for adjusting a heating amount by said heating heat exchanger to control a temperature of the air;

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partitioning means for partitioning said air passage into a first air passage through which inside air flows and a second air passage through which outside air flows to set a double laminar mode when said temperature control means is controlled to maximize the heating amount during an air outlet mode where both of said first opening portion and said second opening portion are opened in such a manner that said first air passage is communicated with said first opening portion and said second air passage is communicated with said second opening portion;

- a first warm air passage for introducing the warm air having passed through said heating heat exchanger into said first and second opening portions, said first warm air passage being defined by a partition wall of said partition means to extend upwardly from an immediately downstream side of said heating heat exchanger;
- a second warm air passage for introducing the warm air having passed through said heating heat exchanger directly into said first opening portion while bypassing said first warm air passage, said second warm air passage having an inlet portion opposite to said heating heat exchanger;

- a communication path provided at an upper side of said partition wall, through which said first warm air passage and said second warm air passage communicate with each other; and

- a warm air bypass door for opening and closing said inlet portion of said second warm air passage, said warm air bypass door being operated between a first position where said inlet portion is opened and said first warm air passage is partitioned into said first air passage and said second air passage so that warm air having passed through an upper part of said heating heat exchanger is introduced toward an upper side through said first warm air passage while warm air having passed through a lower part of said heating heat exchanger is introduced into said second warm air passage through said inlet portion in said double air laminar mode, and a second position where said inlet portion is closed so that all warm air having passed through said heating heat exchanger is introduced toward the upper side through said first warm air passage, wherein:

said second warm air passage is branched from said inlet portion into a front passage extending upwardly from said inlet portion, and a rear passage extending approximately horizontally from said inlet portion; and said first opening portion includes a front-seat foot opening which is provided to communicate with said front passage of said second warm air passage, and a rear-seat foot opening which is provided to communicate with said rear passage of said second warm air passage.

- 2. An air conditioning apparatus according to claim 1, wherein,

said air conditioning case has an inside air introduction port provided at an inlet portion of said first air passage and an outside air introduction port provided at an inlet portion of said second air passage,

said air conditioning apparatus further comprising:

an inside air/outside air switching door for opening and closing said inside air introduction port and said outside air introduction port.

- 3. An air conditioning apparatus according to claim 2, wherein,

when a whole outside air mode where the outside air is introduced from both of said first air passage and said

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second air passage is switched to said double laminar mode where the inside air and the outside air are introduced into said first air passage and said second air passage, respectively, said inside air/outside air switching door is operated to a position of said double laminar mode after said warm air bypass door is operated to a position of said double laminar mode, and

when said double laminar mode is switched to said whole outside air mode, said warm air bypass door is operated to a normal position where said inlet portion is closed after said inside air/outside air switching door is operated to a position of said whole outside air mode.

- 4. An air conditioning apparatus according to claim 3, further comprising:

a first actuator for actuating said inside air/outside air switching door;

a second actuator for actuating said warm air bypass door; calculating means for a temperature control calculation value for controlling said temperature adjusting means; and

control means for controlling said first and said second actuators based on said temperature control calculation value.

- 5. An air conditioning apparatus according to claim 1, further comprising:

a cool air bypass passage through which air bypasses said heating heat exchanger; and

a cool air/warm air mixing space for mixing the cool air from said cool air bypass passage and the warm air from said first warm air passage, said cool air/warm air mixing space being provided adjacent to said communication path,

wherein said temperature adjusting means is an air mixing door for adjusting a ratio of an amount of air passing through said heating heat exchanger and an amount of air passing through said cool air bypass passage.

- 6. An air conditioning apparatus according to claim 1, wherein,

said first opening portion includes a front-seat foot opening portion and a rear-seat foot opening portion, both of which are in communication with each other through said second warm air passage,

said inlet portion of said second warm air passage is disposed to face a surface at an air downstream side, of said heating heat exchanger, and

said rear-seat foot opening portion is disposed at a position immediately after said inlet portion.

- 7. An air conditioning apparatus according to claim 1, wherein a stop position of said warm air bypass door in said double laminar mode is set in such a manner that a top end portion of said warm air bypass door is shifted into an area of said second air passage by a predetermined distance.

- 8. An air conditioning apparatus according to claim 1, further comprising:

a blower unit for blowing air while partitioning inside air and outside air;

a cool air bypass passage through which air bypasses said heating heat exchanger;

a main air mixing door disposed at an air upstream side of said heating heat exchanger, for adjusting a ratio of an amount of air passing through said heating heat exchanger and an amount of air passing through said cool air bypass passage;

an auxiliary air mixing door disposed at an air upstream side of said heating heat exchanger and operated with

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said main air mixing door, for adjusting a ratio of an amount of air passing through said heating heat exchanger and an amount of air passing through said cool air bypass passage; and

an air conditioning unit including said air conditioning case and disposing therein said heating heat exchanger, means for forming said cool air bypass passage, said main air mixing door, and said auxiliary air mixing door,

wherein said main air mixing door fully closes said cool air bypass passage and said auxiliary air mixing door partitions a space at an air upstream side of said heating heat exchanger into said first air passage and said second air passage in said air conditioning case in said double laminar mode.

