

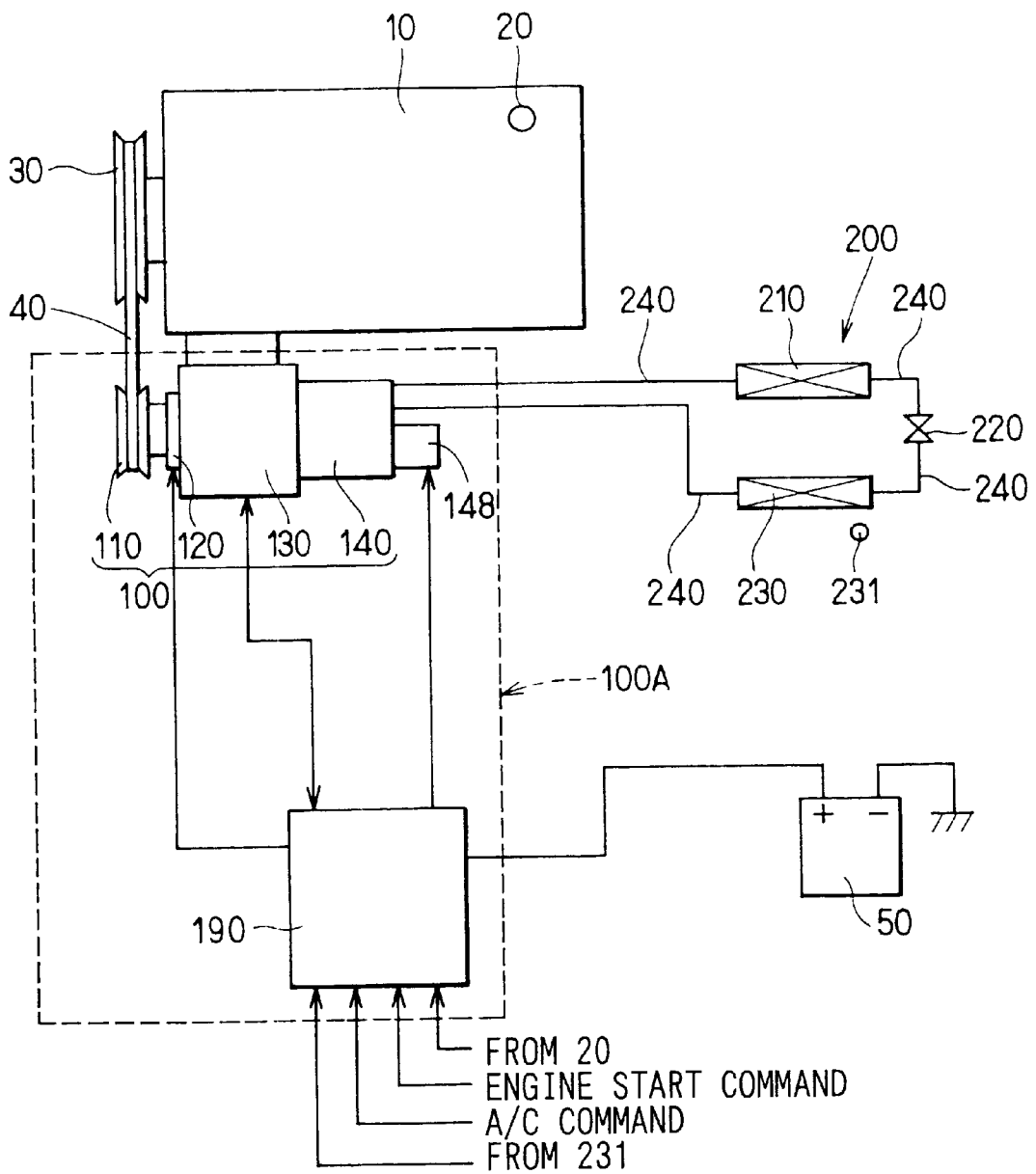


FIG. 2

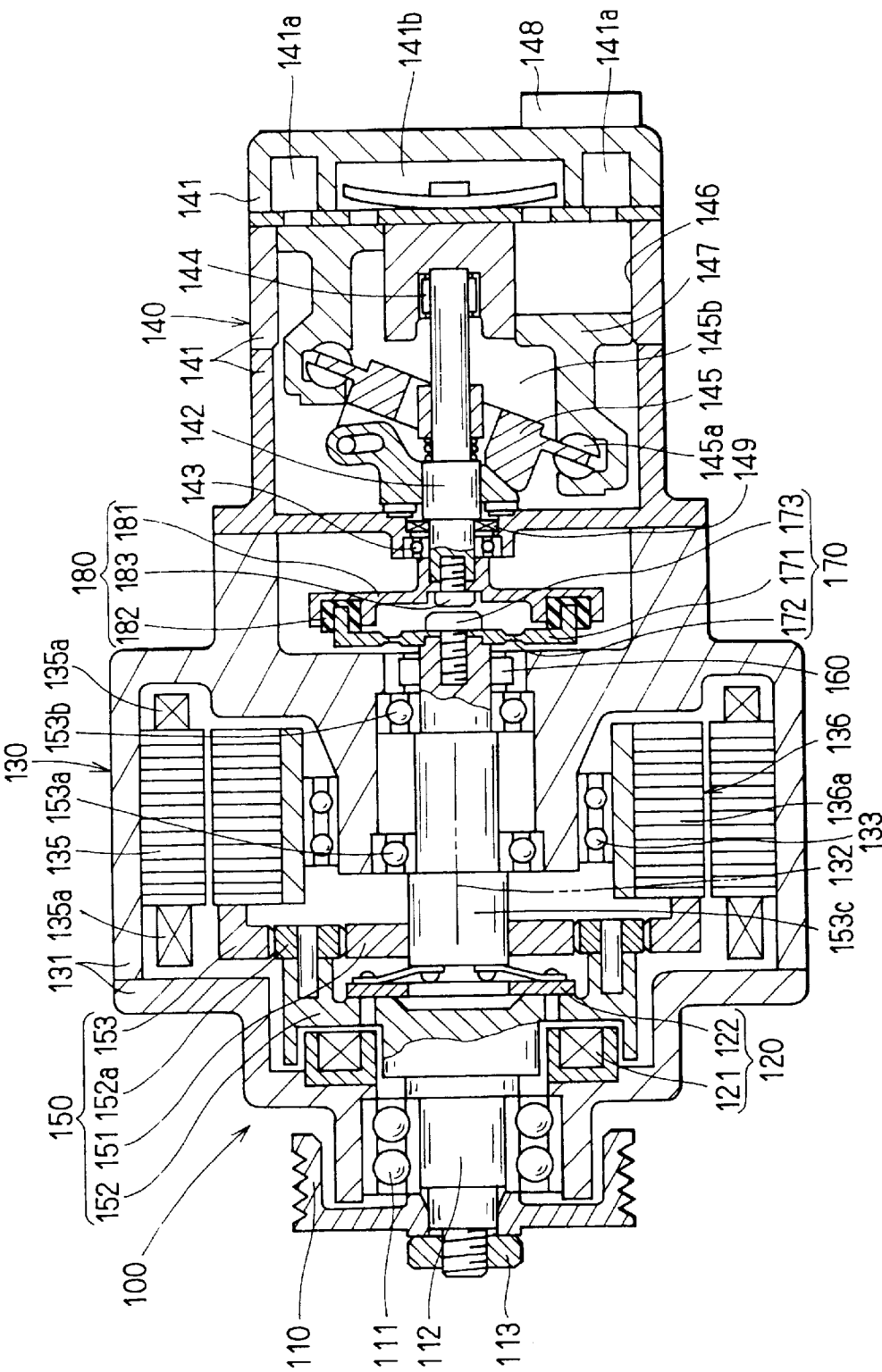


FIG. 3

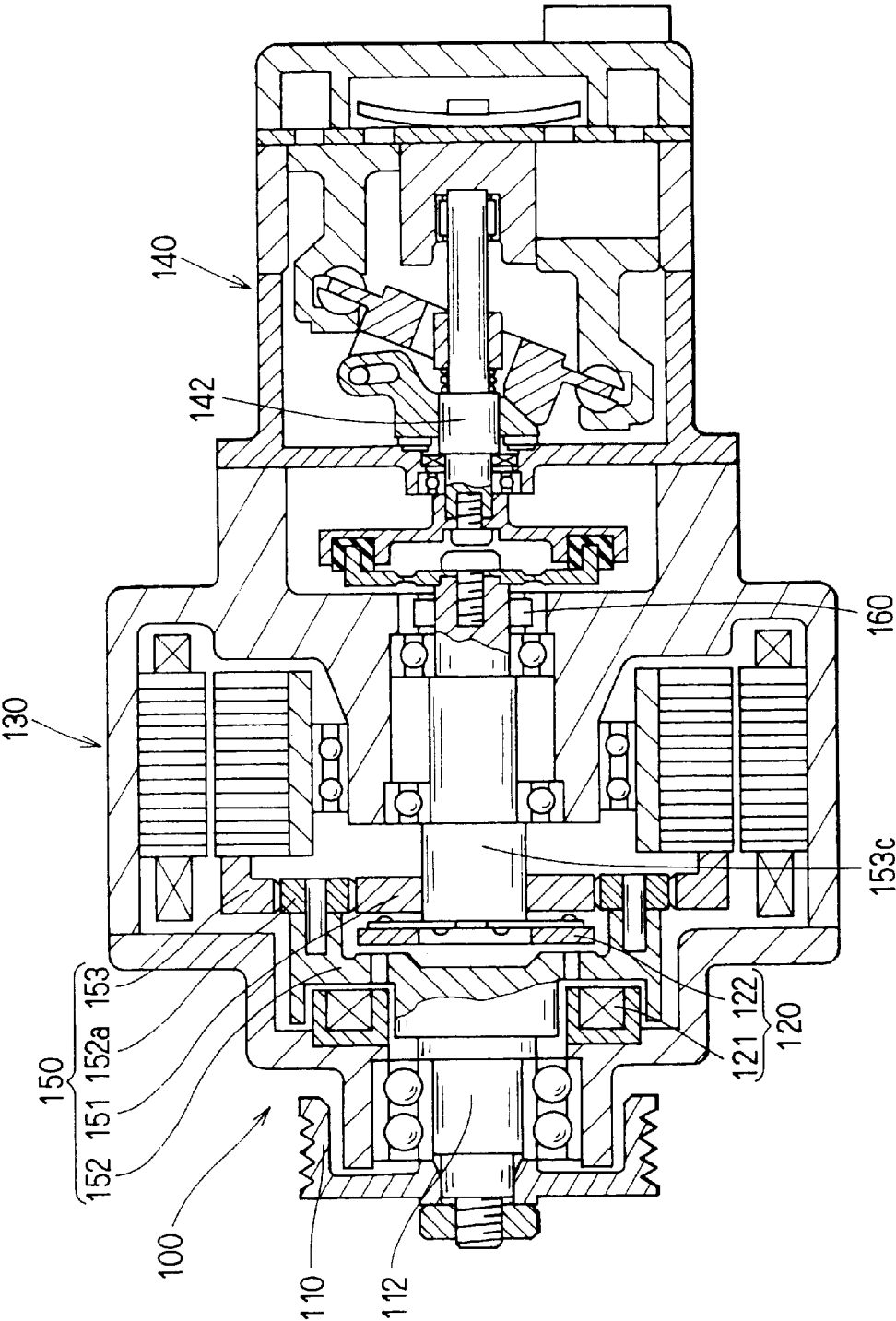


FIG. 4

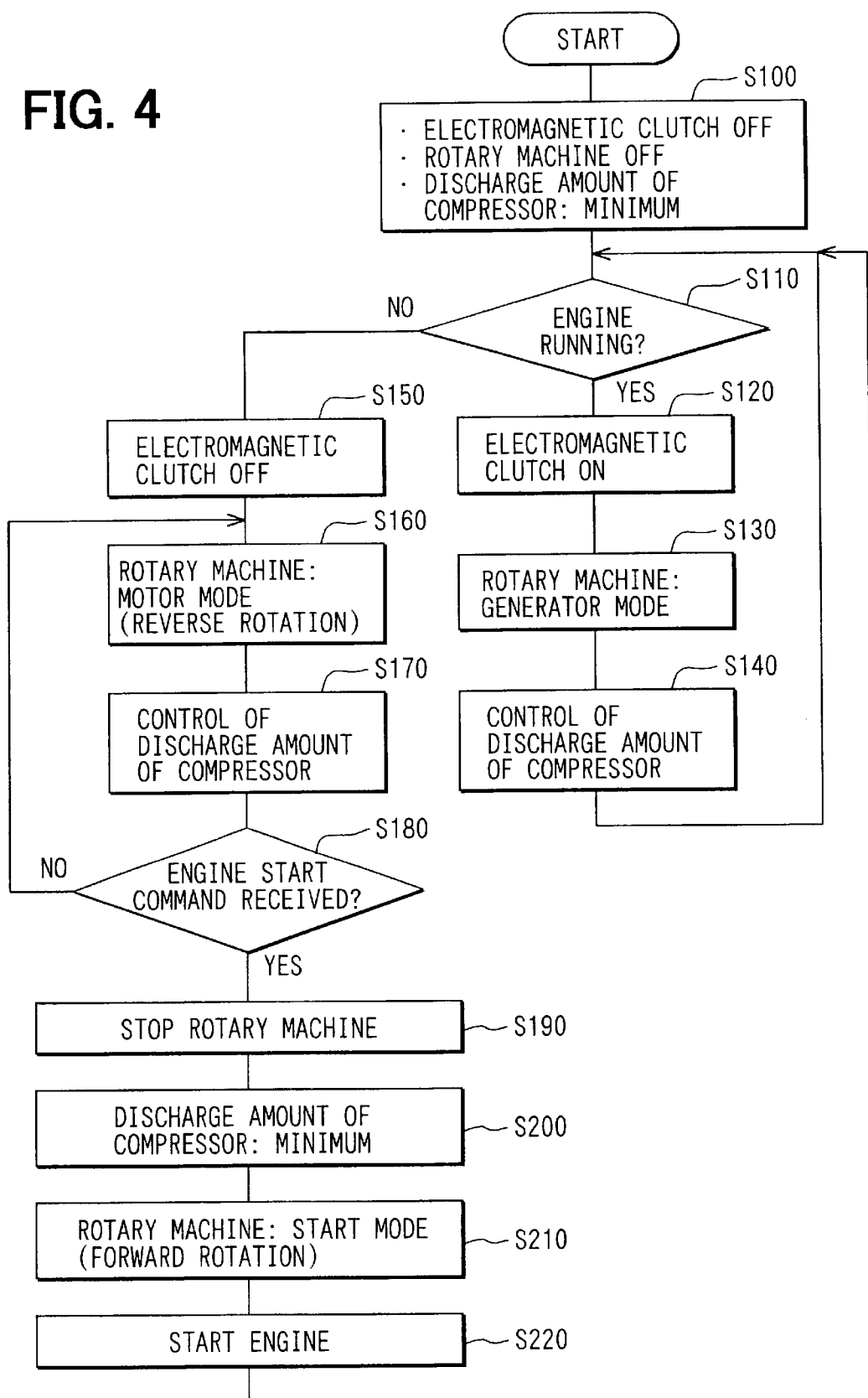


FIG. 5

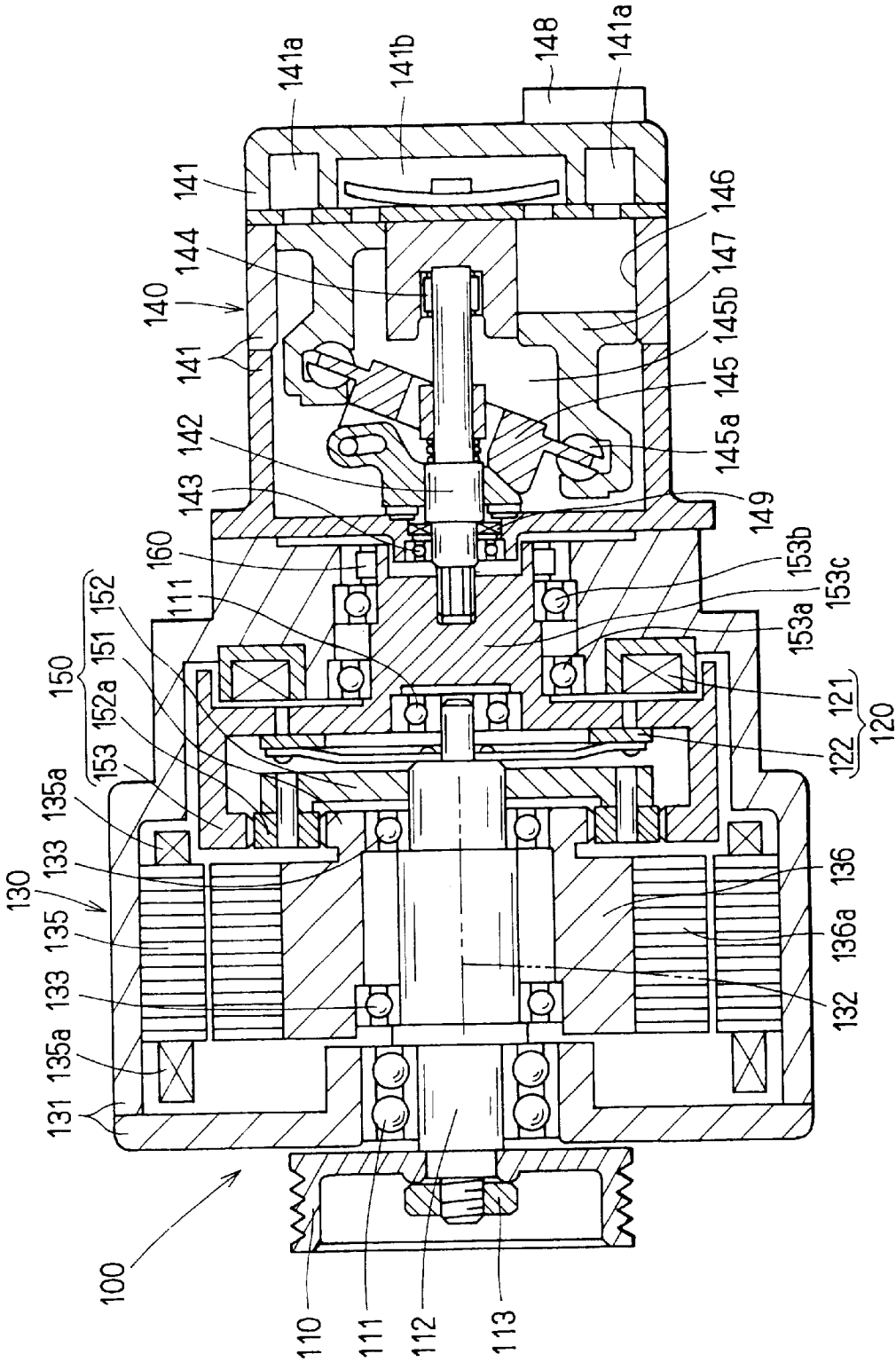


FIG. 6

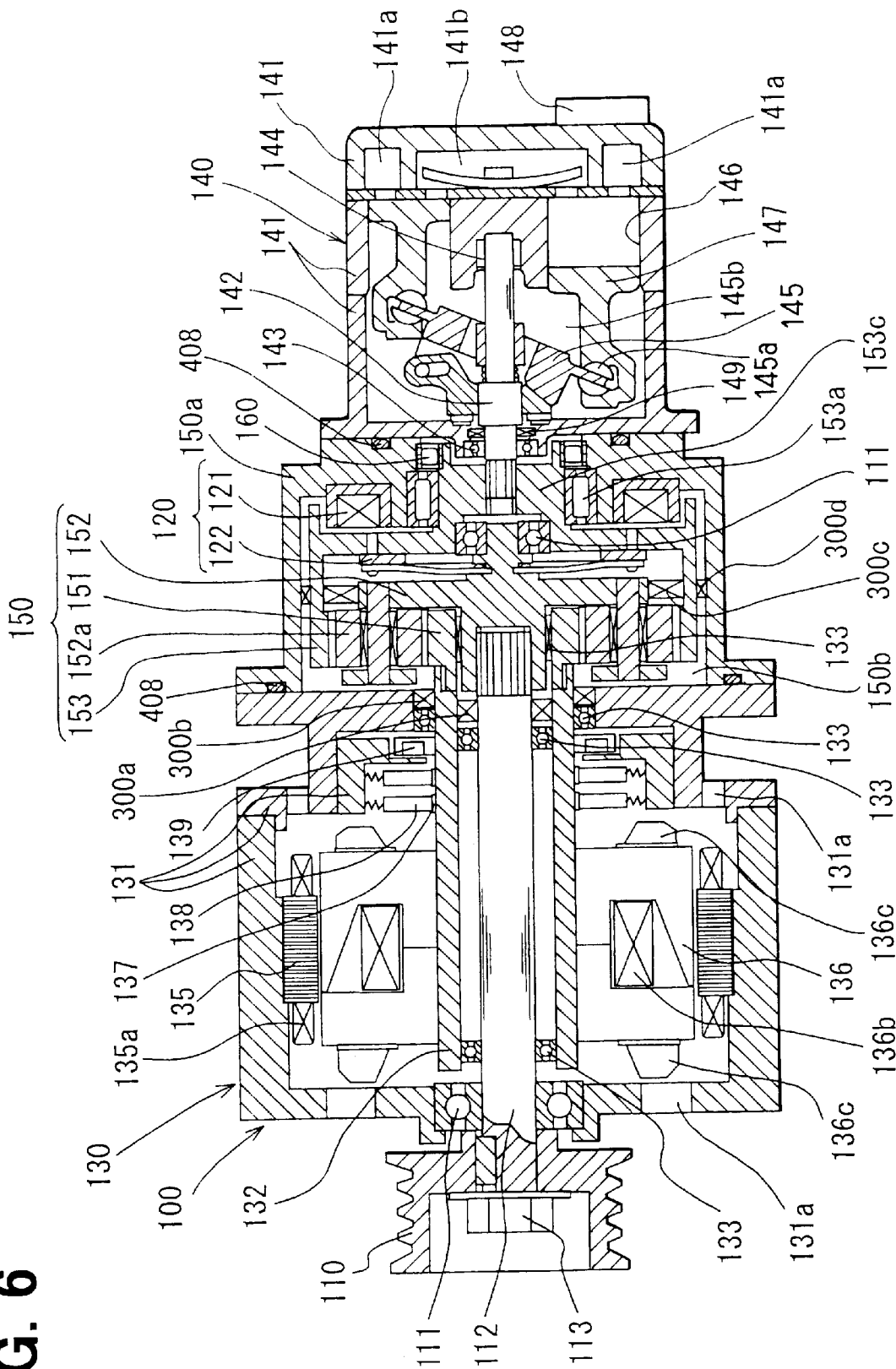


FIG. 7

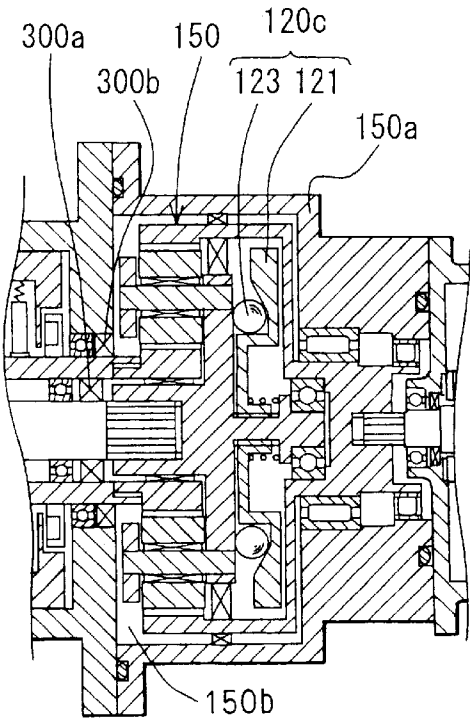


FIG. 8

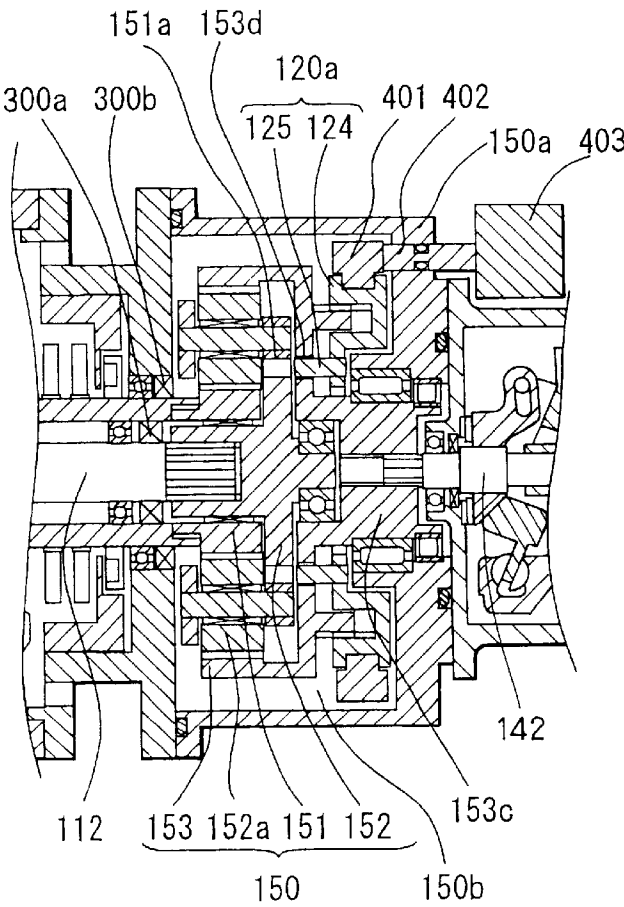
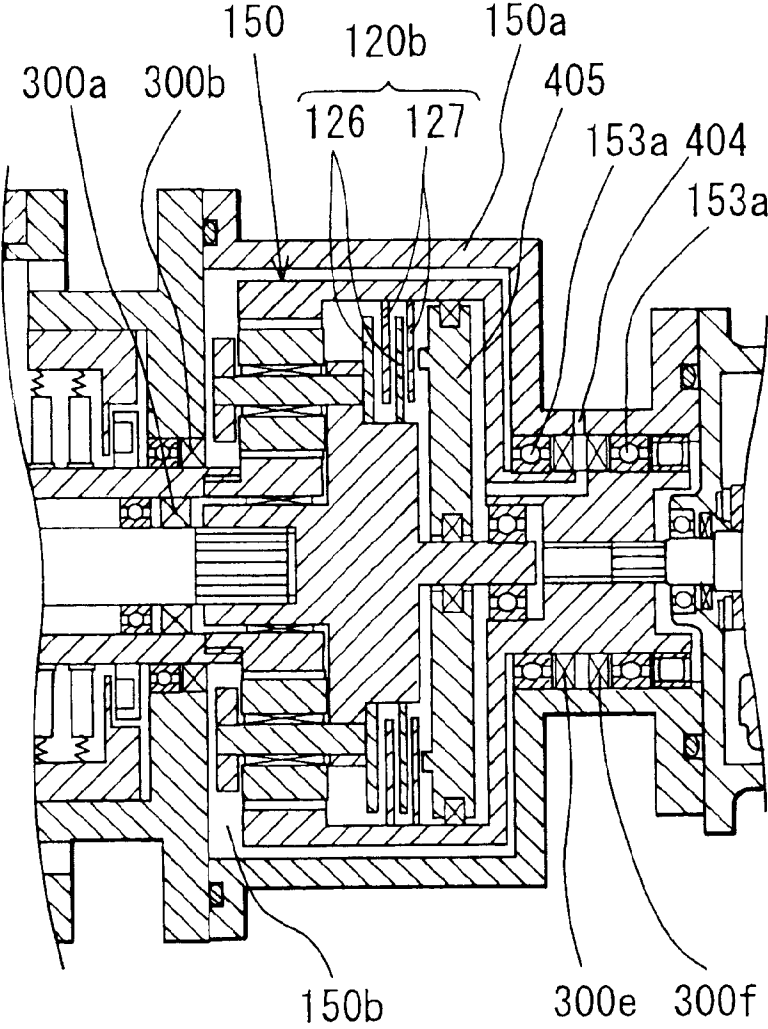


FIG. 9



COMPOSITE AUXILIARY MACHINE FOR VEHICLE AND CONTROL UNIT THEREOF

CROSS REFERENCE TO RELATED APPLICATIONS

This application is based upon, claims the benefit of priority of, and incorporates by reference, the contents of Japanese Patent Application No. 2002-125858 filed Apr. 26, 2002 and No. 2002-164359 filed Jun. 5, 2002.

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present invention relates to a composite auxiliary machine for a vehicle and a control unit thereof that are suitably applied to a refrigeration cycle device installed in a vehicle having an idling-stop function which automatically shuts off the engine when the vehicle stops.

2. Description of the Related Art

Vehicles incorporating the "idling-stop" function for better fuel economy have appeared on the market in recent years. With idling-stop the engine automatically shuts off when the vehicle stops moving, without turning off the ignition. In a conventional vehicle this means that the air-conditioning stops working during the stop because the compressor of the refrigeration cycle device also stops when the power from the engine is stopped.