9. An air conditioning apparatus according to claim 8, wherein,

rotating areas of said main air mixing door and said auxiliary air mixing door are partially overlapped with each other, and

wall surfaces of said first and said second air passages have a level difference at a portion where said rotating areas are overlapped, said level difference contacting only said auxiliary air mixing door without contacting said main air mixing door and forming a sealing surface when said auxiliary air mixing door is stopped.

10. An air conditioning apparatus according to claim 1, wherein:

said warm air bypass door is operated to said first position to partition said first air passage and said second air passage from each other only when said double laminar mode is set during the maximum heating; and

said warm air bypass door is operated to said second position so that said first air passage and said second air passage communicate with each other when the maximum heating is released.

11. An air conditioning apparatus according to claim 1, wherein:

said partitioning means includes a movable partition member disposed at an upstream air side of said heating heat exchanger;

said movable partition member is operated to partition an upstream air passage of said heating heat exchanger into said first air passage and said second air passage when said double laminar mode is set during the maximum heating; and

said movable partition member is operated to communicate said first air passage and said second air passage in the upstream air passage of said heating heat exchanger when the maximum heating is released.

12. An air conditioning apparatus according to claim 1, wherein said air conditioning case is disposed under an instrument panel of the vehicle.

13. An air conditioning apparatus according to claim 8, wherein:

said main air mixing door is rotated around a first rotation shaft;

said auxiliary air mixing door is rotated around a second rotation shaft; and

said first rotation shaft and said second rotation shaft are disposed parallel to each other to extend approximately horizontally.

14. An air conditioning apparatus for a vehicle having a passenger compartment, said air conditioning apparatus comprising:

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an air conditioning case for forming an air passage, said air conditioning case having a first opening portion for blowing air toward a lower portion of the passenger compartment and a second opening portion for blowing air toward an inner surface of a windshield;

a heating heat exchanger for heating air passing through said air passage;

a cool air bypass passage through which air bypasses said heating heat exchanger;

an air mixing door for adjusting a ratio of an amount of air passing through said heating heat exchanger and an amount of air passing through said cool air bypass passage, said air mixing door including a main air mixing door rotated around a first rotation shaft at an air upstream side of said heating heat exchanger, and an auxiliary air mixing door rotated around a second rotation shaft at an air upstream side of said heating heat exchanger and operated with said main air mixing door;

partitioning means for partitioning said air passage into a first air passage through which inside air flows and a second air passage through which outside air flows in the maximum heating operation where said air mixing door is operated to fully close said cool air bypass passage during an air outlet mode where both of said first opening portion and said second opening portion are opened, in such a manner that said first air passage is communicated with said first opening portion and said second air passage is communicated with said second opening portion;

wherein said main air mixing door fully closes said cool air bypass passage and said auxiliary air mixing door partitions a space at an air upstream side of said heating heat exchanger into said first air passage and said second air passage in a double laminar mode where said first air passage and said second air passage are partitioned; and

wherein said first and second rotation shafts are disposed approximately parallel to an extending direction of said partition means;

wherein a stop position of said auxiliary air mixing door in said double laminar mode is set in such a manner that a top end portion of said auxiliary air mixing door is shifted by a predetermined distance into an area of said second air passage.

15. An air conditioning apparatus according to claim 14, wherein,

said main air mixing door and said auxiliary air mixing door are rotated to predetermined positions, respectively, not to interfere with each other, and

in the maximum cooling operation, by a combination operation of said main air mixing door and said auxiliary air mixing door, an air passage into said heating heat exchanger is fully closed and said cool air bypass passage is fully opened.

16. An air conditioning apparatus according to claim 15, further comprising:

a link mechanism for interlockingly operating said main air mixing door and said auxiliary air mixing door; and an actuator for commonly actuating said main air mixing door and said auxiliary air mixing door through said link mechanism.

17. An air conditioning apparatus according to claim 14, further comprising:

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a cooling heat exchanger disposed in said air conditioning case adjacent an air upstream side of said heating heat exchanger, for cooling air;
wherein,
said heating heat exchanger and is disposed in said air conditioning case, and
said auxiliary air mixing door functions as a movable partition member for partitioning an air passage between said heating heat exchanger and said cooling heat exchanger into said first air passage and said second air passage.
18. An air conditioning apparatus according to claim 17, wherein:

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said heating heat exchanger is disposed at a vehicle rear side and said cooling heat exchanger is disposed at a vehicle front side from said heating heat exchanger;
said first air passage is disposed at a vehicle lower side and said second air passage is disposed at a vehicle upper side and said auxiliary air mixing door is disposed at a vehicle lower side from said main air mixing door; and
said main air mixing door and said auxiliary air mixing door are rotated in a vertical direction of the vehicle.

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