The present inventors have shown a composite auxiliary machine for a vehicle in Japanese Patent Application No. 2001-111054 that can resolve this problem. This composite auxiliary machine for a vehicle combines an electric motor/generator with a compressor, and incorporates a torque distributing mechanism which is connected to the engine, motor/generator, and compressor respectively via engaging shafts. A locking mechanism is provided to the engaging shaft of the compressor, and a clutch is provided between the engaging shafts of the electric motor/generator and compressor.

This system not only allows the powering of the compressor by the electric motor/generator during stoppage of the engine but also enables flexible operation of the electric motor/generator and compressor in accordance with the operational state of the engine. The system also achieves simplification of its construction by integrating the electric motor/generator and its drive circuit.

Nevertheless, there is still a need for further improvement in the above invention with respect to the complexity in controlling both the locking mechanism and the clutch when regulating the operational performance of the compressor.

SUMMARY OF THE INVENTION

In view of the foregoing, an object of the present invention is to provide a composite auxiliary machine for a vehicle and a control unit thereof by which performance control of the compressor is readily achieved. To achieve the above object, the present invention employs the following technical means.

According to a first aspect of the present invention, a composite auxiliary machine for a vehicle includes a compressor (140) for compressing a refrigerant in a refrigeration cycle device (200), a rotary machine (130) functioning as a generator and as an electric motor, and a torque distributing mechanism (150) for distributing torque of a drive shaft (112). The drive shaft (112) is rotated by a driving force of a vehicle engine (10).

The torque distributing mechanism (150) distributes torque to a compressor shaft (142) of the compressor (140) and to a shaft (132) of the rotary machine (130), and distributes torque input from the shaft (132) of the rotary machine (130) to the drive shaft (112) and to the compressor shaft (142). Additionally, an intermittent means (120) provided to the torque distributing mechanism (150) connects and disconnects any two of the drive shaft (112), the compressor shaft (142), and the shaft (132) of the rotary machine (130). A locking mechanism (160) restricts rotation of the compressor shaft (142). The compressor (140) includes a variable displacement mechanism (145, 148) for varying an amount of discharge per turn of the compressor shaft.

In this composite auxiliary machine (100) for a vehicle, the intermittent means (120) is connected while the vehicle engine (10) is running, so that torque is transmitted from the drive shaft (112) to the compressor shaft (142) and the shaft (132) of the rotary machine (130) via the torque distributing mechanism (150) and the compressor (140) and the rotary machine (130) are activated simultaneously, whereby operation of the refrigeration cycle device (200) by the compressor (140) and power generation by the rotary machine (130) are both achieved.

While the compressor (140) is operating, the intermittent means (120) remains connected, and an amount of discharge of the compressor (140) is readily changeable in accordance with a predetermined thermal load of the refrigeration cycle device (200) by means of the variable displacement mechanism (145, 148). Because changing the amount of discharge does not involve switching the intermittent means (120), favorable drivability is achieved.

When the vehicle engine (10) is stopped, the intermittent means (120) is disconnected, and the rotary machine (130) is operated as an electric motor and provides power to the compressor (140) through the torque distributing mechanism (150). This permits the compressor (140) to continue providing air-conditioning. In this state also, in which the intermittent means (120) is disconnected, an amount of discharge of the compressor (140) is readily changeable by means of the variable displacement mechanism (145).

Operating the rotary machine (130) as an electric motor in this state with the intermittent means (120) being disconnected and the compressor shaft (142) being restricted by the locking mechanism (160), will rotate the drive shaft via the torque distributing mechanism (150) thereby restarting the vehicle engine (10).

According to a second aspect of the invention, a planetary gear (150) should preferably be used as the torque distributing mechanism (150). When the intermittent means (120) is connected while the engine (10) is running, because of the characteristics of the planetary gear (150), both the compressor shaft (142) and the shaft (132) of the rotary machine (130) rotate with the same rpm as that of the drive shaft. Accordingly, the rotary machine (130) can carry on stable power generation without being affected by changes in the amount of discharge of the compressor (140).

According to a third aspect of the invention, the locking mechanism (160) may be a one-way clutch (160) that allows forward rotation of the compressor shaft (142) and inhibits reverse rotation of the same. Thereby, the locking of the compressor shaft (142) at the time of starting the engine (10) is performed mechanically. Thus the locking mechanism (160) requires no control, and can be simply constructed.

According to a fourth aspect of the invention, the drive shaft (112) is connected to a planetary carrier (152) of the

planetary gear (150), the compressor shaft (142) is connected to a ring gear (153) of the planetary gear (150), the shaft (132) of the rotary machine (130) is connected to a sun gear (151) of the planetary gear (150), and the intermittent means (120) is disposed between the drive shaft (112) and the compressor shaft (142). Thereby, the rotary machine (130) can be constructed compactly and when working as an electric motor, it can provide the same level of power to the compressor (140) or to the vehicle engine (10) at reduced rpm but with increased torque transmission.

According to a fifth aspect of the invention, a limiter mechanism (170) is interposed between the torque distributing mechanism (150) and the compressor shaft (142) for discontinuing the torque transmission between the drive shaft (112) and the compressor shaft (142) and between the shaft (132) of the rotary machine (130) and the compressor shaft (142) when the compressor (140) is locked. Thereby, even if the compressor (140) should be locked for some reason, the vehicle engine (10) and the rotary machine (130) are protected. Therefore, functions related to the drive are not impaired.

According to a sixth aspect of the invention, the limiter mechanism (170) should preferably be disposed closer to the compressor (140) than the one-way clutch (160). Thereby, even when the limiter mechanism (170) is working, the one-way clutch (160) can still restrict the compressor shaft (142) for starting the vehicle engine (10).

The torque distributing mechanism (150) is a significant component as it achieves torque transmission, and thus needs to have good reliability in terms of durability. Therefore, according to a seventh aspect of the invention, the torque distributing mechanism (150) should preferably be located inside an enclosed space (150b), and the drive shaft (112), the shaft (132) of the rotary machine (130), and the compressor shaft (142) should preferably be provided with sealing members (300a-300d) at portions extending into the enclosed space (150b).

Thereby, pre-applied lubrication oil in the torque distributing mechanism (150) is prevented from leaking outside of the enclosed space (150b), and with this lubrication oil retained inside the space, the reliability and durability of the torque distributing mechanism (150) is improved. This arrangement also prevents reliability degradation of the rotary machine (130) or the intermittent means (120) caused by contamination of the rotary machine (130) and intermittent means (120) with lubrication oil, particularly when the rotary machine (130) is a brush type and the intermittent means (120) performs connection and disconnection functions by friction.

According to an eighth aspect of the invention, the intermittent means (120) may be a clutch (120a, 120b) that works with lubrication oil and that is located inside the enclosed space (150b) together with the torque distributing mechanism (150). In such a case, the reliability of both the torque distributing mechanism (150) and the clutch (120a, 120b) is improved because of the retained lubrication oil.

According to a ninth aspect of the invention, the rotary machine (130) may be a brushless rotary machine (130) and be located inside the enclosed space (150b) together with the torque distributing mechanism (150). In this case, oil seals (300b) for the shaft (132) of the rotary machine (130) can be omitted.

According to a tenth aspect of the invention, the intermittent means (120) is a clutch (120a, 120b) that works with lubrication oil. The rotary machine (130) is a brushless rotary machine (130), and the intermittent means (120), the

rotary machine (130), and the torque distributing mechanism (150) are supplied with the refrigerant.

With such a construction, the rotary machine (130) is cooled by the refrigerant, and will have better durability. It can also be constructed more compactly. Because the rotary machine (130) is of a brushless type, there is no risk that lubrication oil in the refrigerant will cause trouble to power supplied elements. Also, the torque distributing mechanism (150) and clutch (120a, 120b) will benefit from the lubrication oil in the refrigerant and will thereby have better durability and operability.

According to an eleventh aspect of the present invention, a control unit of a composite auxiliary machine for a vehicle has a function of automatically shutting off its vehicle engine (10) in accordance with its driving state. The control unit includes the composite auxiliary machine for a vehicle as set forth in any one of the first to tenth aspects, and control means (190) for controlling operations of the variable displacement mechanism (145, 148), the rotary machine (130), and connection and disconnection of the intermittent means (120).

The control of the composite auxiliary machine for a vehicle includes connecting the intermittent means (120) while the vehicle engine (10) is running, disconnecting the intermittent means (120) while the vehicle engine (10) is not running, and causing the rotary machine (130) to rotate in a reverse direction so as to operate as an electric motor for driving the compressor (140). Additionally, control consists of disconnecting the intermittent means (120) when starting the vehicle engine (10), causing the rotary machine (130) to rotate in a forward direction so as to operate as an electric motor for starting the vehicle engine (10), and operating the variable displacement mechanism (145, 148) while the compressor (140) is operating to change an amount of discharge of the compressor (140) in accordance with a predetermined thermal load of the refrigeration cycle device (200).

Further areas of applicability of the present invention will become apparent from the detailed description provided hereinafter. It should be understood that the detailed description and specific examples, while indicating the preferred embodiment of the invention, are intended for purposes of illustration only and are not intended to limit the scope of the invention. Reference numerals described in each of the above means correspond with means of the preferred embodiments described later.

BRIEF DESCRIPTION OF THE DRAWINGS

The present invention will become more fully understood from the detailed description and the accompanying drawings, wherein:

FIG. 1 is a diagram illustrating construction of a first embodiment of the present invention as applied to a refrigeration cycle device in a vehicle having an idling-stop function;

FIG. 2 is a cross-sectional view of the composite auxiliary machine for a vehicle of FIG. 1;

FIG. 3 is a cross-sectional view of the composite auxiliary machine for a vehicle of FIG. 2 in which the electromagnetic clutch is disengaged;

FIG. 4 is a flow chart of the control of the composite auxiliary machine for a vehicle;

FIG. 5 is a cross-sectional view of a composite auxiliary machine for a vehicle according to a second embodiment;

FIG. 6 is a cross-sectional view of a composite auxiliary machine for a vehicle according to a third embodiment;

FIG. 7 is a cross-sectional view of a modified example 1 of the composite auxiliary machine for a vehicle of the third embodiment;

FIG. 8 is a cross-sectional view of a composite auxiliary machine for a vehicle according to a fourth embodiment;

FIG. 9 is a cross-sectional view of a modified example of the composite auxiliary machine for a vehicle of the fourth embodiment; and

FIG. 10 is a cross-sectional view of a composite auxiliary machine for a vehicle according to a fifth embodiment.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

The following description of the preferred embodiment(s) is merely exemplary in nature and is in no way intended to limit the invention, its application, or uses. (First Embodiment)

FIG. 1 to FIG. 4 illustrate a first embodiment of the present invention. First, its specific structures will be described with reference to FIG. 1 and FIG. 2.

Reference numeral 100A in FIG. 1 denotes a control unit of a composite auxiliary machine for a vehicle, which includes a composite auxiliary machine for a vehicle (hereinafter "composite auxiliary machine") 100 and a controller 190. This embodiment shows one example in which the control unit 100A is applied to a refrigeration cycle device 200 mounted on a vehicle having the idling-stop function, which shuts off the vehicle engine 10 when the vehicle comes to a rest. The engine 10 includes an rpm sensor 20.

The refrigeration cycle device 200 includes a compressor 140 that is part of the composite auxiliary machine 100, and creates therein a known refrigeration cycle. The compressor 140 compresses the refrigerant in the cycle to a high temperature, high pressure gas. A refrigerant piping system 240 connects the following elements to the compressor 140 to form a closed loop: a condenser 210 for condensing and liquidizing the compressed refrigerant, an expansion valve 220 for allowing the liquid refrigerant to expand in a thermally insulating manner, and an evaporator 230 for evaporating the expanded refrigerant, thereby absorbing latent heat of vaporization and cooling the air passing therethrough.

A temperature sensor 231 is provided on the downstream side of the air flow of the evaporator 230 for detecting the temperature (Te) of the cooled air behind the evaporator. An A/C command signal carries information of the temperature set by a vehicle occupant, and the difference between the air temperature Te behind the evaporator and the set temperature is used as a representative value of the thermal load of the refrigeration cycle device 200. The larger the difference between the air temperature Te behind the evaporator and set temperature is, the larger the thermal load of the refrigeration cycle device 200 is, i.e., required amount of refrigerant discharged from the compressor 140 is increased.

The composite auxiliary machine 100 is made up integrally of a drive pulley 110, an electromagnetic clutch 120 as intermittent means, a rotary machine 130 (auxiliary machine), and the compressor 140 (auxiliary machine). A more detailed description will be given below with reference to FIG. 2.

The drive pulley 110 is fixed to a drive shaft 112 at the center by a nut 113. The drive shaft 112 is rotatably supported by a bearing 111 disposed at one end of the housing 131 of the rotary machine, so that the driving force from the engine 10 is transmitted as a rotating force through a crank pulley 30 and a belt 40 shown in FIG. 1 to the drive pulley 110.

On the other side of the drive pulley of the drive shaft 112 is provided a planetary gear 150 which constitutes a torque distributing mechanism. The planetary gear 150 includes, as is well known, a sun gear 151 at the center, a planetary carrier 152 revolved therearound by pinion gears 152a on the outer periphery of the sun gear 151, and a ring gear 153 arranged on the outer peripheral side of the pinion gears 152a. One end of the drive shaft 112 on the opposite side of the drive pulley is connected to the planetary carrier 152.

The rotary machine 130 is a "motor/generator" that can function both as a generator and an electric motor, and is comprised of a stator 135 with coils 135a fixed on an inner peripheral face of the housing 131, and a rotor 136 with a permanent magnet 136a on its outer periphery rotatably supported by a bearing 133 fixed to the housing 131. An outer peripheral portion on one side of the rotor 136 is connected to the ring gear 153 of the planetary gear 150. This means that the shaft 132 of the rotary machine 130 is connected to the ring gear 153 because it passes through the center of the bearing 133 as indicated by a two-dot chain line in FIG. 2. The bearing 133 rotatably supports the rotor 136.

An intermediate shaft 153c is rotatably supported by bearings 153a, 153b fixed on an inner side face of the housing 131 of the rotary machine. Its one end on the side of the drive pulley 110 is connected to the sun gear 151 of the planetary gear 150. Provided at the other end of the intermediate shaft 153c opposite the sun gear is a one-way clutch 160. The one-way clutch 160 forms a locking mechanism, and fixed to the housing 131 of the rotary machine, it allows forward rotation of the intermediate shaft 153c and inhibits its reverse rotation by altering states of engagement therebetween.

The compressor 140 is a variable displacement compressor of a swash plate type having a swash plate 145 and a control valve 148 for changing its amount of discharge. The discharge amount of refrigerant is made variable by changing the amount of discharge per turn of the compressor shaft. The discharge amount of refrigerant is expressed by a product of the amount of discharge and rpm of the compressor 140.

The compressor shaft 142 is rotatably supported by bearings 143, 144 fixed to the compressor housing 141. The swash plate 145 is mounted on the compressor shaft 142 and is located inside a swash plate chamber 145b. A plurality of pistons 147 are coupled to the outer periphery of the swash plate 145 via shoes 145a.

At one end of the compressor 140 on the right side of FIG. 2 is disposed the control valve 148, which controls pressure inside the swash plate chamber 145b by adjusting the degree of opening, thereby distributing the pressure inside a discharge chamber 141b into the swash plate chamber 145b or releasing the pressure inside the swash plate chamber 145b into a suction chamber 141a. The inclination angle of the swash plate 145 changes in accordance with the pressure, and so does the stroke of the pistons 147, whereby the amount of discharge is varied.

The inclination angle of the swash plate 145 can be changed continuously. When the angle is at maximum level as shown in FIG. 2, the stroke of the pistons 147 is the largest, whereby a maximum amount of discharge of the compressor 140 is attained. When the inclination angle of the swash plate 145 is at a minimum level where it is substantially orthogonal to the compressor shaft 142, the stroke of the pistons 147 is the smallest and the amount of discharge then is almost zero.

The intermediate shaft 153c and the compressor shaft 142 are connected to each other via a limiter mechanism 170 and

a damping assembly **180**. The limiter mechanism **170** includes a disk-like member **171** formed with a thin portion **172** at its radial center and fastened to the intermediate shaft **153c** by a bolt **173**. The thin portion **172** breaks when, while the outer periphery of the disk-like member **171** is fixedly held, a torque of more than a predetermined value is applied to the central portion of the limiter mechanism **170** via the intermediate shaft **153c**.

The damping assembly **180** includes a disk-like member **181** and a resilient member **182** made of rubber or the like melt-bonded to the outer periphery of the disk-like member **181**. It is fastened to the compressor shaft **142** by a bolt **183**. Torque variations during the operation of the compressor **140** are absorbed by this resilient member **182** and thus are not transmitted to the rotary machine **130** side.

The resilient member **182** of the damping assembly **180** is further melt-bonded to the outer periphery of the disk-like member **171** of the limiter mechanism **170** so that the intermediate shaft **153c** and compressor shaft **142** are connected to each other. The compressor shaft **142** is thus connected to the sun gear **151** of the planetary gear **150** through the intermediate shaft **153c**. The limiter mechanism **170** is located closer to the compressor **140** than the one-way clutch **160**.

To prevent the refrigerant inside the compressor housing **141** from leaking toward the rotary machine **130** side, a sealing assembly **149** is provided on the side of the bearing **143** of the compressor shaft **142**. The planetary gear **150** is provided with an electromagnetic clutch **120** which acts as intermittent means for connecting and disconnecting the drive shaft **112** to and from, respectively, the intermediate shaft **153c**, i.e., the drive shaft **112** and compressor shaft **142**. The electromagnetic clutch **120** includes coils **121** fixed to the housing **131** of the rotary machine and a hub **122** connected to an end face of the intermediate shaft **153c**. The planetary carrier **152** of the planetary gear **150** is located in between these coils **121** and the hub **122**.

When the electromagnetic clutch **120** is turned on, i.e., when electric current is applied to the coils **121**, the hub **122** is attracted toward and touches the planetary carrier **152**, thereby engaging the drive shaft **112** with the intermediate shaft **153c** and compressor shaft **142**. When the electromagnetic clutch **120** is turned off, i.e., when the current supply to the coils **121** is cut off, the hub **122** moves away from the planetary carrier **152**, whereby the intermediate shaft **153c** and compressor shaft **142** are disconnected from the drive shaft **112**. FIG. 3 illustrates this state.

Referring back to FIG. 1, the controller **190** processes various information such as a signal from the rpm sensor **20** indicating the rpm of the engine **10**, an engine start command signal, an A/C command signal, and a signal from the temperature sensor **231** indicating the air temperature behind the evaporator, and determines the thermal load of the refrigeration cycle device **200** based on these signals. At the same time the controller **190** determines the state of the traveling vehicle and controls the operations of the electromagnetic clutch **120**, rotary machine **130**, and variable displacement mechanism **145**, **148** of the compressor **140** as required.

The following is a description of control by the controller **190** and operation of the composite auxiliary machine **100** with reference to the flowchart of FIG. 4. The initial settings at step **S100** are that the electromagnetic clutch **120** is turned off, the rotary machine **130** is turned off, and the amount of discharge of the compressor **130** is zero or minimal.

At step **S110**, the controller judges whether the engine **10** is running or not based on a detection signal from the rpm

sensor **20**. If the rpm is more than zero, the rotary machine **130** and the compressor **140** are activated using the driving force from the engine **10**.

That is, the electromagnetic clutch **120** is turned on at the next step **S120** so as to connect the drive shaft **112** and compressor shaft **142**, whereby the latter is rotated with the same rpm as the drive shaft **112**. Because the rotor **136** (and the shaft **132** of the rotary machine) connected to the planetary gear **150** is also rotated with the same rpm, this is followed by step **S130**, a power generation mode, in which the rotary machine **130** acts as a generator, and the battery **20** is charged with an electric current generated in the stator **135**. Since the amount of discharge of the compressor **140** at this time is zero or minimal, the torque required to activate the compressor **140** is minimal, so the electromagnetic clutch **120** is turned on smoothly without causing shock.

At step **S140**, the amount of discharge of the compressor **140** is varied. Specifically, the degree of opening of the control valve **148** and the inclination angle of the swash plate **145** are adjusted, so as to change the stroke of the pistons **147**, to attain a required discharge amount of refrigerant in accordance with the thermal load of the refrigeration cycle device **200**. The steps **S110**–**S140** are repeated when the engine **10** is running.

If the controller determines that the engine rpm is zero, i.e., the engine **10** is at a halt due to the idling-stop function at step **S110**, then it begins powering the compressor **140** by the rotary machine **130**. The electromagnetic clutch **120** is turned off at step **S150**, and power is supplied from the battery **50** to the coils **135a** of the stator **135** at step **S160**, whereby the rotor **136** is rotated. Thus the compressor **140** is operated through the planetary gear **150** by the rotary machine **130** acting as an electric motor. Since the engine **10** is stopped and the drive shaft **112** is stationary, the compressor **140** is operated in the forward direction by rotating the rotor **136** or the shaft **132** of the rotary machine in the reverse direction.

At step **S170**, the amount of discharge of the compressor **140** is varied similarly to step **S140**, so as to attain a required discharge amount of refrigerant in accordance with the thermal load of the refrigeration cycle device **200**. In this idling-stop state, if an engine start command signal is input, the control of the rotary machine **130** is executed in the following sequences **S190**–**S210** to start the engine **110**.

First, the rotary machine **130** is stopped at step **S190**. In addition, the amount of discharge of the compressor **140** is set to be zero or minimal at step **S200**, so that the electromagnetic clutch **120** can be turned on smoothly without causing shock the next time round. At step **S210**, the rotary machine **130** is rotated in the forward direction so that it functions as an electric motor or starter. While the compressor shaft **142** tries to rotate in the reverse direction via the planetary gear **150** in this engine start mode, its rotation is inhibited by the one-way clutch **160**. Thus the drive shaft **112** alone is rotated by the electric motor, whereby the engine **10** is started at step **S220**. If no engine start command has been input at step **S180**, steps **S160** and **S170** are to be repeated.

To summarize, the characteristic features of the composite auxiliary machine **100** and its control unit **100A** according to the present invention are as follows: while the engine **10** is running, it provides power to the compressor **140** and the rotary machine **130**, because the electromagnetic clutch **120** enables torque transmission from the drive shaft **112** to the compressor shaft **142** and the shaft **132** of the rotary machine via the planetary gear **150**, and so the compressor **140** operates the refrigeration cycle device **200** and the rotary

machine **130** performs power generation. While the compressor **140** is working, the electromagnetic clutch **120** remains in the ON-state, and the amount of discharge can be varied in accordance with the thermal load of the refrigeration cycle device **200** by controlling the variable displacement mechanism **145**, **148**. Since no switching of the electromagnetic clutch **120** is necessary during this time, good drivability is achieved.

The planetary gear **150** has characteristics that enable both the compressor shaft **142** and the shaft **132** of the rotary machine to rotate with the same rpm as that of the drive shaft **112** when the electromagnetic clutch **120** is turned on. Accordingly, the rotary machine **130** can carry on stable power generation without being affected by changes in the amount of discharge of the compressor **140**.

When the engine **10** is stopped, the electromagnetic clutch **120** is turned off, so that the rotary machine **130** acts as an electric motor, by which the compressor **140** is operated via the planetary gear **150** to provide continuous air-conditioning. Variable control of the amount of discharge by the variable displacement mechanism **145**, **148** is also possible while the electromagnetic clutch **120** is in the OFF-state.

To restart the engine **10**, the rotary machine **130** is operated as an electric motor, with the electromagnetic clutch **120** being turned off, and the compressor shaft **142** being restricted in one direction by the one-way clutch **160**. Thus the drive shaft **112** is rotated via the planetary gear **150** to start the engine **10**. Because the locking mechanism is constructed with the one-way clutch **160**, the locking of the compressor shaft **142** at the time of restarting the engine **10** is performed mechanically. Thus the locking mechanism requires no control, and can be simply constructed.

Furthermore, because of the limiter mechanism **170** provided between the planetary gear **150** and the compressor shaft **142** closer to the compressor than the one-way clutch **160**, should the compressor **140** be locked for some reason, the engine **10** and the rotary machine **130** are protected, so that deterioration of driving performance is not experienced. Even when the limiter mechanism **170** is working, the one-way clutch **160** can still restrict the compressor shaft **142** when restarting the engine **10**.

(Second Embodiment)

FIG. 5 illustrates a second embodiment of the present invention, in which the arrangement of the shaft **132** of the rotary machine and the compressor shaft **142** relative to the planetary gear **150** is different from that of the foregoing embodiment. The rotor **136** or the shaft **132** of the rotary machine **130** is connected to the sun gear **151** of the planetary gear **150**, and the intermediate shaft **153c** is connected to the ring gear **153** of the planetary gear **150**. The limiter mechanism **170** and the damping assembly **180** present in the previous embodiment are omitted here and the intermediate shaft **153c** is directly connected to the compressor shaft **142**.

This arrangement enables the rotary machine **130** to be constructed more compactly, because when working as an electric motor, it can provide the same level of power to the compressor **140** or to the engine **10** at reduced rpm with increased torque transmission.

(Third Embodiment)

FIG. 6 illustrates a third embodiment of the present invention, in which part of the structure of the rotary machine **130** is changed, and in which the planetary gear **150** is placed in an enclosed space **150b** as compared to the second embodiment.

The shaft **132** of the rotary machine is cylindrical and supported rotatably by bearings **133**. The rotary machine

130 is a brush-type, having a brush **138** in contact with a slip ring **137** electrically connected to coils **136b** of the rotor **136**. Power generation control of the rotary machine **130** having this generator structure is easier. An angle sensor **139** is attached to the shaft **132** for detecting the rotation angle position of the rotor **136**, so that electric current is supplied to the rotary machine **130** when it is working as an electric motor in accordance with the rotation angle position of the rotor **136** to achieve a required rotation speed. A plurality of fans **136c** are provided on side faces of the rotor **136**, and a plurality of air-cooling apertures **131a** are formed at either end of the housing **131** of the rotary machine, so that outside air will pass through the inside of the housing **131** for air cooling while the rotary machine **130** is in operation.

The planetary gear **150** is accommodated in an enclosed space **150b** inside a middle housing **150a** between the housing **131** of the rotary machine and the housing **141** of the compressor. Sealing members such as oil seals **300a** and **300b** are respectively provided at portions where the drive shaft **112** and the shaft **132** of the rotary machine extend into the enclosed space **150b** inside the middle housing **150a**. Also, oil seals **300c** and **300d** are provided to the ring gear **153** connected to the compressor shaft **142** between the planetary gear **150** and the electromagnetic clutch **120**. Between the housing **131** of the rotary machine and the middle housing **150a**, and between the middle housing **150a** and the housing **141** of the compressor are provided respective packings **408** for seals.

Thereby, pre-applied lubrication oil in the planetary gear **150** is prevented from leaking outside of the enclosed space **150b**. With the lubrication oil well retained inside the space **150b**, the reliability and durability of the planetary gear **150** is improved. This arrangement also prevents reliability degradation of rotary machine **130** and electromagnetic clutch **120** caused by contamination of the brush **138** and electromagnetic clutch **120** with lubrication oil.

The electromagnetic clutch **120** may be constructed as a centrifugal clutch as shown in FIG. 7, in which the connecting and disconnecting of shafts are achieved by balls **123** that move toward an outside diameter by centrifugal force. This structure is simpler and less expensive.

(Fourth Embodiment)

FIG. 8 illustrates a fourth embodiment, in which the clutch used as intermittent means is a lubricated mechanical clutch **120a** as compared to the third embodiment. The mechanical clutch **120a** includes a sleeve **124**, a plurality of pins **125** fitted thereto, and a shift fork **401** coupled to an outer peripheral portion of the sleeve. The shift fork **401** is joined to a rod **402**, which is moved by an actuator **403** to cause the sleeve **124** to slide in an axial direction of the intermediate shaft **153c**.

The ring gear **153** of the planetary gear **150** has insertion holes **153d** opposite the pins **125**, in which the pins **125** are fitted. The sun gear **151** has engaging holes **151a** opposite the pins **125**. When the sleeve **124** slides towards the sun gear **151**, the pins **125** fit into the holes **151a**, whereby the sun gear **151** and the ring gear **153** are engaged with each other, and the drive shaft **112** and compressor shaft **142** are connected to each other.

Because the mechanical clutch **120a** achieves the engaging and disengaging of both shafts by a sliding movement and engagement, lubrication oil is pre-applied to the mechanical clutch **120a** for ensuring its reliability in terms of performance and durability. It is also located in the enclosed space **150b** inside the middle housing **150a** together with the planetary gear **150**.

Therefore, the lubrication oil of the mechanical clutch **120a** is well retained as with the planetary gear **150**, and they

both have improved reliability. Moreover, because the oil seals **300c** and **300d** provided between the planetary gear **150** and clutch as in the third embodiment are omitted, a cost reduction is achieved.

The mechanical clutch **120a** may also be a wet multi-plate clutch **120b** as shown in FIG. 9, in which a number of plates **126** and **127** provided to the sun gear **151** and the ring gear **153**, respectively, perform the connecting and disconnecting functions. The same effects as described above will thereby be achieved.

The plates **126** and **127** of the wet multi-plate clutch **120b** work with lubrication oil, as is well known. They are moved by a piston **405** which is slid by oil pressure applied through an inlet **404** provided with oil seals **300e** and **300f**. (Fifth Embodiment)

FIG. 10 illustrates a fifth embodiment of the present invention in which refrigerant from the refrigeration cycle device **200** is supplied to the rotary machine **130**, the planetary gear **150**, and the mechanical clutch **120a**.

The composite auxiliary machine **100** has basically the same structure as in the fourth embodiment. The housing **131** of the rotary machine and the middle housing **150a** are respectively formed with apertures **131b**, **131c** and **150c** for refrigerant. The aperture **150c** communicates to the suction chamber **141a** of the compressor **140** through a linking passage **406**. The aperture **131b** communicates to the evaporator **230** of the refrigeration cycle device **200**. A lip seal **407** is provided at the end of the drive shaft **112** on the side of the drive pulley **110** for forming an air tight seal.

The rotary machine **130** is a brushless type, whose rotor **136** has a permanent magnet **136a** and requires no power supply. The intermittent means is the mechanical clutch **120a** as with the fourth embodiment. Refrigerant from the evaporator **230** flows into the rotary machine **130** through the aperture **131b**, and further into the enclosed space **150b** inside the middle housing **150a** through the aperture **131c**. In the enclosed space **150b** inside the middle housing **150a** the planetary gear **150** and mechanical clutch **120a** are located. The refrigerant then flows into the suction chamber **141a** of the compressor **140** through the aperture **150c** and the linking passage **406**.

The rotary machine **130** is thus cooled by the refrigerant, which permits the rotary machine **130** to be more durable. It can also be compactly constructed. Because the rotary machine **130** is a brushless type, there is no risk that lubrication oil in the refrigerant will cause trouble to elements that receive power.

The planetary gear **150** and the mechanical clutch **120a** will benefit from the lubrication oil in the refrigerant. They will thereby have better durability and operability. As an alternative, the refrigerant may be circulated from the discharge chamber **141b** of the compressor **140** into the enclosed space **150b** in the middle housing **150a** and inside the rotary machine **130**.

(Other Embodiments)

Various other modifications can be made to the above embodiments. For example, a differential gear mechanism may be employed as the torque distributing mechanism in place of the planetary gear **150**. The electromagnetic clutch **120** may not necessarily be provided between the drive shaft **112** and the compressor shaft **142**, but between the drive shaft **112** and the shaft **132** of the rotary machine, or between the compressor shaft **142** and the shaft **132** of the rotary machine.

An internal component that constitutes the rotary machine **130** may be accommodated inside the enclosed space **150b** in the middle housing **150a** together with the planetary gear

150. By adopting a brushless type rotary machine, the oil seal **300b** for the shaft **132** of the rotary machine can be omitted.

The present invention can be applied not only to the vehicles having the idling-stop function but also to gasoline-electric hybrid vehicles and other vehicles. The torque distributing mechanism is described as a means which distributes the torque, however, it may be a mechanism that distributes power. Therefore, the description of the invention is merely exemplary in nature and, thus, variations that do not depart from the gist of the invention are intended to be within the scope of the invention. Such variations are not to be regarded as a departure from the spirit and scope of the invention.

What is claimed is:

1. A composite auxiliary machine for a vehicle, comprising:

a compressor for compressing a refrigerant in a refrigeration cycle device;

a rotary machine that functions as a generator and as an electric motor;

a torque and power distributing mechanism for distributing torque of a drive shaft rotated by a driving force of a vehicle engine to a compressor shaft of said compressor and to a shaft of said rotary machine, and for distributing torque and power input from said shaft of the rotary machine to said drive shaft and to said compressor shaft;

intermittent means provided to said torque and power distributing mechanism for connecting and disconnecting any two of said drive shaft, said compressor shaft, and said shaft of the rotary machine; and

a locking mechanism for restricting rotation of said compressor shaft, wherein

said compressor includes a variable displacement mechanism for varying an amount of discharge per turn of the compressor shaft.

2. The composite auxiliary machine for a vehicle according to claim 1, wherein said torque and power distributing mechanism is a planetary gear.

3. The composite auxiliary machine for a vehicle according to claim 1, wherein said locking mechanism is a one-way clutch that permits forward rotation of said compressor shaft and inhibits reverse rotation of said compressor shaft.

4. The composite auxiliary machine for a vehicle according to claim 2, wherein

said drive shaft is connected to a planetary carrier of said planetary gear,

said compressor shaft is connected to a ring gear of said planetary gear,

said shaft of the rotary machine is connected to a sun gear of said planetary gear, and

said intermittent means is disposed between said drive shaft and said compressor shaft.

5. The composite auxiliary machine for a vehicle according to claim 1, further comprising:

a limiter mechanism interposed between said torque and power distributing mechanism and said compressor shaft, wherein the limiter mechanism discontinues the torque and power transmission between said drive shaft and said compressor shaft and between said shaft of the rotary machine and said compressor shaft (**142**), when said compressor is locked.

6. The composite auxiliary machine for a vehicle according to claim 5, wherein said limiter mechanism is disposed closer to said compressor than said one-way clutch.

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7. The composite auxiliary machine for a vehicle according to claim 1, wherein said torque and power distributing mechanism is located inside an enclosed space, and

said drive shaft, said shaft of the rotary machine, and said compressor shaft are provided with sealing members at portions extending into said enclosed space.

8. The composite auxiliary machine for a vehicle according to claim 7, wherein said intermittent means is a clutch that works with lubrication oil and is located inside said enclosed space together with said torque and power distributing mechanism.

9. The composite auxiliary machine for a vehicle according to claim 8, wherein said rotary machine is a brushless rotary machine and is located inside said enclosed space together with said torque and power distributing mechanism.

10. The composite auxiliary machine for a vehicle according to claim 1, wherein

said intermittent means is a clutch that works with lubrication oil,

said rotary machine is a brushless rotary machine, and said intermittent means, said rotary machine, and said torque and power distributing mechanism are supplied with said refrigerant.

11. A control unit of a composite auxiliary machine for a vehicle having a function of automatically shutting off its

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vehicle engine in accordance with its driving state, the control unit including the composite auxiliary machine for a vehicle according to claim 3, and control means for controlling operations of said variable displacement mechanism, said rotary machine, and said intermittent means, wherein said control means executes a control method of the composite auxiliary machine for a vehicle, the method comprising the steps of:

connecting said intermittent means while said vehicle engine is running;

disconnecting said intermittent means while said vehicle engine is not running, and causing said rotary machine to rotate in a reverse direction so as to operate as an electric motor for driving said compressor;

disconnecting said intermittent means when starting said vehicle engine, and causing said rotary machine to rotate in a forward direction so as to operate as an electric motor for starting said vehicle engine; and

operating said variable displacement mechanism while the compressor is working to change an amount of discharge of said compressor in accordance with a predetermined thermal load of said refrigeration cycle device.

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